

AN ECONOMY ANALYSIS OF
AUTOMOBILE ENGINES BY A
NEW METHOD

A THESIS

PRESENTED BY

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TO THE

PRESIDENT AND FACULTY

OF

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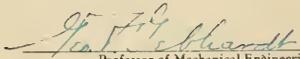
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OBJECT.

The object of this thesis is to develop a more comprehensive method of testing variable speed automobile engines. Up to the present time, the common method of testing variable speed automobile engines, using gasoline as fuel, has been to take the required readings under wide open throttle conditions, or at some more or less fixed partial load. It is obvious that the data thus obtained is very limited and do not simulate closely all of the actual conditions under which the engine operates. It is the aim of this thesis, therefore, to develop a system of taking data so that the results obtained would overcome the deficiencies outlined above, especial stress being laid on the study of the fuel economy.

Results of preliminary tests along these lines are found in the thesis of Cooban, Palmer and Stepanek, Class of 1915, Armour Institute of Technology.

PART 1.

AN ECONOMIC ANALYSIS of AUTOMOBILE ENGINES
by
A NEW METHOD.

PART 1.

Chalmers 3400 R.P.M. Engine.

The engine used in these tests was a Chalmers 3400 R.P.M. engine: type 35A and number 15039. It is a six cylinder, four cycle, L head engine having a bore of 3-1/4 " and a stroke of 4-1/2 ". The six cylinders are cast enbloc. The valves are located on the right side looking from the front, and the pushrods are encased by removable cover plates. The engine has a thermo-syphon system of water cooling. The ignition consists of a six volt Remy battery distributor and coil system. The Bosch 7/8 " three point type spark plugs were used. The carburetors were heated by hot air from a stove on the exhaust manifold.

Electric Dynamometer.

The Sprague electro-dynamometer was used in the testing of the Chalmers 3400 R.P.M. engine. Briefly,

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it consists of a one-hundred horse-power direct current inter-pole generator mounted on a cast iron bed plate. The torque is taken from knife edges screwed to the frame of the generator and transmitted thru a drawbar and spring balance scales to a set of chatillion scales. The length of this arm is equal to 1.315 feet, so that the torque multiplied by the R.P.M. and divided by 4000 gives the horse power developed. Ways are cast in the bed plate for holding down the engine stands; these stands can be adjusted so as to accomodate any size engine and have the engine lined up with the armature shaft in such a way that a flexible coupling can be inserted between the two. The switchboard is mounted on pipe stands within reach of the scale beam. It contains the control switches, field rheostat, circuit breaker, ammeter and voltmeter, and electro-tachometer. In order to maintain a steady field flux that will not vary with the speed, the field is separately excited. The drawing at the back of this thesis shows the

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electrical connections of the apparatus.

Water Cooling Apparatus.

Considerable difficulty was encountered in keeping the cooling water temperatures constant. Many methods of creating circulation were tried before the inlet temperature could be held at approximately 130 deg. fahr., and the outlet temperature at approximately 160 deg. fahr.

In all of the series of tests, the engine fan was disconnected and the radiator replaced by a cooling water tank.

Diagram A shows the water connections used on the preliminary test. The key to diagrams A, B, C, D, E and F, is as follows:

M - Engine

T - Cooling water tank

O - Overflow

V - Valve

J - Jet or nozzle

P - Pump (driven by an electric motor)

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H.C. - Heating coil.

Connections made as indicated in diagram A did not prove satisfactory in that the outlet water temperature was far above 160 deg. fahr., the normal running temperature. The jet (J) was used to create circulation in the direction of the arrows. The hot outlet water entered the tank (T) at the top connection, and any excess water drained off through the overflow (O).

Diagram B shows the addition of a circulating pump (P) to the inlet pipe. Although this was some improvement, it did not correct the wide variation in the two temperatures, and could not keep the temperatures constant throughout the run. When the outlet temperature was correct, the inlet temperature was far too low.

To raise the inlet temperature and still have enough circulation to keep down the outlet temperature, the pump was connected as in diagram C, which shows the hot outlet water being drawn through the

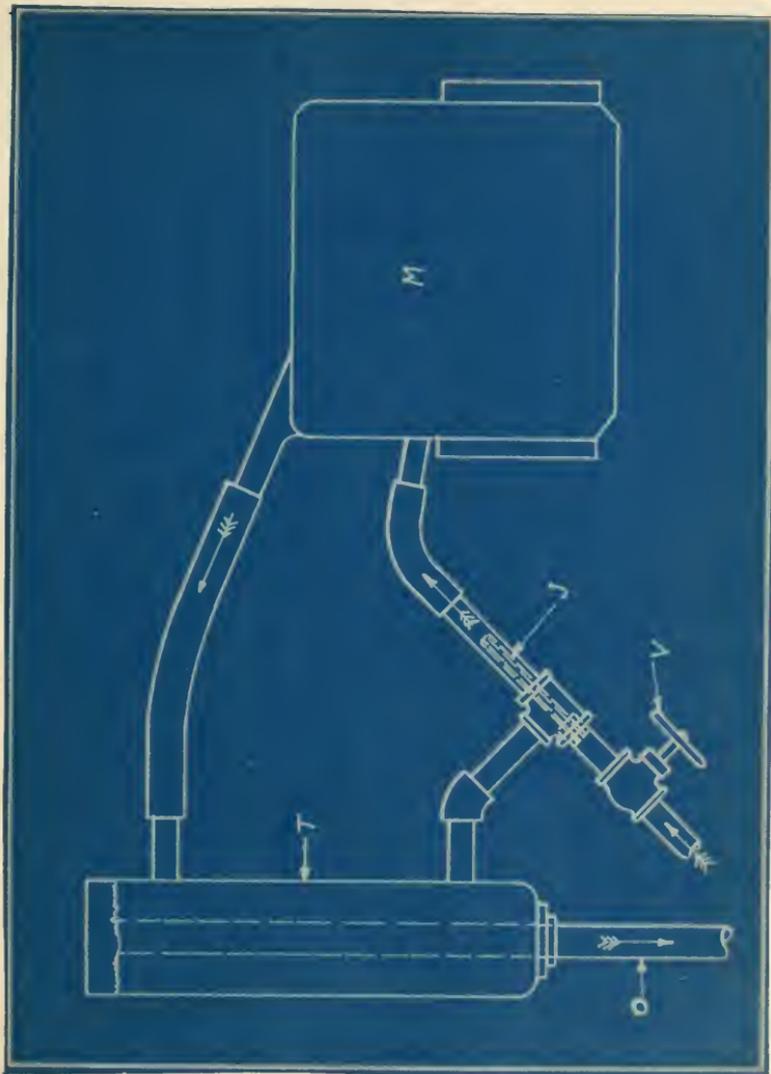


Diagram A

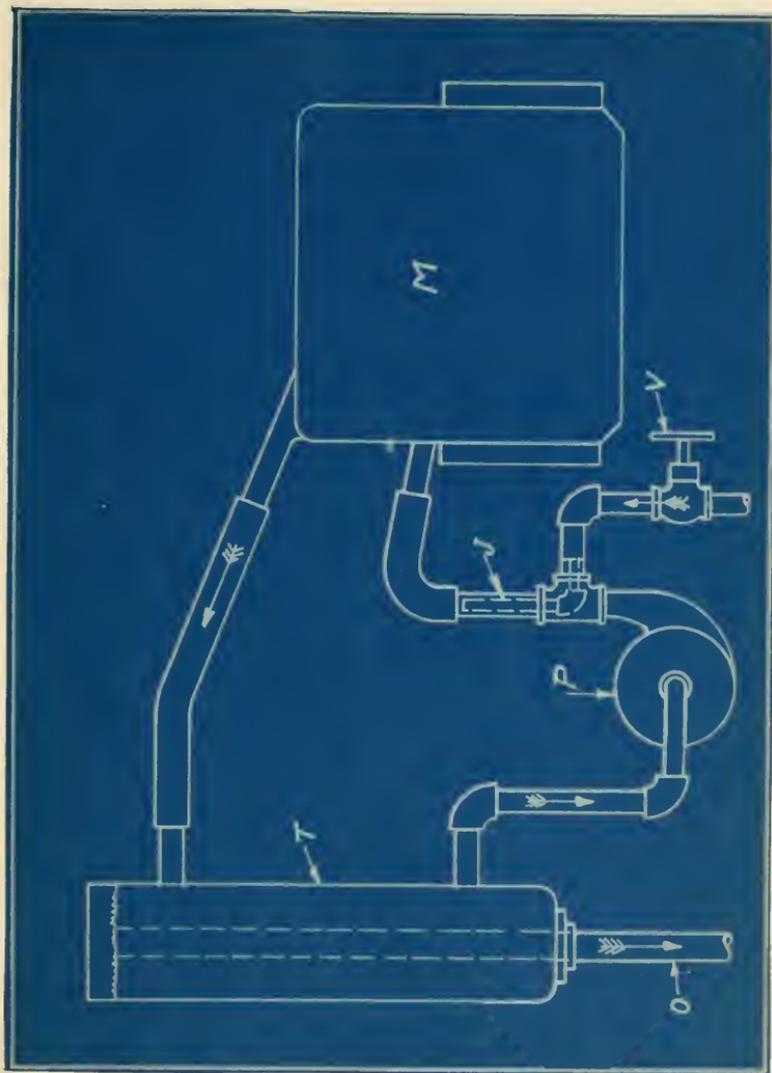


Diagram B

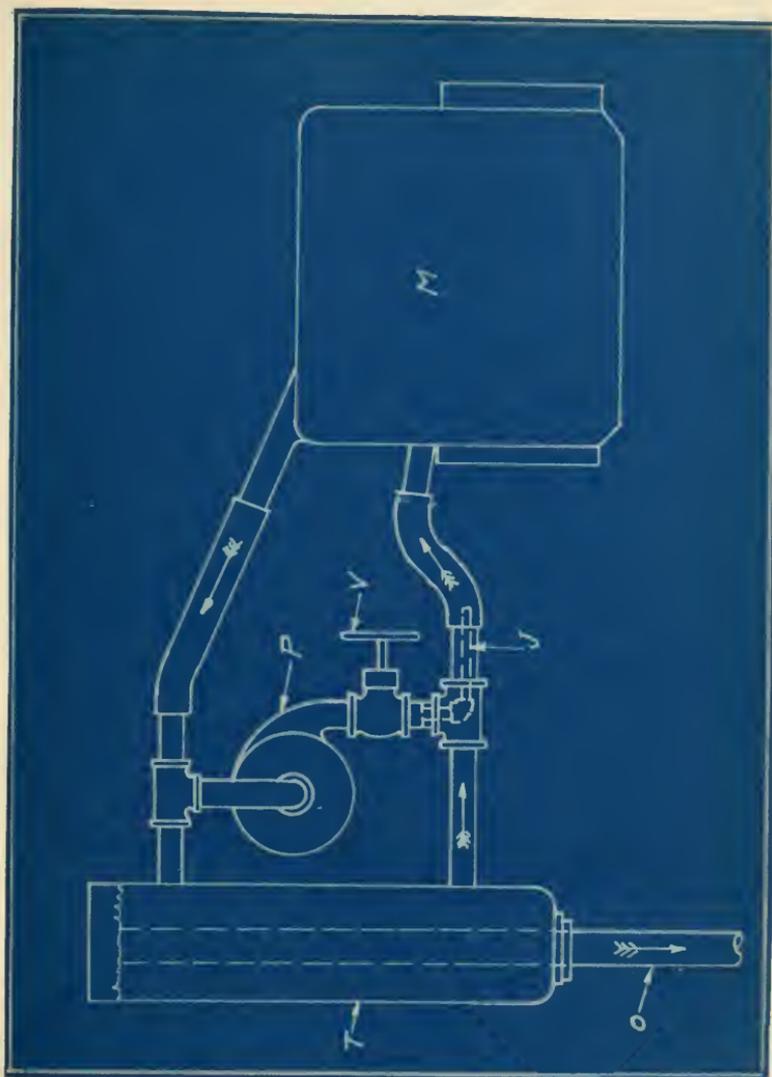


Diagram C



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pump in order to make the inlet temperature high enough. Although an improvement was shown in the temperatures, the connections were again changed as shown in diagram D.

The jet (J) was moved up to the outlet and in this manner relieved the inlet of a cold stream of water which held its temperature far too low. With the jet at the upper opening, the cold water in the tank entered the engine through the bottom connection and again the temperatures were not in accordance with those of the actual operating automobile engine.

The connections were next made as shown in diagram E, in which a water heating coil was used. The jet was brought down to the bottom in order to pull the jet of water through the heater. This method also proved unsuccessful.

When the connections were made as in diagram F, the temperatures of the inlet and outlet could easily be kept as desired. The jet and engine connections were just the same as those shown in diagram D,

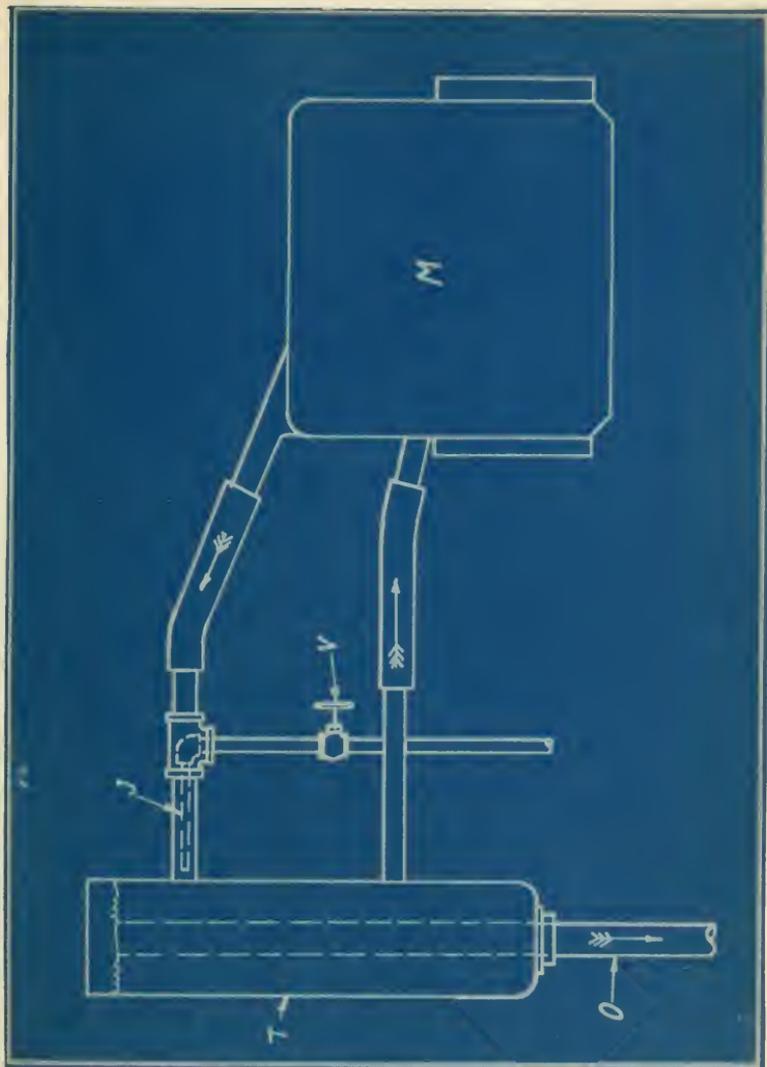


Diagram D

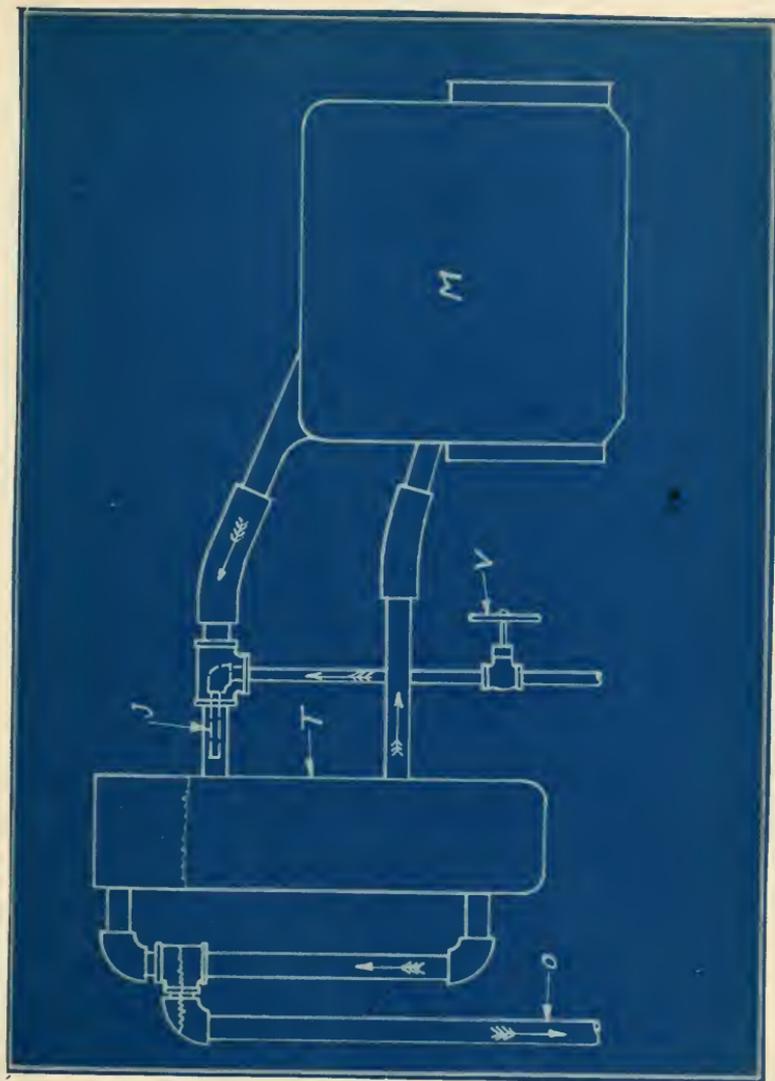


Diagram F

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the only difference being in the draining of the tank. The overflow, as shown in diagram F, had two tank connections, one at the top above the water line, and one at the lowest possible point in the tank. The cold water in the bottom of the tank drained out of the lower outlet to the left and up to the overflow line. The upper tank outlet was used only to prevent the siphoning of all of the water in the tank into the overflow.

The connections made as shown in diagram F were used in the Stromberg test #2 and in the Rayfield test, and the desired temperatures were easily maintained.

Gasoline Weighing Apparatus.

Figure A shows clearly the construction of the gasoline weighing apparatus used in these tests. The gasoline tank shown near the center of the picture is mounted upon three legs. These legs are screwed tightly into the weighing platform of a small platform scale. Gasoline is poured into the top of the

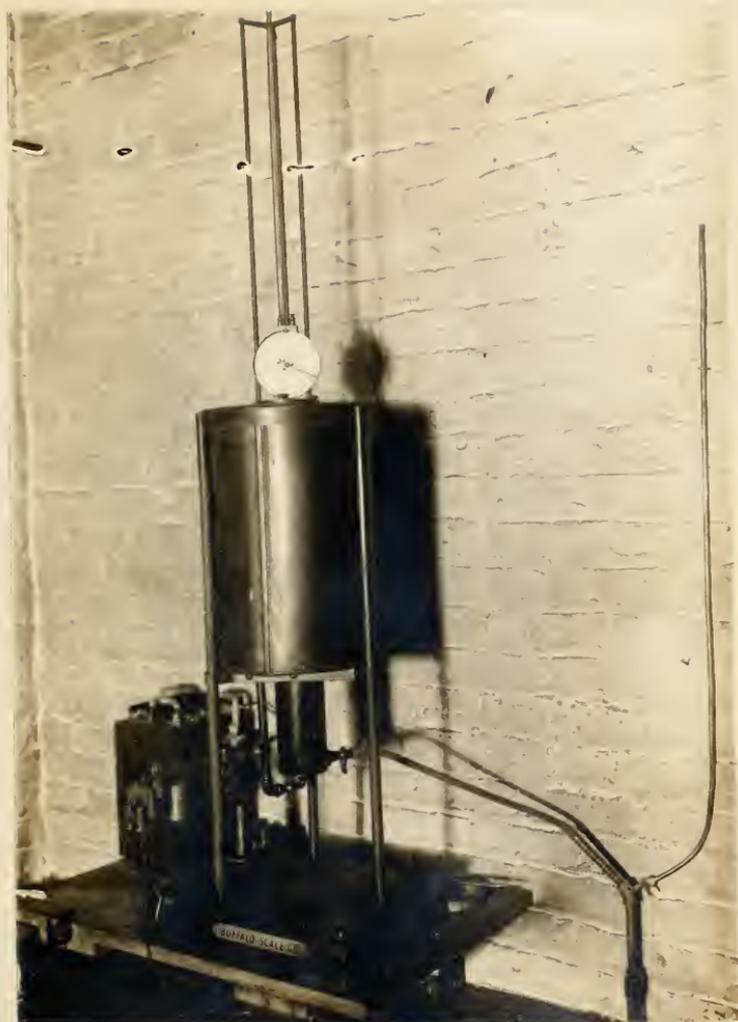


Fig. A

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tank without removing the apparatus from its position on a shelf which is approximately two feet above the carburetor on the test engine. As the gasoline is consumed by the engine the gasoline in the tank drains down through the soft rubber hose connection shown just above the right hand corner of the supporting shelf. The vertical copper tube shown at the right of the picture is an atmospheric vent which allows any air bubbles in the gasoline line to escape. At the extreme end of the balance beam is an electrical contact which is made in two very small cups of mercury when the beam drops. This contact rings an electric bell as soon as the beam drops. When a test is being made the gasoline is continuously flowing from the tank. When the weight of the gasoline and the tank just balance, the beam will start to drop and the alarm bell will ring. A stop watch is used and starts with the ringing of the bell. The slide weight is now pushed over until the weight of gasoline to be used in

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the run has been accounted for on the beam slide weight. When that amount of gasoline is used up, the weight of the remaining gasoline and the tank will balance the beam again and the second alarm bell rings. The stop watch indicates the time that the engine takes to use up the given weight of gasoline.

Manifold Depression Apparatus.

The apparatus used for measuring the depression in the intake manifold consists of a "U" shaped manometer connected by means of a copper tube to a brass tube tapped into the intake manifold. Figure B shows the manometer mounted as it was in the test. The difference in heights of the two columns was measured with a steel scale with less than one-sixteenth inch error on average readings and within three-sixteenths of an inch on the sixteen and eighteen inch readings. In the constant manifold depression tests, the throttle and spark were adjusted until the difference in heights of the mer-

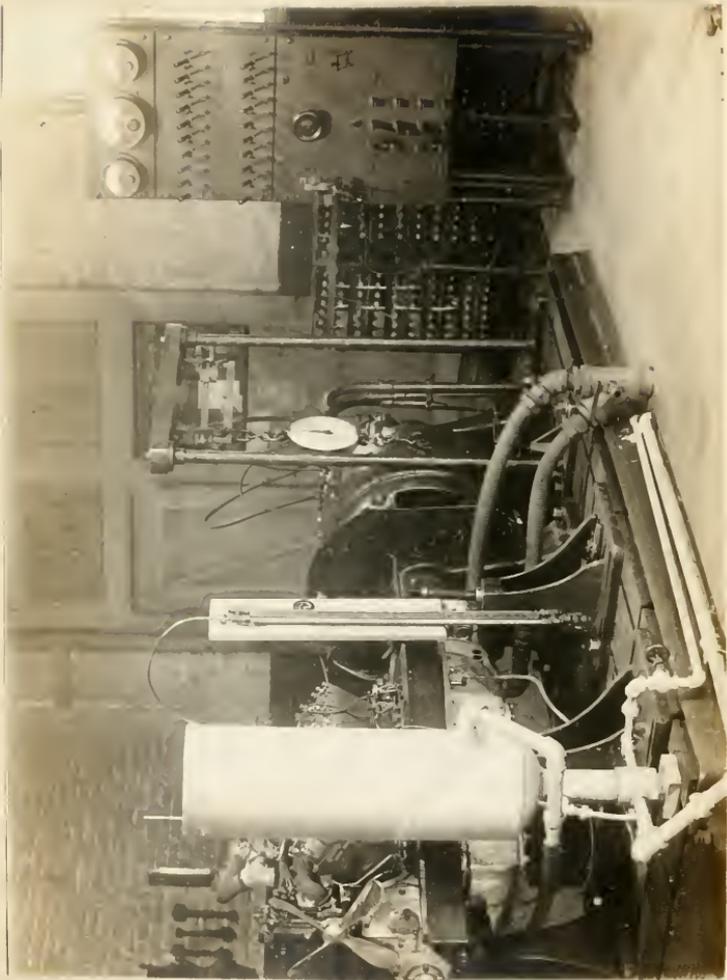


Fig. B

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cury columns became constant and as high as required then the actual run was made.

Spark Advance Mechanism.

Figure C shows the end of the crank shaft upon which the spark advance mechanism was mounted. This mechanism consists of a stationary ring about five inches in diameter as shown in the figure. The outer part of the face of this ring is divided into degrees from zero to 180 in each direction, each degree on the ring representing a degree of spark advance or retard. Fixed tightly onto the end of the crank shaft is a lever about two and one-half inches from the center to the end. A sharp pointed adjustable brass screw is near the end of this lever, the point of the screw being about three-sixteenths of an inch from the face of the ring. This lever being connected to the crank shaft is in the ground circuit of the engine. In order to get the spark advance for any particular run, a piece of secondary wire was connected from a binding post on the

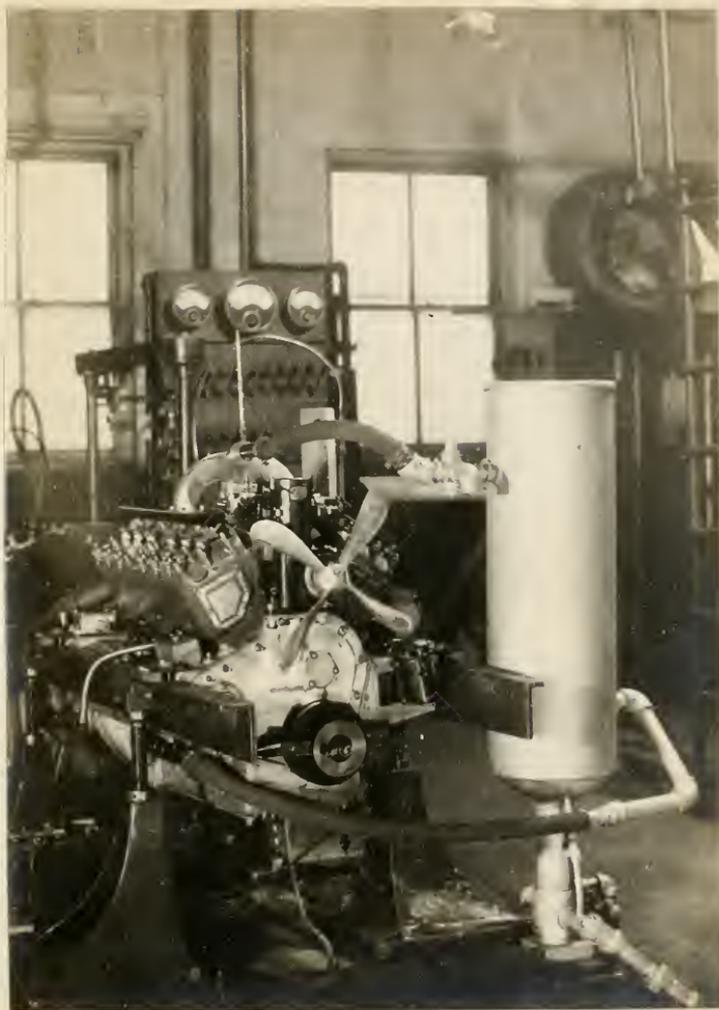


Fig. C

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ring to the spark plug of a certain cylinder of the engine. This wire led the current down to the ring point where it jumped the gap instead of jumping the spark plug points. The zero of the ring divisions was set in such a position that the spark occurred at zero when the piston of the chosen cylinder was at the end of its stroke.

Speed Counting Apparatus.

Figure D shows the speed counting apparatus attached to the end of the dynamometer shaft. The speed counting apparatus consists of a small revolution counting device (an odometer) the shaft of which can be connected or disconnected to the shaft of the dynamometer by means of a small electrically operated clutch. In order to get the number of revolutions that the engine is making per minute, the reading of the odometer is first noted and then a stop watch and an electric switch are pressed simultaneously. The current through the switch magnetizes the core of a solenoid and pulls the speed

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Fig. D

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counter clutch into place. At the end of a minute, as indicated on the stop watch, the switch is thrown out and the odometer reading is again taken. The difference between the two odometer readings is the number of revolutions per minute made by the engine.

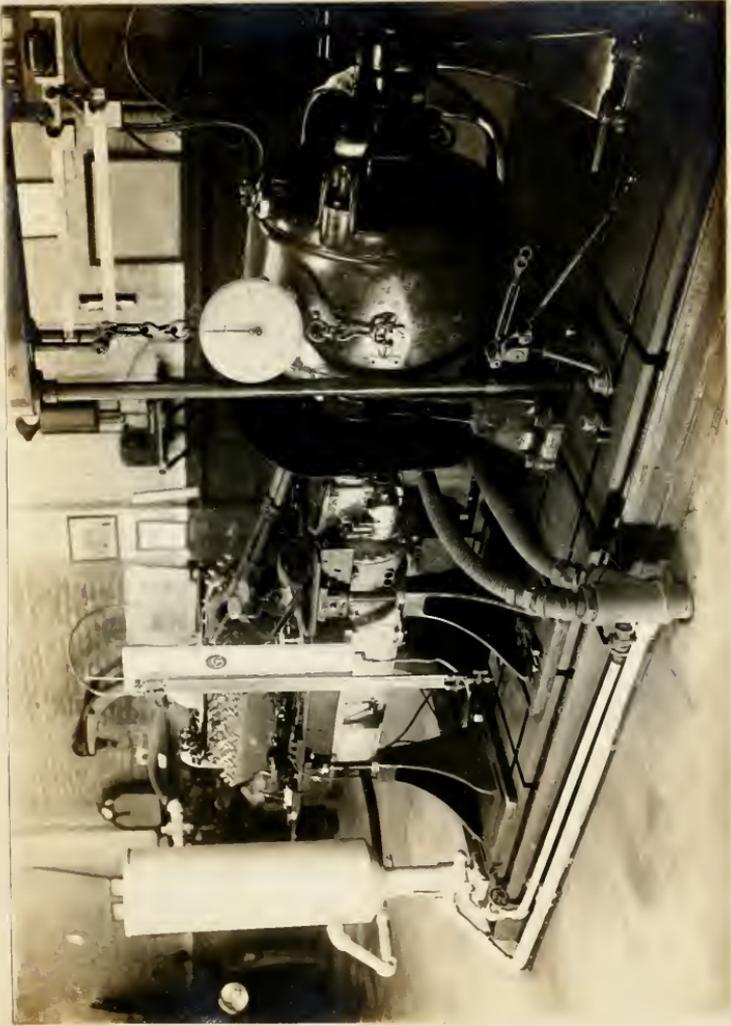


Fig. E



Fig. F

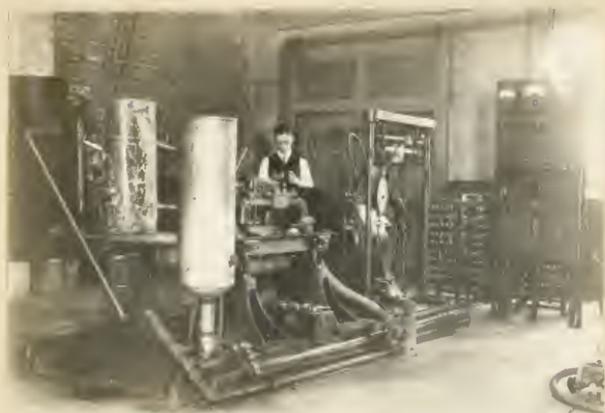


Fig. G



Fig. H

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Preliminary Observations.

Before beginning the actual test of this kind it is always desirable to take a preliminary set of runs to see that the engine is in the proper condition for testing.

Probably the best thing to examine first is the coupling between the engine and the dynamometer, and also the supports and fastenings of the engine. All of these should be tight and free from excessive vibration at high speeds and heavy loads.

The carburetor was the next thing in consideration. It was set to give fair acceleration, good economy and to allow the engine to throttle down nicely.

All of the instruments were now examined to see that they were in good working order and then a short run was made to see that everything was in condition for testing.

A set of runs was now taken for the purpose of obtaining the characteristics of the engine. As

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the engine was fairly tight, no heavy loads were applied. However, a very brief run was made at wide open throttle, only the torque and R.P.M. being taken to determine whether the engine would develop as much power as we had reason to expect. Having seen that the engine and instruments were in good condition, the engine was ready for the first set of runs.

Before commencing the tests the specific gravity and the temperature of the gasoline were taken. The barometer was also read at the time of each test.

Method of Testing.

In pursuing these investigations, two methods of testing were employed, namely, constant suction and constant torque.

Tests number 1 and 2 were each made by taking a series of runs each set being at a different constant manifold depression, except one set of runs which was made at wide open throttle, in this case the depression for each run was measured. This

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wide open throttle run was made to secure a brake horse-power curve that would serve as a limit for the other brake horse-power curves taken at the other manifold depressions. Each set of runs was composed of runs taken at various speeds. With each change in speed the load and throttle opening were changed so as to keep the manifold depression constant.

Test number 3 was made by taking a series of runs, each set of runs being taken at a constant torque, except one set of runs which was at wide open throttle. Each set of runs was composed of runs made at various speeds. As the speed was increased the throttle valve was opened so that a constant torque was maintained.

- The friction horse-power was secured by taking five sets of runs, each set being taken at nearly a constant speed. In each set of runs, the dynamometer, now acting as a motor was set to run at a certain speed. The depression in the manifold was

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set at the various settings used in the previous tests and the torque and R.P.M. recorded for each run. Readings for wide open throttle were taken first and then the throttle was gradually closed to secure each manifold depression. As the throttle was closed, the torque became larger and consequently the R.P.M. fell off a little in each set of runs. Readings could not be secured above 1200 R.P.M. because the current taken by the motor would exceed 120 amperes and the circuit breaker would go out. This made it necessary to extend the curves in order to get complete data.

In these tests the data was taken by starting the runs at the slower speeds and then increasing the speeds to about 2000 R.P.M., after which the engine was given a few minutes to cool off between each run. This cooling was done only on the wide open throttle and a few of the low depression sets of runs during which the exhaust manifold became red hot. The engine was not stopped during a set of

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runs. Cooling was accomplished by unloading the engine and letting it run at a slow speed.

The length of time required to take a run depended upon the rate of consumption of the gasoline. Weights of gasoline of one, five-tenths, and three-tenths pounds were usually used, the one pound weight being used for the higher speeds and loads, and the lower weights for the slower speeds and loads. The runs lasted between one and four minutes depending upon the rate of consumption of gasoline. The engine was first set for the required conditions and when it was observed that there was no variation in the running of the engine, the gasoline consumption was taken. This was done by taking the stop watch time for a given weight of gasoline to be consumed. As soon as this reading was taken, the engine was set for the next run and so on until high speeds were reached, where a little more time between the runs was allowed.

The spark advance was taken immediately after

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each run before any conditions were changed. This was accomplished by disconnecting the wire leading to the spark plug in number one cylinder and touching it to the spark advance indicator and noting the angle indicated by the device. The wire was then returned to the spark plug and the next setting made.

The R.P.M. was taken by means of the electrically operated odometer. The time was taken for one minute by means of a stop watch.

The temperature of the water was controlled by the valve which controls the flow of cold water into the system. The temperatures at the inlet and outlet of the engine were held as nearly constant as possible. These were read at the end of each run.

The torque was taken directly from the beam of the scale for each run and in case of a variation of the torque during the run, an average was taken.

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Method of Procedure in Calculations.

During the test the following data were taken: torque, R.P.M., time required to consume a known weight of fuel, manifold depression (suction, inches of mercury), spark advance and inlet and outlet cooling water temperatures. These data are shown in tables 1-10 inclusive, 46-54 inclusive, and 71-79 inclusive. For the friction horse-power test the following data were taken: torque, R.P.M., manifold depression (suction, inches of mercury) and average inlet and outlet cooling water temperatures. These data are shown in tables 40-42 inclusive.

In calculating both the brake horse-power and the friction horse-power, the formula

$$\text{B.H.P.} = \frac{\text{Torque} \times \text{R.P.M.}}{4000}$$

was used, the constant 4000 being composed of the factors $\frac{2 R}{33000}$, where R is the lever arm of the brake, which in this case is 1.315 feet.

The pounds of gasoline per hour were calcu-

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lated from the formula:

$$\text{Lb. gas./hr.} = \frac{3600 \times W}{\text{Time in seconds.}}$$

in which W was the weight, in pounds, of gasoline used for that particular run.

In calculating the pounds of gasoline per hour per brake horse-power, the pounds of gasoline per hour were divided by the brake horse-power.

The mechanical efficiency was obtained by dividing the brake horse-power of each run by the sum of the brake and friction horse-powers of the same run.

The mean effective pressure net (M.E.P.) was obtained by multiplying the torque by the constant 0.886. This constant was obtained from the equality:

$$\text{B.H.P.} = \frac{P L A N}{33000} = \frac{\text{Torque} \times \text{R.P.M.}}{4000}$$

where P = mean effective pressure (lb./sq.in.)

L = stroke in feet

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A = area of piston in square inches.

N = explosions per min. = $3/2 \times$ R.P.M.

$$\begin{aligned} \text{Therefore } P &= \frac{2 \times 33000}{3 \times 4000 \times L \times A} \times \text{torque} \\ &= 0.886 \times \text{torque} \end{aligned}$$

The thermal efficiency was calculated by means of the Sherman and Kropf formula for the heat value of gasoline:

$$\begin{aligned} \text{Therm. eff.} &= \frac{2546}{M \times [18320 + 40 (B_e - 10)]} \\ &= \frac{2546}{M \times 20180} \end{aligned}$$

where M is the pounds of gasoline per brake horse-power per hour.

The method of obtaining the constant R.P.M. curves for equal gasoline economy and equal mechanical efficiency was as follows: In the case of the former, the brake horse-power and the pounds of gasoline per hour per brake horse-power were "picked off" from the curves for each run in their re-

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spective sets at arbitrarily chosen engine speeds. This data were then tabulated, and are shown in tables 21-25 inclusive for the Stromberg test #1, tables 55-59 inclusive for the Stromberg test #2, and tables 80-83 inclusive for the Rayfield test. In the case of the constant R.P.M. curves for equal mechanical efficiency, the brake horse-power and the mechanical efficiency were "picked off" from the curves for each run in their respective sets at predetermined engine speeds. This data were also tabulated, and are shown in tables 31-34 inclusive for the Stromberg test #1, and tables 64-67 inclusive for the Stromberg test #2.

The above data were then plotted as shown in figures 67, 69, 71 for the equal gasoline economy, and figures 35 and 56 for equal mechanical efficiency.

From these curves data were tabulated which was obtained in the following manner: A certain line of pounds of gasoline per hour per brake horse-

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power or of mechanical efficiency was chosen, and then where ever this line cut a curve, the corresponding R.P.M. and brake horse-power were noted. For example, in figure 35, the line of 80% mechanical efficiency cuts the 1800 R.P.M. curve at 32.5 horse-power, and so on across the sheet (see table 37). This interpolation data from both the equal gasoline economy and equal mechanical efficiency curves were plotted and the curves in figures 33, 54, 66, and figures 55 and 34, respectively, were obtained.

Sample Calculations.

In the following sample calculations, run #6, Stromberg test #1 was used:

Brake horse-power:

$$\begin{aligned} \text{B.H.P.} &= \frac{\text{Torque} \times \text{R.P.M.}}{4000} \\ &= \frac{89.5 \times 1346}{4000} = 30.1 \end{aligned}$$

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Lb. gasoline per hour:

$$\begin{aligned} \text{Lb. gas./hr.} &= \frac{3600 \times W}{\text{Time in sec.}} \\ &= \frac{3600 \times 0.5}{75.2} = 23.90 \end{aligned}$$

Lb. gasoline per B.H.P. per hour:

$$\begin{aligned} \text{Lb. gas./B.H.P./hr.} &= \frac{\text{Lb. gas./hr.}}{\text{B.H.P.}} \\ &= \frac{23.90}{30.1} = 0.794 \end{aligned}$$

Mechanical efficiency:

$$\begin{aligned} \text{Mech. eff.} &= \frac{\text{B.H.P.}}{\text{B.H.P.} \quad \text{F.H.P.}} \times 100 \\ &= \frac{30.1}{34.2} \times 100 = 88 \% \end{aligned}$$

Mean effective pressure net:

$$\begin{aligned} \text{M.E.P.} &= 0.886 \times \text{Torque} \\ &= 0.886 \times 89.5 = 79.4 \text{ lb./sq.in.} \end{aligned}$$

Thermal efficiency:

$$\begin{aligned} \text{Therm. eff.} &= \frac{2546}{M \times 20180} \\ &= \frac{2546}{0.794 \times 20180} = 15.9 \% \end{aligned}$$

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Final Observations.

After each test the engine was accelerated to determine whether it was as limber as at the start of the run. After test number one was completed, the spark plugs were replaced, the porcelain of one of them having been broken.

After completing the last test, the valve lift diagrams were taken. An examination of the cylinders showed that they were slightly scored.

PART 111.

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Discussion of Results.

In this test of the Chalmers 3400 R.P.M. engine, the primary object was to determine the performance and economy of the engine with two different makes of carburetors.

The Stromberg test #1 and the Stromberg test #2 were both made with the same carburetor, the only main difference being the temperature of the inlet and outlet cooling water. Test #3 was made with a Rayfield carburetor and with stimulated water cooling circulation.

In test #1, the wide range in temperature between the inlet and outlet cooling water did not make it possible to operate the engine in its most efficient manner. As can be seen from table 1, there was a range of approximately 65 deg. fahr. The data obtained under these conditions were so inconsistent that it was necessary to revise it by means of the curves. The data taken during the test were plotted and smooth curves, consistent with or-

CHAPTER IV

The first part of the book is devoted to a general survey of the history of the subject. It begins with a discussion of the early stages of the development of the subject, and then proceeds to a more detailed examination of the various branches of the subject. The author discusses the contributions of the various schools of thought, and the influence of the various countries on the development of the subject. He also discusses the various methods of research, and the various theories of the subject. The second part of the book is devoted to a more detailed examination of the various branches of the subject. It begins with a discussion of the history of the subject, and then proceeds to a more detailed examination of the various branches of the subject. The author discusses the contributions of the various schools of thought, and the influence of the various countries on the development of the subject. He also discusses the various methods of research, and the various theories of the subject.

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dinary practice, were drawn through these points. These curves were drawn so that they could not represent any impossible conditions. To illustrate this point, reference is made to figures 21 and 22. In figure 21 are the curves plotted from the data taken during the test. It is obvious, if the engine was running in a consistent manner, that the horse-power curves plotted from the data taken at 2, 4, 6, and 8 inch manifold depressions should not cross each other; and, furthermore, that, as the manifold depression in the wide open throttle test went as high as 6.5 inches of mercury, it could be assumed that the horse-power curves obtained from data taken at the 2, 4, and 6 inch manifold depressions should intersect and be limited by the points on the wide open throttle horse-power curve at which there was a 2, 4, and 6 inch manifold depression, respectively. It was with these assumptions in view that the curves in figure were drawn. In a like manner it was assumed that under consistent running conditions,

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the engine could not have the same rate of fuel consumption at two different throttle openings and the same speed. Therefore, the pounds of gasoline per hour curves were made to conform with these assumptions. The data in tables 11-20 inclusive were obtained by means of these two sets of revised curves. The brake horse-power and the pounds of gasoline per hour were taken from these curves at every 200 R.P.M., and the pounds of gasoline per brake horse-power per hour were calculated from this data. The torque was obtained by multiplying the brake horse-power by 4000 and dividing by the R.P.M. The M.E.P. net and the thermal efficiency were obtained in the usual manner heretofore described, using the revised data. The mechanical efficiency was calculated by using the same friction horse-power, but the revised brake horse-power. The curves representing this data are shown in figures 22-33 inclusive.

The only fair comparison, therefore, as far as carburetors are concerned would be to use the Strom-

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berg test #2 and the Rayfield test, as these were both made under the same water cooling conditions. With reference to figures 54 and 66, it will be observed that the characteristics of these two carburetors are very different. As can be seen from figure 54, in the case of the Stromberg carburetor, the zones of equal fuel consumption are very nearly parallel to the general direction of the wide open throttle horse-power curve. This condition would seem to indicate that for constant manifold depressions the fuel consumption would be nearly uniform. However, in the case of the Rayfield carburetor (fig. 66), with the sheet of combined horse-power curves in mind (fig.22), it can be seen that there is a wide range of manifold depression and speed at which there is practically a constant fuel consumption (below an economy of 0.9 lb./B.H.P./hr.). It would seem to indicate, from the curves, that the Rayfield carburetor would be the more economical at heavy loads and at any speed; whereas the Stromberg car-

ECONOMY ANALYSIS.

buretor would appear to be more economical at light loads and at any speed. As far as brake horse-power is concerned, the maximum horse-power developed with the Stromberg carburetor on the engine was 37.8 (table 46); while the maximum horse-power developed with the Rayfield carburetor attached was 39.2 (table 71).

In general, it can be said that with a decrease in manifold depression there is an increase in thermal efficiency, or in other words, with a decrease in the manifold depression there is a decrease in the pounds of fuel used per brake horse-power per hour. This characteristic varies with different engines, as indicated by the slope and extent of the closed economy lines on the two contour curves under discussion. It has been observed that some engines will not give a closed economy line under any conditions. In this case the general statement holds true exactly.

It will be observed from figures 1, 36, and 57,

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that with wide open throttle the suction in the manifold increases. This is a measure of the resistance of the carburetor. In the design of a carburetor for maximum power, it is the aim to keep this suction as low as possible so as to avoid an insufficient supply of mixture, thereby reducing the weight of the charge and therefore the horse-power output.

During the progress of this thesis, several facts were brought out that would be an improvement in the method of plotting the final results. It was observed that a more comprehensive study of the characteristics of the engine could be made by plotting the final results on three co-ordinates, thus making a surface instead of a sheet of contour lines. The three variables to be used being: brake horse-power, R.P.M., and pounds of gasoline per brake horse-power per hour. Also, that under the present method, that is, plotting all results against R.P.M. and using total B.H.P., two different engines whose

ECONOMY ANALYSIS.

bore and stroke differed could not be compared directly. It is therefore suggested that in the future tests along this line, the brake horse-power be reduced to brake horse-power per cubic inch piston displacement, and that all results be plotted against piston speed in feet per minute, thus giving a true commercial comparison between the two engines. It was also found that a certain number of inches of mercury depression in the manifold was not a true measure of the absolute pressure in the manifold, due to the changing of the barometric pressure. It is, therefore, suggested that in the future tests, the runs be made at constant absolute pressures in the intake manifold instead of constant intake manifold depressions.

In concluding, it may be said that this method of testing promises to be the most concise and accurate means of determining and comparing the characteristics of variable speed automobile engines.

PART IV.

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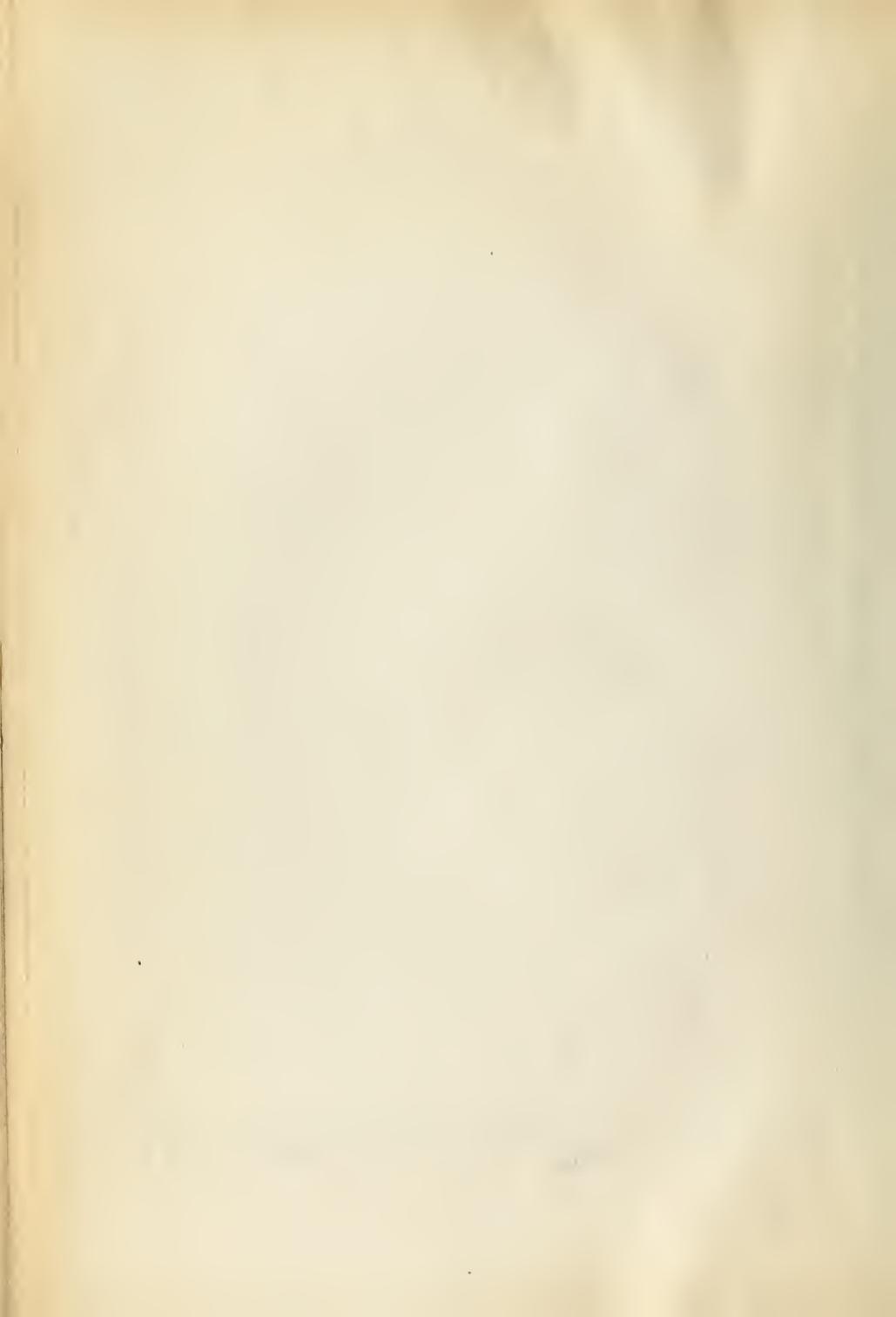
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Test No. 1. Motor: Chalmers Date of Test: 10 / 11 / 16
 Object: Economy Model: 35 A. 3400 R. P. M. Barometer = 29.42 in. Hg
 Constant: Suction Number: 15039 Room Temperature °F.
 No. Cyl. 6. Gasoline 56.5° B. at 64°F.

Run No.	R.P.M.	Torque Brake		Frict.		B.H.P.	F.H.P.	M.E.P. Net Eff. %	Jacket Temp.		Intake Suct. In. Hg	Gasoline		Therm. Deg.			
		lb.	HP	HP	HP				In.	Out.		lb.	Time		lb/HP per Hr.	Eff. %	
1	526	92.0	12.1	0.8	12.9	93.6	81.5	61	130	1.0	1.0	.5	2:20.1	12.84	1.060	11.9	8
2	637	92.5	14.7	1.1	15.8	93.0	82.0	65	121	0.7	0.7	.5	2:28.2	12.14	0.823	15.3	10
3	800	92.0	18.4	1.6	20.0	92.0	81.5	68	121	0.9	0.9	.5	2:01.3	14.83	0.806	15.7	18
4	1020	93.0	23.7	2.6	26.3	90.1	82.4	69	121	1.3	1.3	.5	1:33.2	19.30	0.614	15.5	18
5	1154	92.5	26.7	3.2	29.9	89.4	81.5	68	129	1.6	1.6	.5	1:29.2	20.17	0.755	16.7	21
6	1346	89.5	30.1	4.1	34.2	88.0	79.3	68	131	2.0	2.0	.5	1:15.2	23.90	0.794	15.9	25
7	1575	87.5	34.3	5.5	39.8	86.2	77.5	67	136	2.7	2.7	.5	1:07.4	26.75	0.782	16.2	26
8	1765	85.0	36.6	7.1	43.7	83.8	73.6	68	129	3.5	3.5	.5	0:59.4	30.30	0.628	15.3	30
9	2130	72.5	38.6	10.9	49.5	78.0	64.2	69	123	4.8	4.8	.5	0:55.4	32.50	0.642	15.0	31
10	2490	57.0	35.6	15.8	51.4	63.2	50.5	69	133	6.5	6.5	1.0	1:35.4	37.60	1.062	11.9	31

Table 1



Test No. 1. Motor: Chalmers Date of Test: 10 / 12 / 16
 Object: Economy Model: 35 A. 3400 R. F. M. Barometer = 29.40 in. Hg
 Constant: Suction Number: 15089 Room Temperature °F.
 No. Cyl. 6. Gasoline 56.5° B. at 64° F.

Run No.	RPM	Torque lb.	Brake HP	Frict. HP	B. HP F. HP	Mech. Eff. %	MEP Net. lb./sq. in.	Jacket Temp.		Intake Suct. in. Hg	Gasoline		lb/HP per Hr.	Therm. Deg. Eff. %
								In.	Out.		lb.	Time		
1	595	77.5	11.5	1.2	12.7	90.5	68.7	72	130	2.0	1:29.4	12.03	1.047	12.1 12
2	790	87.0	17.2	1.7	18.9	90.9	77.1	66	125	2.0	2:01.0	14.89	0.867	14.6 21
3	855	89.0	19.1	2.0	21.1	90.4	78.9	62	127	2.0	1:58.1	15.22	0.792	15.6 22
4	1187	91.0	27.0	3.4	30.4	88.9	80.6	62	132	2.0	1:21.2	22.17	0.822	15.4 25

Table 2

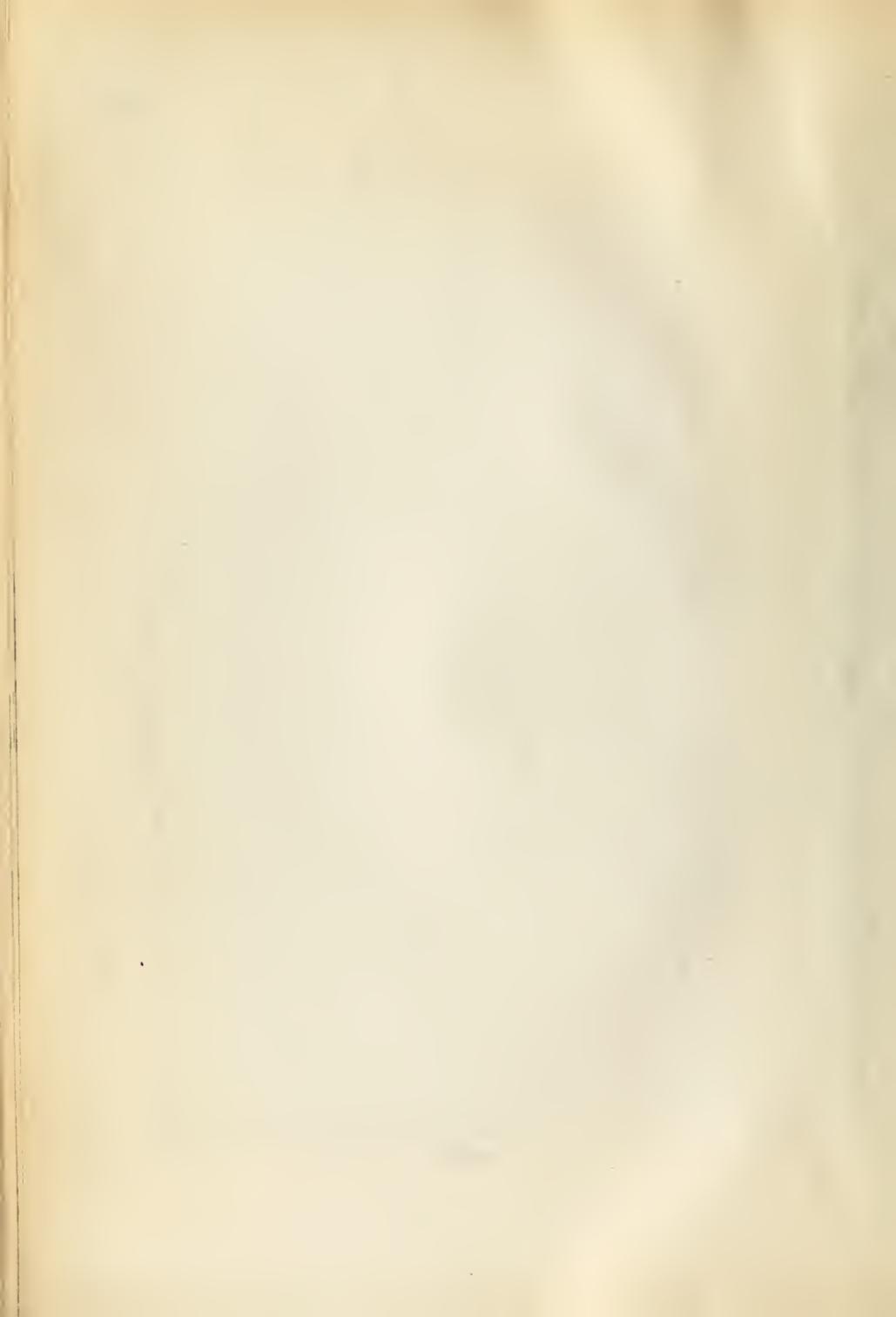
Test No. 1.		Motor: Chalmers		Date of Test: 10 / 12 / 16													
Object: Economy		Model: 35 A, 3400 R. P. M. Barometer = 29.40 in. Hg		Room Temperature °F													
Constant: Suction		Number: 15039		Gasoline 56.5°B. at 64° F.													
No. Cyl. 5.																	
Run No.	RPM.	Torque lb.	Brake HP	Frict. HP	B. HP	F. HP	Mech. Eff.	MEP Net. lb/sq. in.	Jacket Temp.		Intake Suct. in. Hg	Gasoline		Therm. Deg. per Hk. Eff. Spent			
									In.	Out.		lb. per Hour	Time				
1	503	72.7	9.1	1.0	10.1	90.2	64.5	65	135	4.0	4.0	.3	2:24.3	7.45	0.817	15.5	12
2	615	80.0	12.3	1.3	13.6	90.4	70.9	63	125	4.0	4.0	.5	3:26.4	8.72	0.708	17.8	19
3	836	81.0	16.9	2.0	18.9	89.4	71.8	62	130	4.0	4.0	.5	2:01.4	14.81	0.575	14.5	20
4	1137	85.7	23.7	3.2	26.9	88.1	74.2	63	135	4.0	4.0	.5	1:24.5	21.26	0.595	14.1	23
5	1403	81.0	28.4	4.7	33.1	85.8	71.8	66	132	4.0	4.0	.5	1:15.4	23.90	0.545	15.0	28
6	1894	68.5	32.4	8.7	41.1	78.8	60.7	68	133	4.0	4.0	.5	0:58.1	31.00	0.956	13.2	32

Table 3

Test No. 1. Motor: Chalmers Date of Test: 10 / 12 / 16
 Object: Economy Model: 35 A. P. M. Barometer = 29.40 in. Hg
 Constant: Suction Number: 15039 Room Temperature °F.
 No. Cyl. 5. Gasoline 56.5° B. at 68°F.

Run No.	RPM	Torque lb.	Brake HP	Frict. HP	B. HP	Mach. Eff. %	MEP. Net. lb./sq. in.	Jacket Temp. °F.	Intake Suct. in. Hg	Gasoline		lb per Hour	Therm. Deg. Eff.			
										Time	lb. / HP					
1	447	64.5	7.2	0.9	8.1	88.9	57.2	60	124	6.0	.3	1:18.7	13.73	1.910	6.6	20
2	617	68.0	10.5	1.4	11.9	88.4	60.2	60	128	6.0	.5	3:35.4	8.36	0.796	15.9	23
3	745	71.0	13.2	1.8	15.0	88.0	62.9	60	130	6.0	.5	2:56.8	10.18	0.770	16.4	26
4	1080	73.0	19.7	3.1	22.8	86.2	64.7	63	130	6.0	.5	1:23.7	20.06	1.013	12.4	22
5	1195	68.0	20.3	3.7	24.0	84.6	60.2	60	132	6.0	.5	2:23.2	12.59	0.618	20.4	26
6	1507	73.0	27.5	5.6	33.1	83.7	64.7	62	138	6.0	.5	1:19.9	22.50	0.813	15.4	31
7	1893	70.0	33.1	9.1	42.2	78.5	62.0	69	125	6.0	.5	1:02.7	28.70	0.565	14.6	37
8	2189	65.0	35.6	12.5	48.1	74.0	57.6	69	134	6.0	.5	0:55.4	32.50	0.913	13.8	40
9	2326	60.0	34.9	14.2	49.1	71.0	53.2	69	125	6.0	.5	0:39.0	46.20	1.323	9.5	41

Table 4

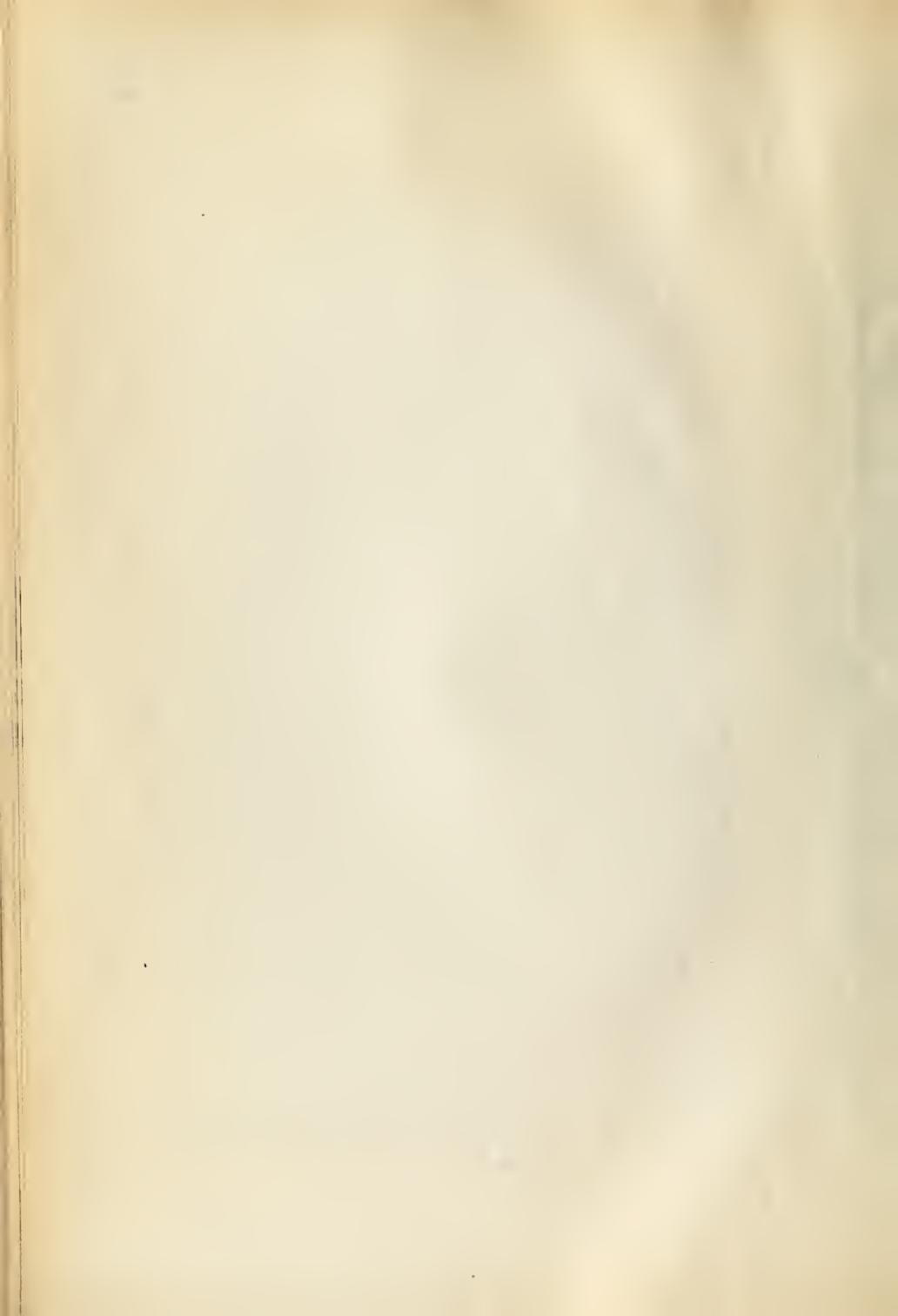


Run No.	R.P.M.	Torque lb.	Brake HP	Frict. HP	B.F.P.	Mech. Eff.	M.E.P. Net. $\frac{lb}{sq. in.}$	Jacket Temp.		Intake Succ. $\frac{in. Hg.}{lb.}$	Gasoline		Therm. Deg. per Hk. Err. $\frac{Deg.}{lb. per Hk.}$			
								In.	Out.		lb. per Hour	Time				
1	451	54.0	6.1	1.0	7.0	86.0	47.9	60	117	8.0	.3	3:07.2	5.77	0.948	13.3	13
2	756	59.2	11.2	1.9	13.1	85.9	52.5	66	114	8.0	.5	4:00.8	7.47	0.668	18.9	16
3	1000	64.5	16.1	2.9	19.0	84.6	57.2	70	111	8.0	.5	2:32.5	11.80	0.733	17.2	20
4	1280	62.2	19.9	4.3	24.2	82.2	55.2	69	123	8.0	.5	1:49.2	16.48	0.827	15.3	26
5	1588	66.7	26.5	6.4	32.9	80.6	59.2	67	128	8.0	.5	1:19.4	22.70	0.856	14.8	31
6	1926	65.0	31.3	9.4	40.7	76.9	57.6	69	127	8.0	.5	1:04.9	27.77	0.887	14.2	33
7	2354	55.7	32.9	14.6	47.5	69.2	49.4	70	129	8.0	.5	0:56.2	32.02	0.971	13.0	38

Test No. 1. Motor: CHalmers Date of Test: 10 / 11 / 15
 Object: Economy Model: 35 A. 3400 H. P. M. Barometer = 23.42 in. Hg
 Constant: Suction Number: 15039 Room Temperature of Gasoline 56.5° B. at 54° F.

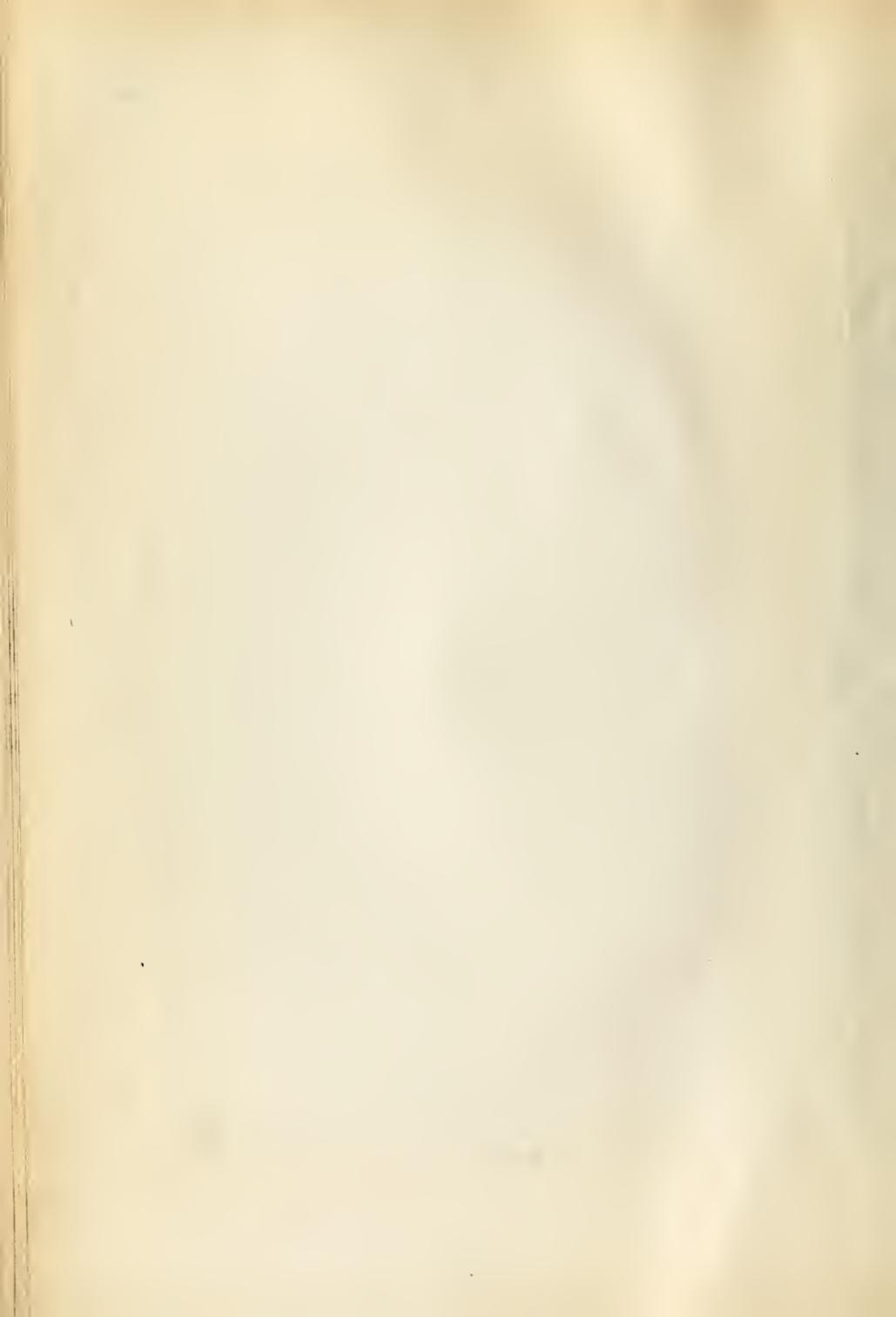
No. Cyl. 6.

Table 5



Test No. 1.		Motor: Chalmers		Date of Test: 10 / 11 / 16											
Object: Economy		Model: 35 A. 3400 R. F. II. Barometer = 29.42 in. Hg		Room Temperature °F											
Constant: Suction		Number: 15039		Gasoline 56.°B. at 60°F.											
No. Cyl. 6.															
Run No.	R.P.M.	Torque lb.	Brake HP	Frict. HP	B.H.P. F.H.P.	Mech. Eff.	M.E.P. Net. lb/sq. in.	Jacket Temp.		Intake Suct. in. Hg	Gasoline lb. per Hour	Therm. Deg. Eff. %			
								In.	Out.						
1	506	45.5	5.7	1.3	7.0	82.3	40.3	60	122	10.0	4:50.3	5.13	1.077	11.7	23
2	792	51.0	10.1	2.2	12.3	82.1	46.2	65	117	10.0	3:32.3	8.48	0.841	15.0	28
3	1006	52.5	13.2	3.1	16.3	81.0	46.6	61	127	10.0	2:32.1	11.63	0.896	14.1	35
4	1145	55.0	15.7	3.7	19.4	80.9	48.7	65	123	10.0	2:12.1	13.65	0.868	14.5	32
5	1420	54.2	19.3	5.3	24.6	78.5	48.1	65	127	10.0	1:47.0	16.82	0.873	14.5	35
6	1663	58.0	24.1	7.1	31.2	77.2	51.4	65	128	10.0	1:23.4	21.59	0.895	14.1	34
7	1960	55.0	26.9	9.9	36.8	73.1	48.7	68	135	10.0	1:13.6	24.44	0.309	13.9	37
8	2247	52.0	29.2	13.2	42.4	68.9	46.1	70	125	10.0	1:03.8	28.20	0.965	13.1	39

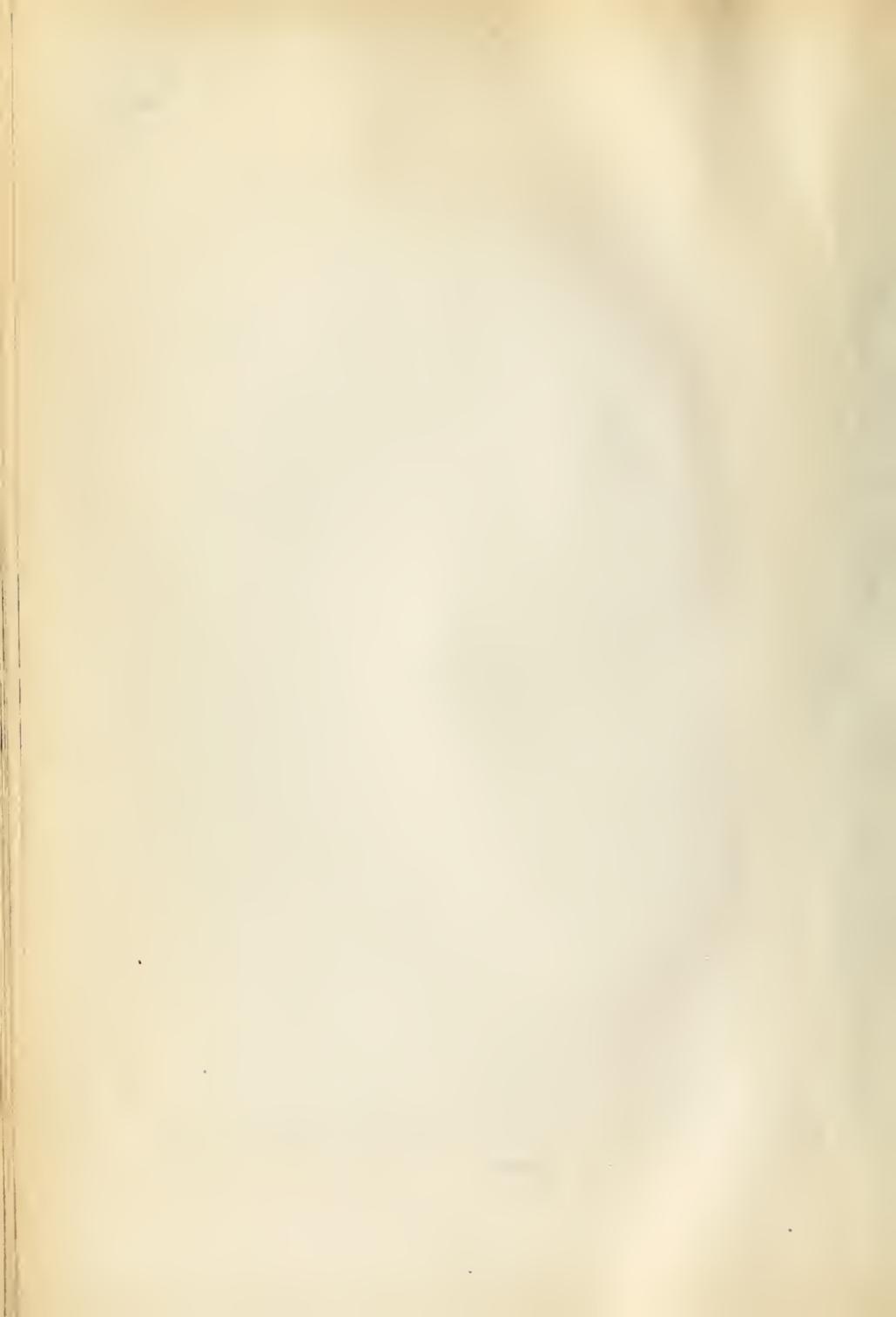
Table 6



Test No. 1. Motor: Ohlmeers Date of Test: 10 / 12 / 16
 Object: Economy Model: 35 A. 3400 R. P. M. Barometer = 29.40 in. Hg
 Constant: Suction Number: 15039 Room Temperature °F.
 No. Cyl. 5. Gasoline 86.5°B. at 64°F.

Run No.	RPM	Torque lb.	Brake HP		Frict. HP		B. HP		Mech. Eff. %	MEP Net. lb./sq. in.	Jacket Temp.		Intake Suct. In. Hg	Gasoline		Therm. Deg. per Hr.	Spent
			HP	HP	HP	HP	In.	Out.			Time	Hour					
1	561	39.0	5.5	1.6	7.1	77.4	34.6	71	135	12.0	.3	3:12.3	5.62	1.029	12.3	31	
2	703	41.5	7.3	2.0	9.3	78.2	36.8	61	128	12.0	.3	2:52.9	6.25	0.857	14.7	33	
3	907	45.2	10.3	2.9	13.2	78.0	40.1	61	132	12.0	.3	2:09.4	8.35	0.813	15.5	36	
4	1144	44.0	12.6	4.1	16.7	75.5	39.0	61	133	12.0	.3	1:52.2	9.62	0.765	16.5	42	
5	1594	45.0	17.9	6.8	24.7	72.5	39.9	61	130	12.0	.3	1:02.1	17.39	0.968	13.0	45	
6	2110	40.0	21.1	10.6	31.7	56.5	35.4	63	140	12.0	.5	1:15.6	23.80	1.130	11.2	47	
7	2324	35.0	20.3	14.4	34.7	56.5	31.3	66	138	12.0	.5	1:03.9	28.20	1.389	9.1	52	

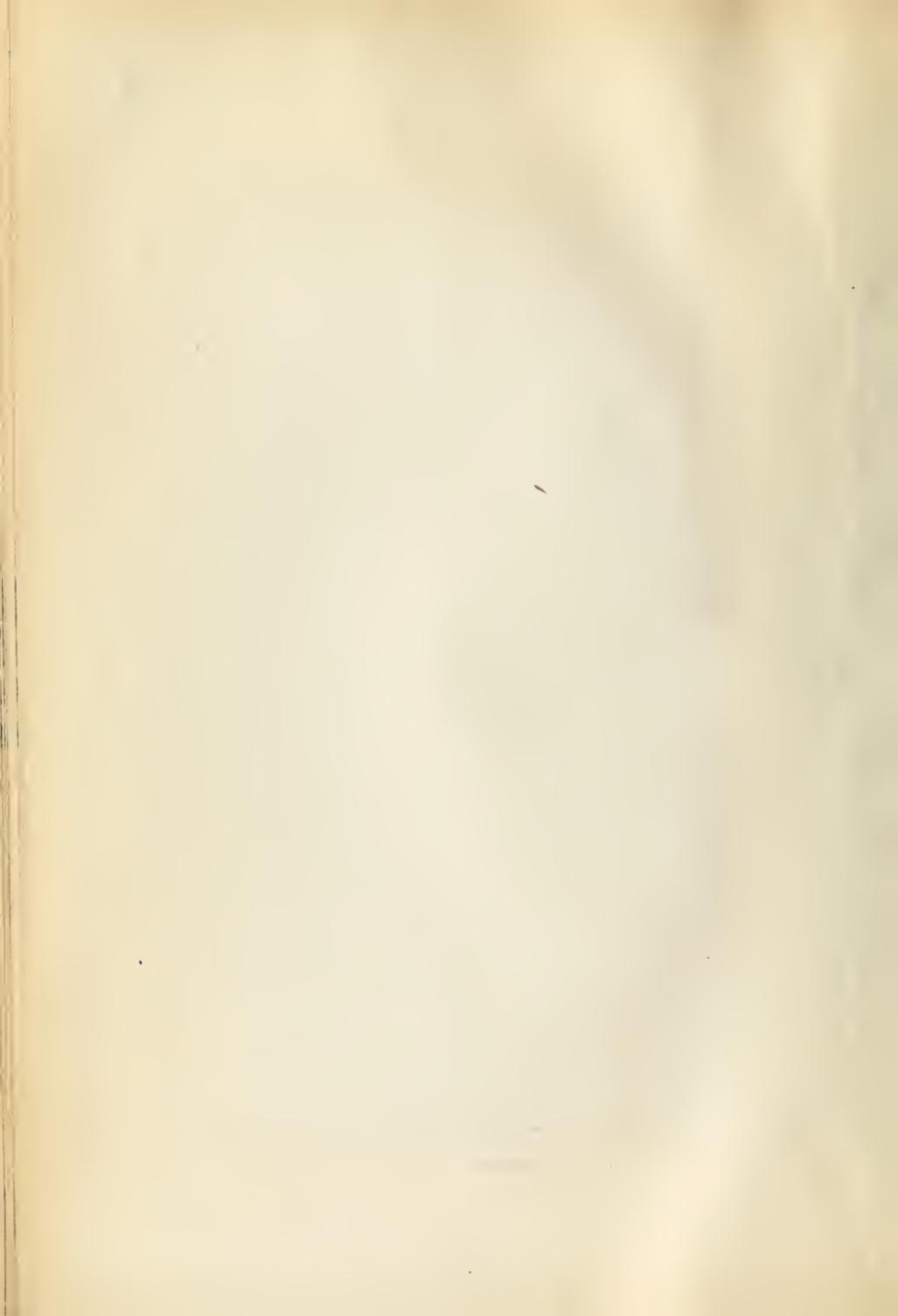
Table 7



Test No. 1. Motor: Chalmers Date of Test: 10 / 12 / 16
 Object: Economy Model: 35 A. 3400 R. P. M. Barometer = 29.40 in. Hg
 Constant: Suction Number: 15039 Room Temperature °F.
 No. Cyl. 6. Gasoline 56.5° B. at 64°F.

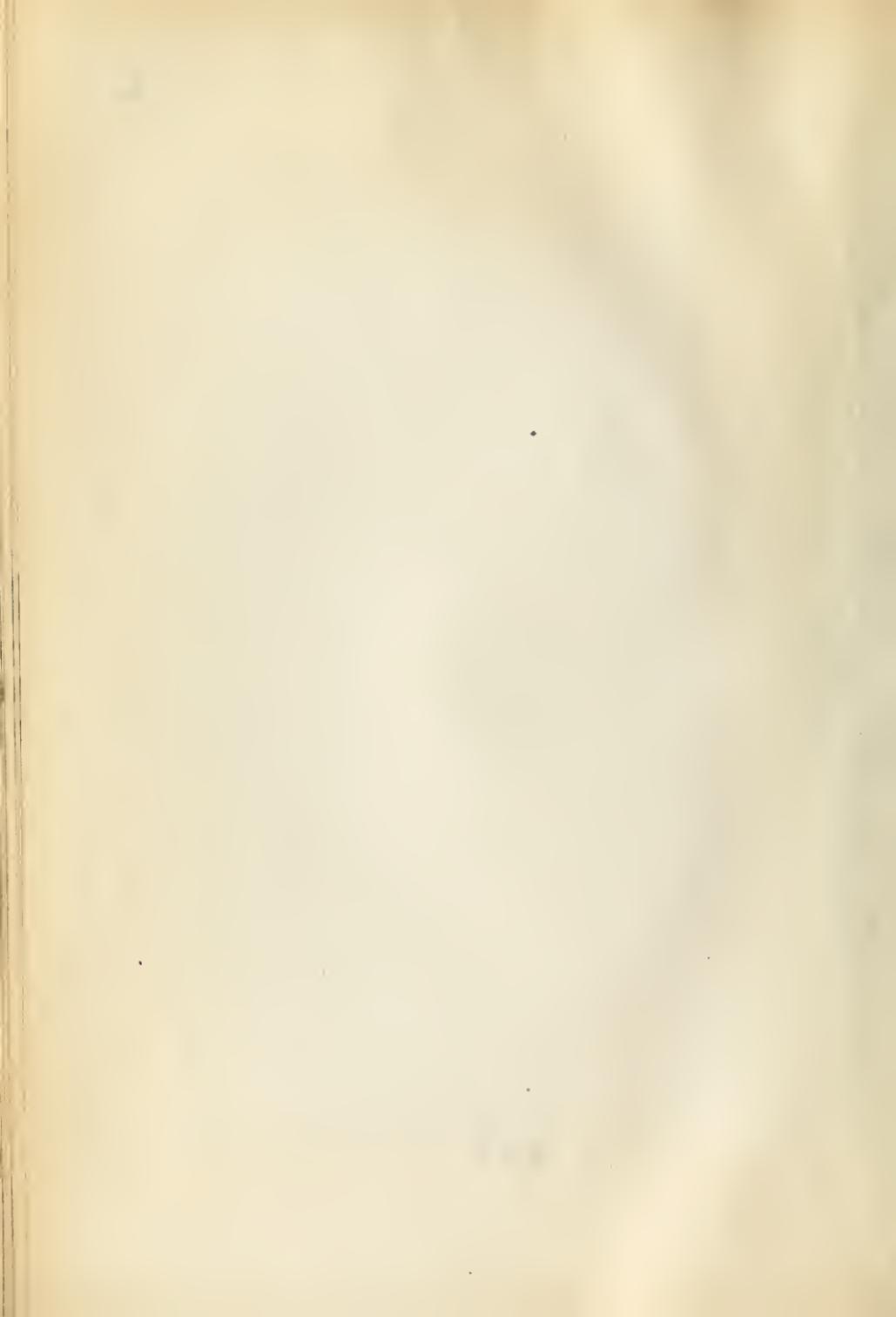
Run No.	RPM	Torque		Brake Frict. HP	B HP	F HP	Mech. Eff. %	M.F.P. Net. lbs/ft.	Jacket Temp.		Inlet Suct. In. Hg	Gasoline		Therm. Deg.		
		lb.	HP						In.	Out.		lb/HR	Time		lb/HP	Spent
1	539	30.7	4.1	1.5	5.6	73.5	27.2	67	150	14.0	.2	3:01.9	3.98	0.960	13.2	35
2	853	36.2	7.7	2.8	10.5	73.5	32.2	61	138	14.0	.2	1:16.1	3.46	1.223	10.3	39
3	1100	37.5	10.3	4.1	14.4	71.7	33.3	61	135	14.0	.3	1:56.2	3.29	0.901	14.0	42
4	1326	37.0	12.3	5.5	17.8	69.0	32.8	61	142	14.0	.3	1:47.3	10.05	0.820	15.4	46
5	1634	37.5	15.3	7.9	23.2	65.3	33.3	62	140	14.0	.3	1:15.1	14.38	0.937	13.5	48
6	2082	36.0	18.7	12.6	31.3	59.6	31.9	63	143	14.0	.3	0:55.6	19.40	1.034	12.2	51
7	2328	29.2	17.0	15.7	32.7	52.1	25.9	68	128	14.0	.4	0:50.9	28.30	1.660	7.6	55

Table 8



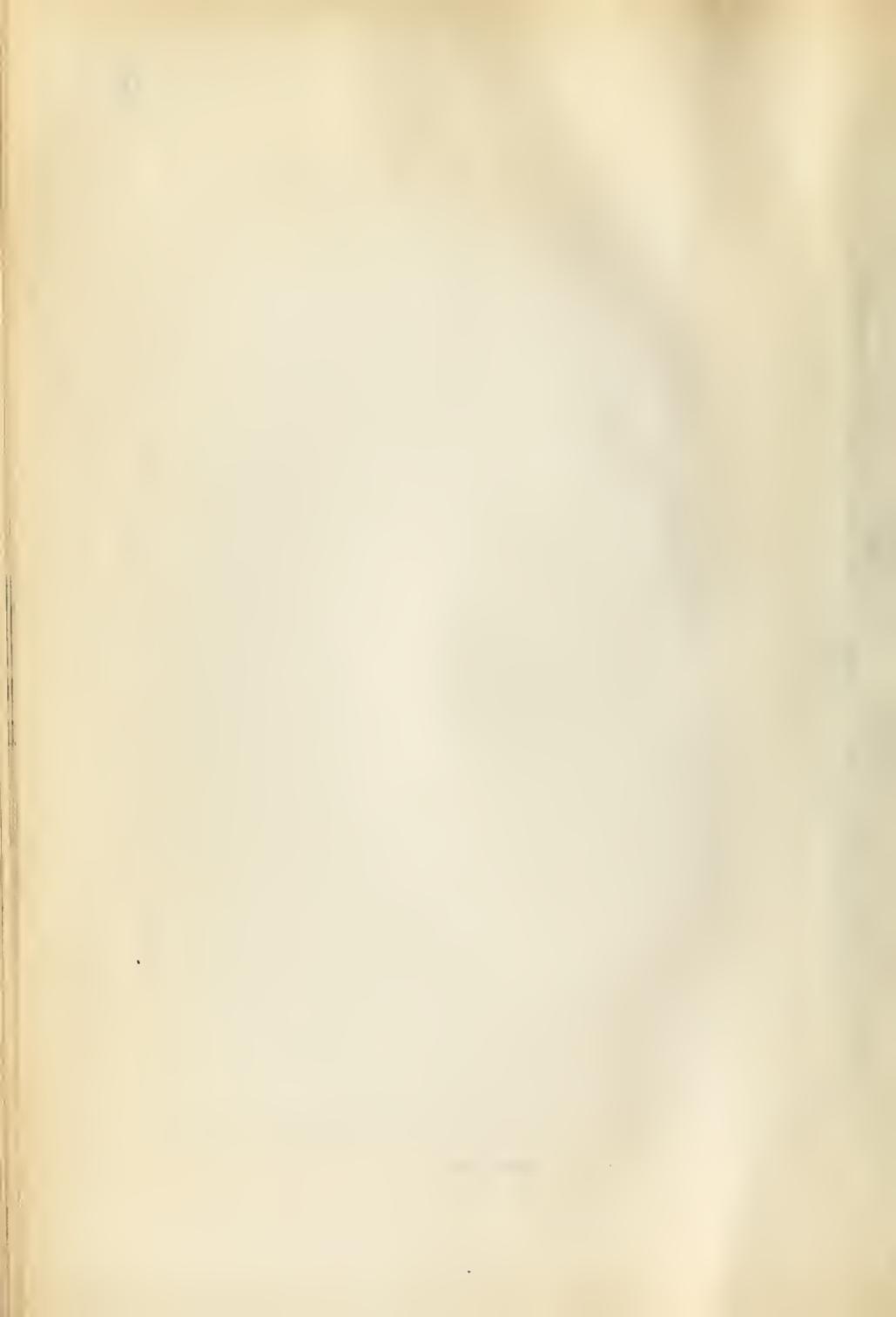
Test No. 1.		Motor: Chalmers		Date of Test: 10 / 12 / 16												
Object: Economy		Model: 35 A. 3400 R. P. M. Barometer = 23.4 in. Hg		Room Temperature °F.												
Constant: Suction		Number: 15029		Gasoline 56.5° B. at 64° F.												
No. Cyl. 6.																
Run No.	R.P.M.	Torque lb.	Brake Frict.		B. HP	F. HP	Mach. Eff. %	Mech. Net. 1/270°	Jacket Temp.		Intake Suct. In. Hg	Gasoline		lb. per Hour	lb. / HP	Therm. Deg. Eff. Spent
			HP	HP					In.	Out.		lb.	Time			
1	657	25.0	4.1	1.7	5.8	70.7	22.2	76	145	16.0	.2	2:15.6	5.31	1.295	8.8	30
2	807	25.0	5.0	2.7	7.7	65.0	22.2	61	128	16.0	.2	2:09.9	5.50	1.092	11.6	41
3	1088	27.0	7.8	4.2	12.0	65.1	23.9	62	138	16.0	.2	1:30.8	7.94	1.011	12.5	48
4	1630	29.2	11.9	8.0	19.9	59.9	25.9	61	134	16.0	.2	1:02.9	11.45	0.960	13.2	58
5	1826	27.0	12.3	10.0	22.3	52.8	23.9	65	137	16.0	.3	1:22.2	13.14	1.069	11.5	61
6	2319	21.5	12.5	16.2	28.7	43.5	19.1	65	139	16.0	.3	0:55.8	19.35	1.551	8.2	63

Table 9



Test No. 1.		Motor: Chalmers		Date of Test: 10 / 12 / 16												
Object: Economy		Model: 35 A. 3400 R. F. M.		Barometer = 29.42 in. Hg												
Constant: Suction		Number: 15039		Room Temperature °F.												
No. Cyl. 6.		Gasoline 56.8° B. at 64°F.														
Run No.	RPM	Torque lb.	Brake HP	Fict. HP	B. HP	F. HP	Mech. Eff.	MEP Net. lb/sq. in.	Jacket Temp. °F.	Inlet Suct. In. Hg	Gasoline		Therm. Deg. per Hr.	Therm. Deg. per Hr.	Est. Spark	
											lb. per Hour	lb. per Hour				
1	646	12.5	2.1	2.1	4.2	50.0	11.1	63	128	18.0	.5	3:20.0	5.40	2.895	4.9	33
2	1036	14.7	3.8	4.0	7.8	48.9	13.1	61	146	18.0	.5	2:58.2	6.83	1.790	7.1	45
3	1643	16.0	6.6	8.4	15.0	43.9	14.2	63	140	18.0	.5	3:10.8	9.44	1.434	8.8	60
4	2362	14.2	8.4	17.0	25.4	33.1	12.6	63	158	18.0	.5	2:16.6	13.20	1.568	8.0	65

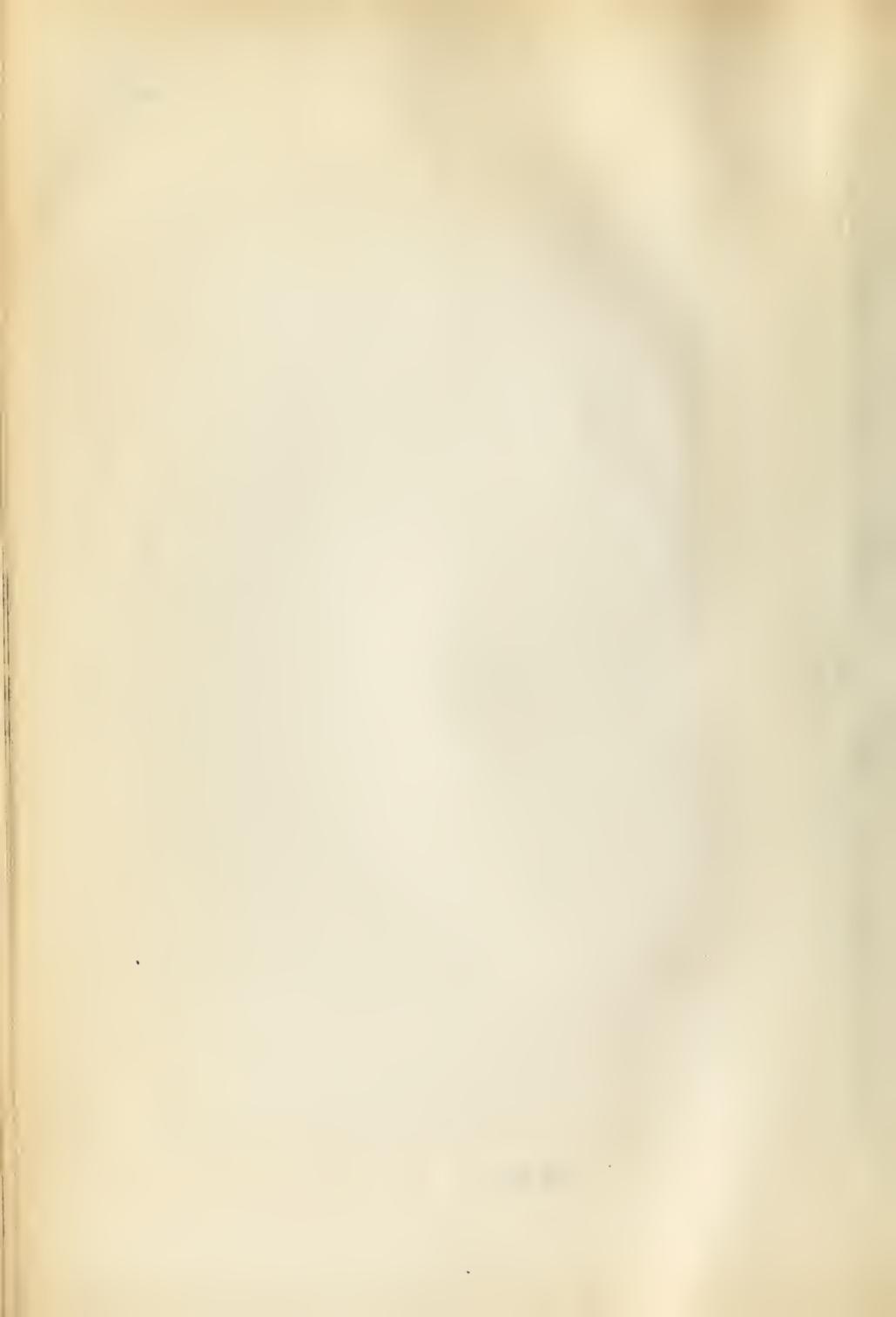
Table 10



Test No. 1. Revised. Motor: Chalmers Date of Test:
 Object: Economy Model: 35 A, 3400 R, P. M. Barometer = in. Hg
 Constant: Suction Number: 15039 Room Temperature °F.
 No. Cyl. 6. Gasoline °B. at °F.

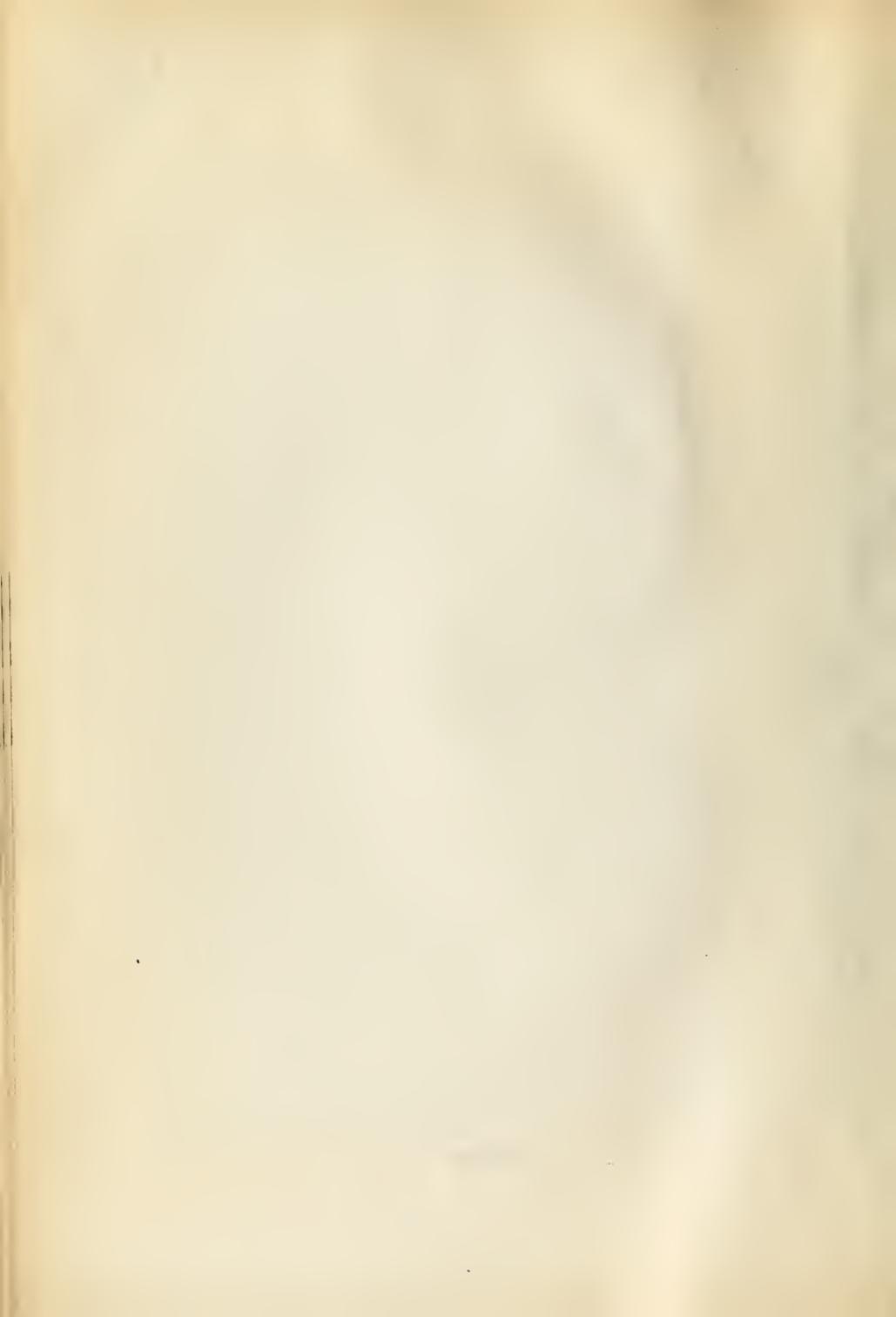
Run No.	R.P.M.	Torque		Brake/Frict.		Mech. Eff. %	M.F.P. Net lb/ft ²	Jacket Temp.		Inlet Suct. In. Hg	Gasoline		lb/HP Hour	Therm. Deg.
		lb.	HP	HP	HP			In.	Out.		In. Hg	lb. Time		
1	400	82.0	8.2	0.6	8.8	93.2	72.6			1.2				
2	600	93.3	14.0	1.1	15.1	92.6	82.6			0.8			12.7	0.908 13.9 12
3	800	95.9	19.2	1.7	20.9	91.9	85.0			0.9			13.6	0.708 17.8 16
4	1000	95.5	23.9	2.4	26.3	90.9	84.6			1.2			16.8	0.718 17.6 19
5	1200	94.2	28.3	3.4	31.7	89.2	83.4			1.6			21.0	0.744 17.0 22
6	1400	90.6	31.7	4.4	36.1	87.8	80.3			2.2			24.8	0.783 16.1 24
7	1600	86.7	34.7	5.8	40.5	85.6	76.8			2.9			28.0	0.807 15.6 27
8	1800	82.9	37.3	7.5	44.8	83.2	73.4			3.6			30.4	0.814 15.5 29
10	2000	77.0	38.5	9.4	47.9	80.4	68.2			4.4			31.9	0.829 15.2 30
10	2200	70.0	38.5	11.7	50.2	76.7	62.0			5.3			34.9	0.906 13.9 32
11	2400	61.7	37.0	14.2	51.2	72.2	54.7			6.1			36.8	0.994 12.7 32

Table 11



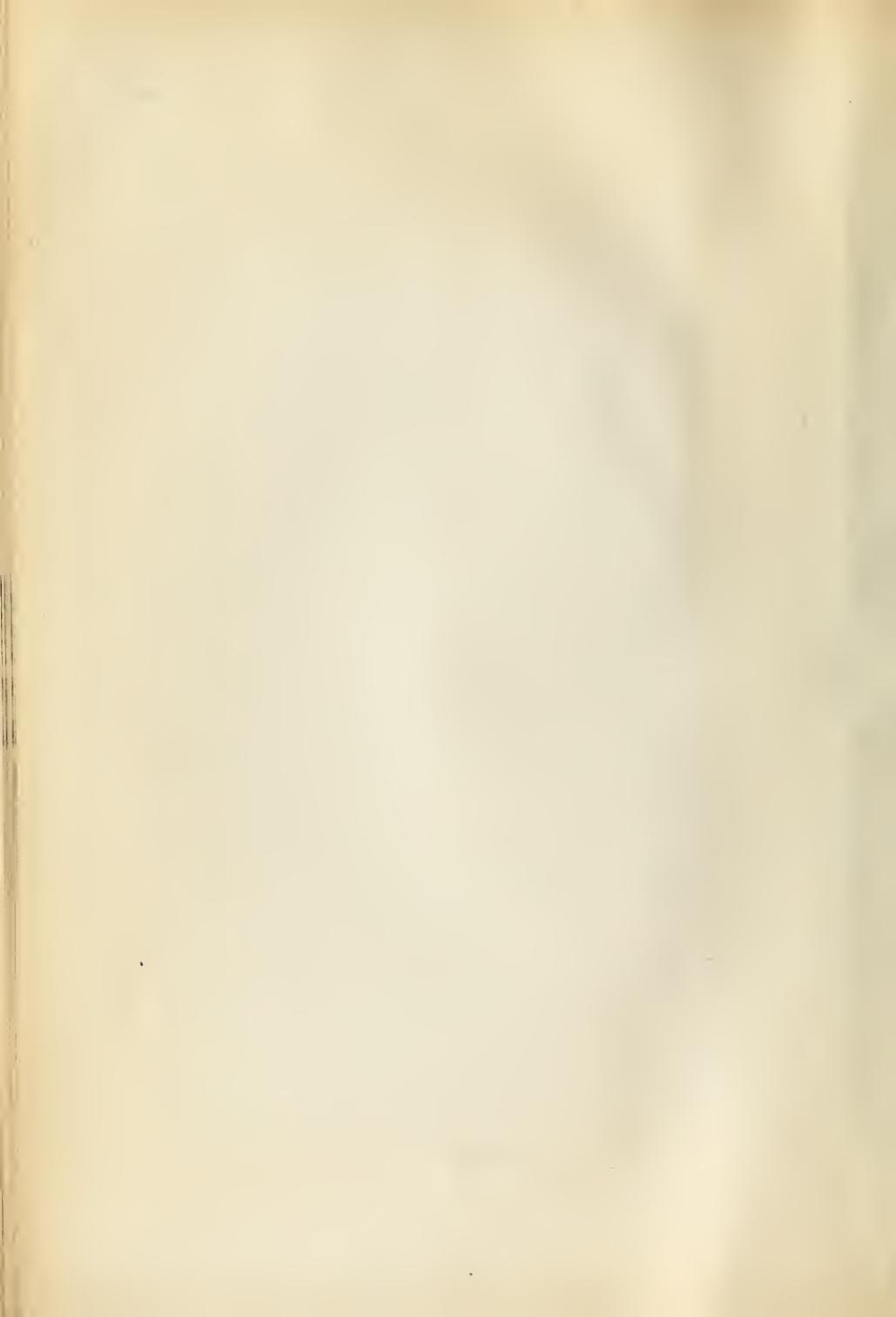
Test No. 1. Revised		Motor: Chalmers		Date of Test:						
Object: Economy		Model: 35 A, 3400 B, I. M.		Barometer = in. Hg						
Constant: Station		Number: 15039		Room Temperature °F.						
		No. Cyl. 6.		Gasoline °B. at °F.						
Run No.	R.P.M.	Torque		Mech. Eff. %	M.F.P. Net. lb/ft ²	Jacket Temp. In. Out.	Intake Suct. in. Hg	Gasoline		Therm. Deg
		Brake HP	Frict. HP					B.T.P. F. HP	lb per Hour	
1	400	79.0	7.0	0.6	7.6	92.1	62.0			
2	600	84.6	12.7	1.2	13.9	91.3	74.9		12.3	0.958 13.0 16
3	800	90.4	18.1	1.8	19.9	91.0	80.1		15.0	0.828 15.2 20
4	1000	92.4	23.1	2.6	25.7	89.9	81.8		18.0	0.779 16.2 23
5	1200	92.3	27.7	3.5	31.2	88.7	81.7		21.4	0.773 16.3 25
6	1349	91.4	30.8	4.2	35.0	89.0	81.0		24.1	0.783 16.1

Table 12



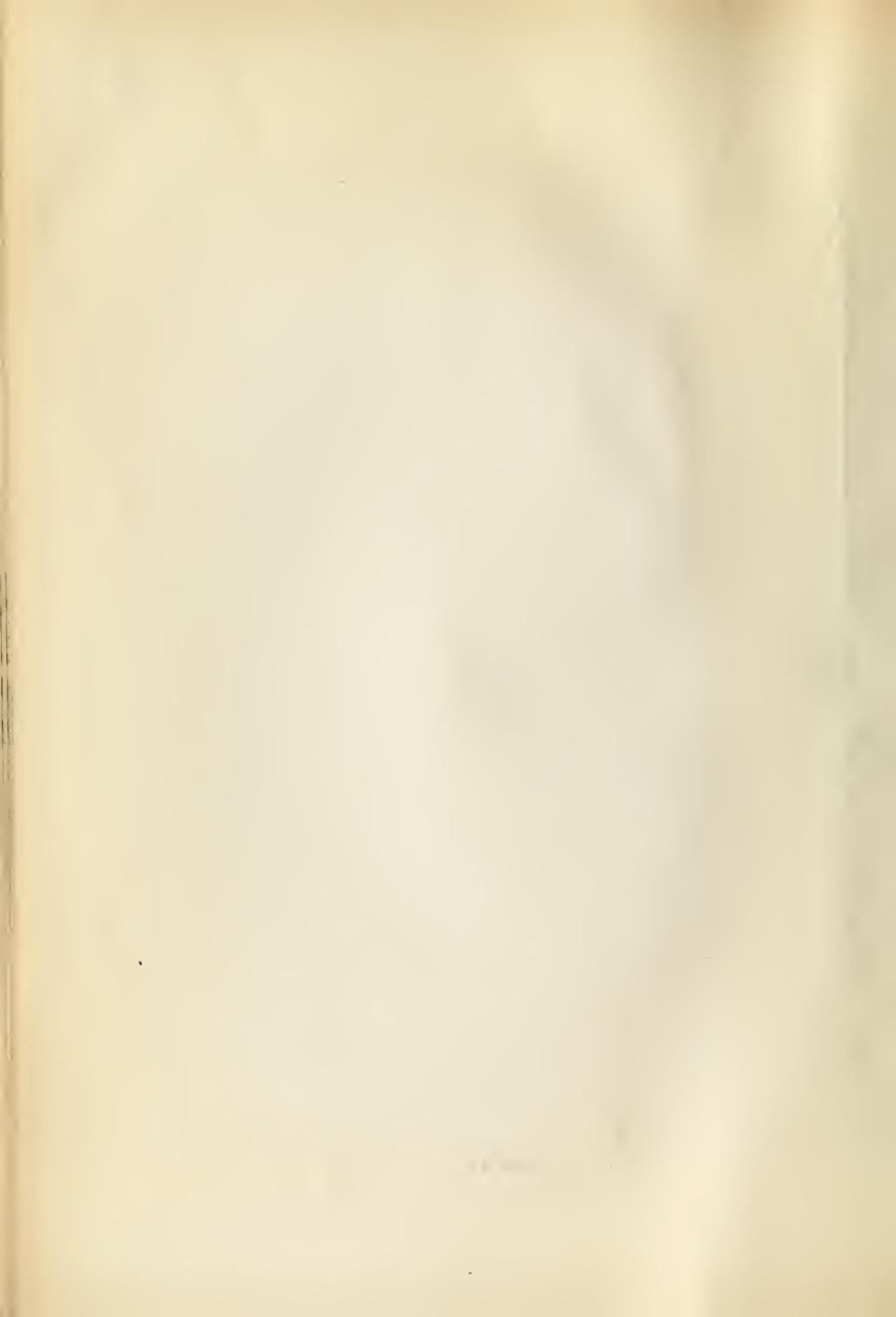
Test No. 1, Revised		Motor: Chalmers		Date of Test:										
Object: Economy		Model: 35 A, 3400 R. V. 2, Barameter =		in. Hg										
Constant: Suction		Number: 15039		Room Temperature °F.										
		No. Cyl. 6.		Gasoline °B. at °F.										
Run No.	R.P.M.	Torque lb.	Brake HP	Frict. HP	B.H.P. F.H.P.	Mech. Eff. %	MEP Net lb./sq. in.	Jacket Temp.		Inlet Suct. In. Hg	Gasoline lb. per Hr.	Therm. Deg. Eff. Spark		
								In.	Out.					
1	400	60.0	6.0	0.8	6.8	88.2	53.2			4.0				
2	500	75.9	11.4	1.2	12.6	90.5	67.2			4.0	9.3	0.816	15.5	18
3	800	81.9	16.4	1.9	18.3	89.5	72.5			4.0	13.9	0.847	14.9	19
4	1000	84.7	21.2	2.6	23.8	89.0	75.1			4.0	17.7	0.855	15.1	21
5	1200	85.3	25.6	3.5	29.1	88.0	75.6			4.0	21.0	0.820	15.4	23
6	1400	84.6	29.6	4.6	34.2	86.5	74.9			4.0	24.0	0.811	15.6	25
7	1600	83.3	33.3	6.0	39.3	84.7	73.8			4.0	26.8	0.799	15.8	28
8	1800	81.3	36.6	7.8	44.4	82.5	72.1			4.0	29.6	0.809	15.6	31
9	1900	80.0	38.0	9.9	47.9	79.4	70.9			4.0	30.8	0.810	15.6	34

Table 13



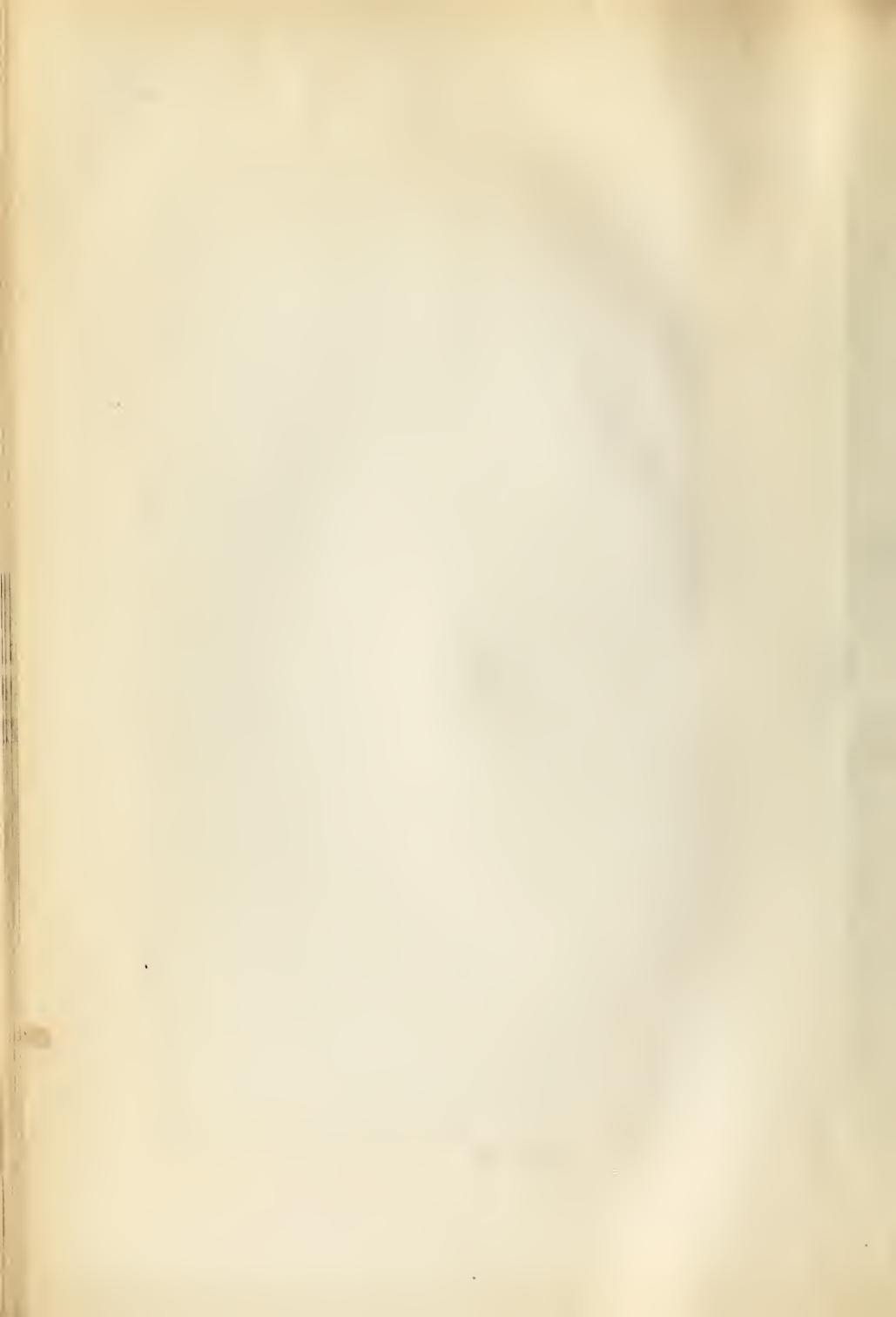
Test No. 1. Revised		Motor: Chalmers		Date of Test:					
Object: Economy		Model: 35 A. 3400 R. F. M. Barometer = in. Hg							
Constant: Suction		Number: 15039		Room Temperature °F.					
		No. Cyl. 6.		Gasoline °B. at °F.					
Run No.	RPM.	Torque		Mech. Eff. %	Intake Suct. In. Hg	Jacket Temp. In.	Gasoline lb. per Hour	Therm. Deg. Eff. Gasoline	
		Brake Frict. HP	HP						Intake Suct. In. Hg
1	400	52.0	5.2	0.8	6.0	46.1	10.0	1.926	6.6
2	600	66.0	9.9	1.4	11.3	58.5	12.3	1.242	10.2
3	800	73.0	14.6	2.0	16.6	64.7	14.2	0.972	13.0
4	1000	75.5	18.9	2.8	21.7	66.9	16.4	0.868	14.5
5	1200	75.9	22.8	3.8	26.6	67.2	18.7	0.820	15.4
6	1400	76.0	26.6	4.8	31.4	67.3	21.0	0.789	16.0
7	1600	74.5	29.8	5.3	35.1	66.0	23.9	0.802	15.7
8	1800	72.7	32.7	8.1	40.8	64.4	27.4	0.837	15.1
9	2000	70.2	35.1	10.2	45.3	62.2	32.2	0.924	13.7
10	2200	65.9	36.8	12.6	49.4	59.2	38.5	1.047	12.1
11	2380	62.4	37.1	15.0	52.1	55.2	45.0	1.213	10.4

Table 14



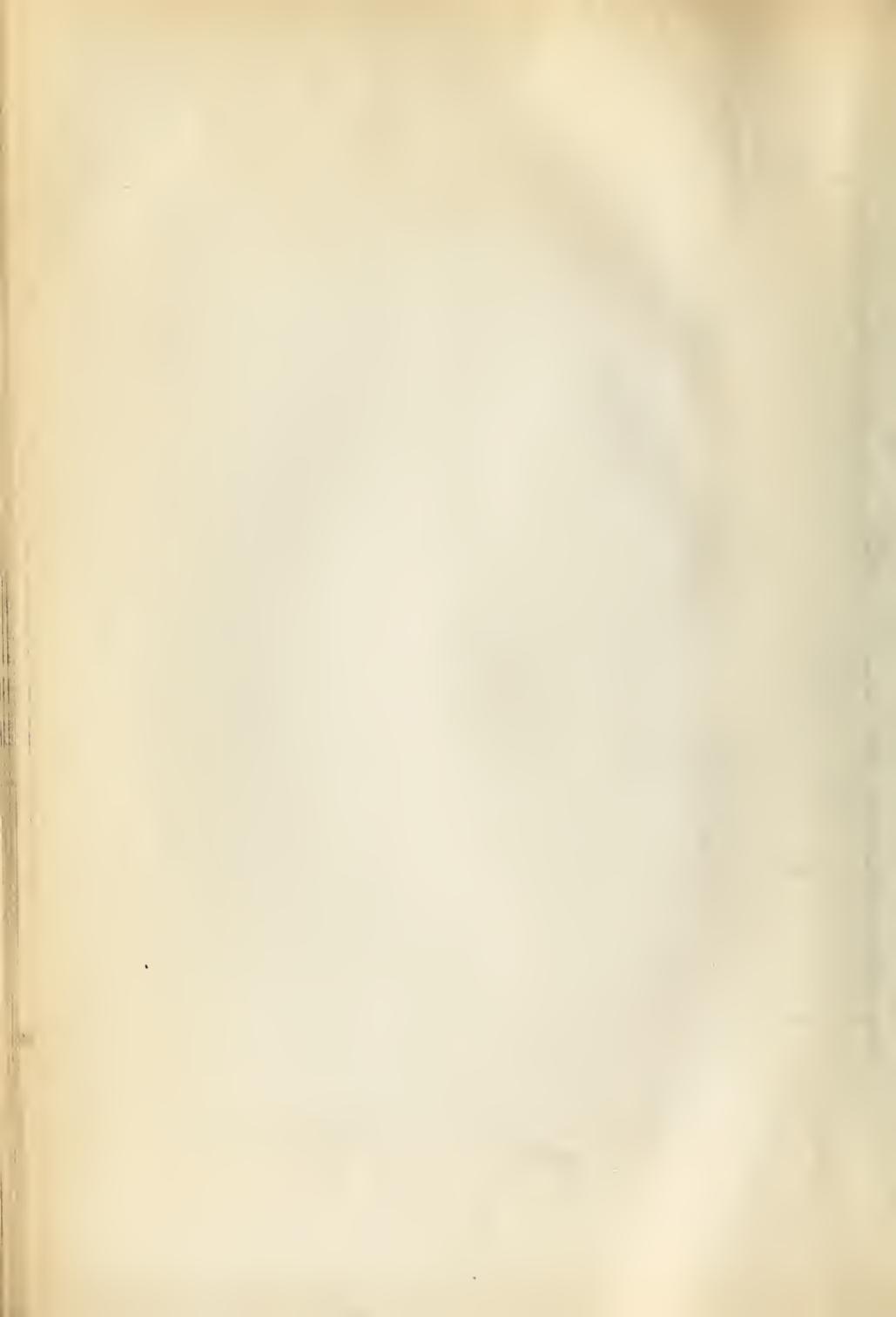
Test No. 1. Revised		Motor: Chalmers		Date of Test:									
Object: Deonov		Model: 35 A, 3400 R. P. M. Barometer =		in. Hg									
Constant: Suction		Number: 15039		Room Temperature °F.									
		No. Cyl. 5.		Gasoline °B. at °F.									
Run No.	RPM	Torque lb.	Brake Frict. HP	B. HP F. HP	Mech. Eff. %	MEP Net lb/sq. in.	Jacket Temp.		Intake Suct. in. Hg	Gasoline lb. per Hour	Therm. Deg. Eff. %		
							In.	Out.				lb. Time	
1	400	43.0	4.3	0.9	5.2	82.7	39.1		8.0	5.0	1.163	10.9	11
2	600	56.0	8.4	1.4	9.8	85.7	43.5		8.0	6.0	0.714	17.7	14
3	800	62.4	12.5	2.0	14.5	86.2	55.3		8.0	8.0	0.641	19.7	17
4	1000	66.3	16.6	2.9	19.5	85.1	58.7		8.0	11.0	0.553	19.1	21
5	1200	68.7	20.6	3.9	24.5	84.1	60.9		8.0	14.6	0.708	17.8	25
6	1400	69.4	24.3	5.0	29.3	82.9	61.5		8.0	19.0	0.782	16.1	28
7	1600	69.0	27.6	6.4	34.0	81.2	61.1		8.0	22.5	0.815	15.5	31
8	1800	67.2	30.2	8.1	38.3	78.8	59.6		8.0	26.0	0.862	14.6	32
9	2000	65.5	31.8	10.2	42.0	75.7	56.4		8.0	28.5	0.896	14.1	34
10	2200	59.5	32.7	12.5	45.2	72.3	52.7		8.0	30.8	0.942	13.4	37
11	2400	54.5	32.7	15.1	47.8	68.4	48.3		8.0	32.0	0.978	12.9	40

Table 15



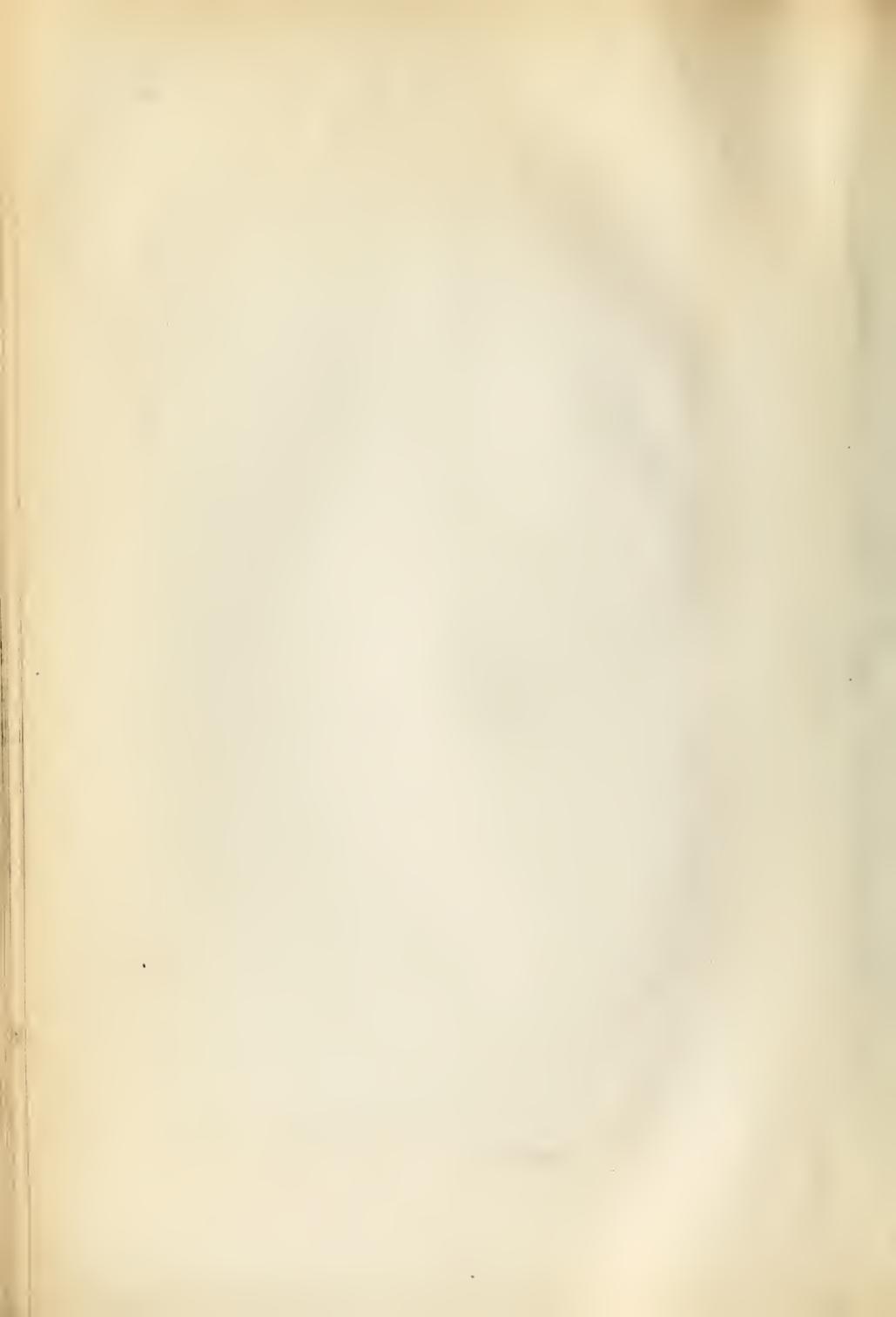
Test No. 1, Revised		Motor: Chalmers		Date of Test:							
Object: Economy		Model: 35 A, 3400 R. P. M.		Barometer = in. Hg							
Constant: Suction		Number: 15039		Room Temperature °F							
		No. Cyl. 6.		Gasoline °B. at °F.							
Run No.	R.P.M.	Brake Horse Power	Frict. Horse Power	Mech. Eff. %	MEP Net (lb./sq. in.)	Jacket Temp. In.	Jacket Temp. Out.	Intake Suct. in. Hg	Gasoline lb. per Hour	lb./HP Therm. Deg	Therm. Deg Eff. Suct.
1	420	42.0	4.4	1.0	5.4	81.5	37.2	10.0	5.5	1.377	9.2
2	500	48.0	7.2	1.5	8.5	82.7	42.5	10.0	7.0	0.972	13.0
3	800	52.9	10.6	2.3	12.9	82.1	46.9	10.0	9.1	0.859	14.7
4	1000	55.2	13.8	3.1	16.9	81.6	48.9	10.0	11.8	0.855	14.8
5	1200	56.2	16.9	4.0	20.9	80.8	49.8	10.0	14.7	0.870	14.5
6	1400	57.2	20.0	5.2	25.2	79.3	50.6	10.0	17.7	0.865	14.3
7	1600	57.0	22.7	5.6	29.3	77.5	50.3	10.0	20.2	0.890	14.2
8	1800	56.0	25.2	8.3	33.5	75.2	49.6	10.0	23.0	0.912	13.8
9	2000	54.2	27.1	10.4	37.5	72.2	48.1	10.0	25.3	0.933	13.5
10	2200	50.0	27.5	12.7	40.2	68.4	44.3	10.0	27.8	1.012	12.5
11	2360	48.2	25.5	14.7	40.2	63.4	35.3	10.0	29.2	1.145	11.0

Table 16



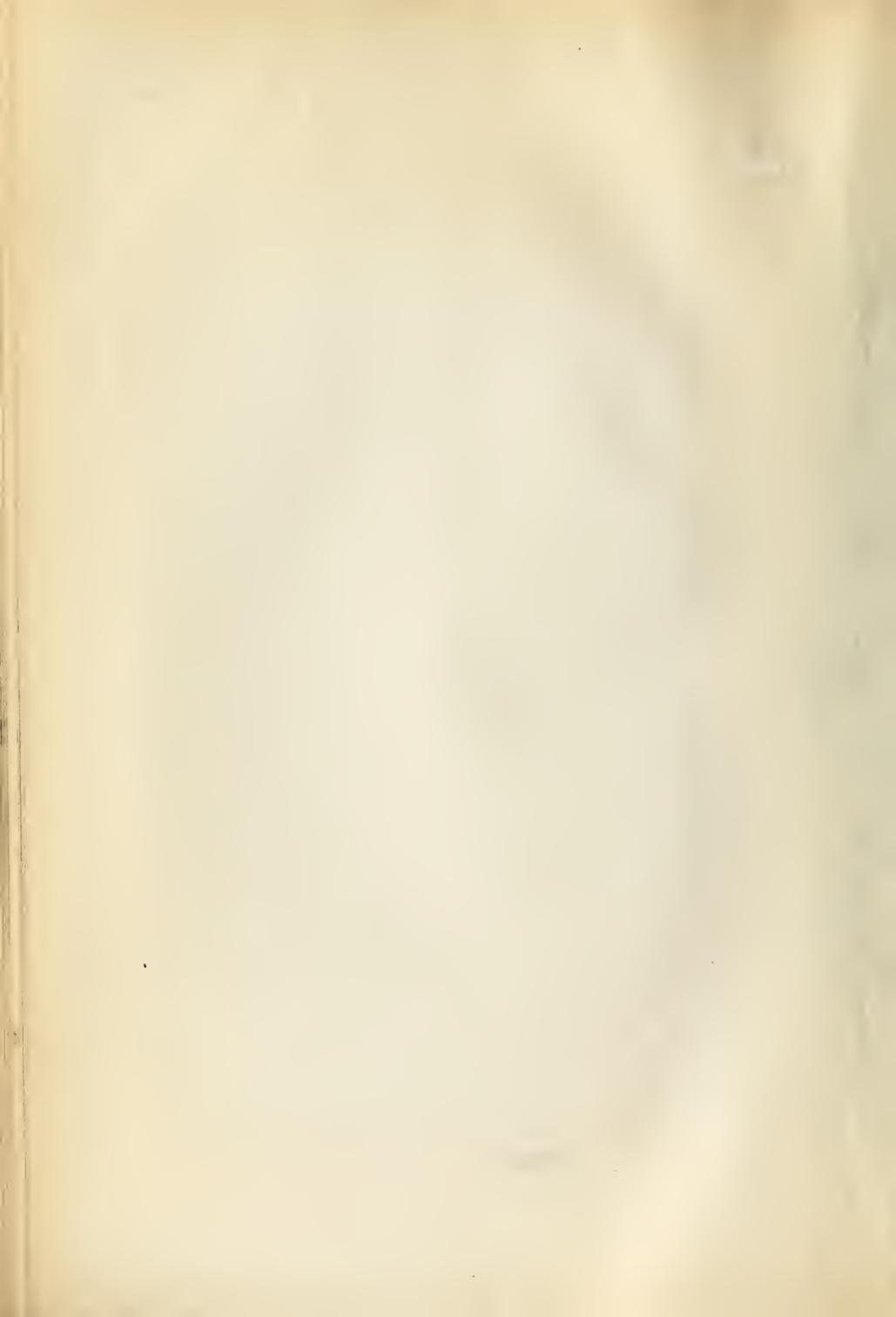
Test No. 1. Revised.		Motor: Chalmers		Date of Test:									
Object: Economy		Model: 35 A. 5400 R. P. M. Barometer = in. Hg											
Constant: Suction		Number: 15039		Room Temperature °F.									
		No. Cyl. 6.		Gasoline °B. at °F.									
Rev. No.	R.P.M.	Torque lb.	Brake HP	Frict. HP	B.H.P.	Mech. Eff. %	M.E.P. lb./sq. in.	Jacket Temp. In.	Out. In.	Intake Suct. In. Hg	Gasoline lb. per Hour	lb./HP per Hk. Eff.	Therm. Deg. Eff. Spark
1	450	32.2	3.7	1.3	5.0	74.0	28.5			12.0			
2	600	40.0	6.0	1.7	7.7	77.9	35.4			12.0	6.1	1.018	12.4
3	800	44.0	8.8	2.4	11.2	78.6	39.0			12.0	6.7	0.761	16.6
4	1000	46.0	11.5	3.3	14.8	77.7	40.8			12.0	8.0	0.696	18.1
5	1200	46.3	13.9	4.3	18.2	76.3	41.0			12.0	9.9	0.713	17.7
6	1400	46.0	16.1	5.4	21.5	74.8	40.8			12.0	12.2	0.757	16.7
7	1600	45.0	18.0	6.8	24.8	72.6	39.9			12.0	15.1	0.639	15.1
8	1800	43.6	19.6	8.5	28.1	69.7	38.6			12.0	16.7	0.954	13.2
9	2000	41.6	20.8	10.5	31.3	66.5	36.8			12.0	22.5	1.082	11.77
10	2200	38.2	21.0	12.8	33.8	62.1	33.9			12.0	26.7	1.272	9.9
11	2330	34.3	20.0	14.4	34.4	58.1	30.4			12.0	29.2	1.461	8.6

Table 17



Test No. 1., Revised		Motor: Chalmers		Date of Test:							
Objct: Economy		Model: 35 A. 3400 K. P. M. Barometer = in. Hg		Room Temperature °F.							
Constant: Suction		Number: 15039.		Gasoline °B. at °F.							
No. Cyl. 6.											
Run No.	RPM.	Torque Brake		F.P.	Mech. Eff.	Net Eff.	Jacket Temp.	Intake Succ.	Gasoline		lb./hr Therm Deg.
		lb.	HP						HP	lb. Time	
1	520	27.7	3.6	1.5	5.1	70.5	24.6	14.0	3.8	1.057	12.0 34
2	600	31.3	4.7	1.8	6.5	72.3	27.8	14.0	4.0	0.851	14.8 35
3	800	37.0	7.4	2.6	10.0	74.0	32.8	14.0	5.4	0.731	17.3 38
4	1000	33.2	3.8	3.6	13.4	73.1	34.8	14.0	7.4	0.755	16.7 41
5	1200	40.0	12.0	4.7	16.7	71.8	35.4	14.0	9.9	0.826	15.3 44
6	1400	40.0	14.0	6.0	20.0	70.0	35.4	14.0	12.2	0.872	14.5 46
7	1600	39.8	15.3	7.6	23.5	67.6	35.3	14.0	14.9	0.937	13.5 47
8	1800	39.1	17.6	9.5	27.1	64.9	34.6	14.0	17.4	0.989	12.8 48
9	2000	37.6	16.8	11.7	30.5	61.7	33.3	14.0	19.9	1.060	11.9 50
10	2200	33.6	18.6	14.1	32.6	56.7	29.8	14.0	22.0	1.190	10.6 52
11	2340	23.2	17.1	15.9	33.0	51.8	25.9	14.0	23.6	1.362	9.1 55

Table 18

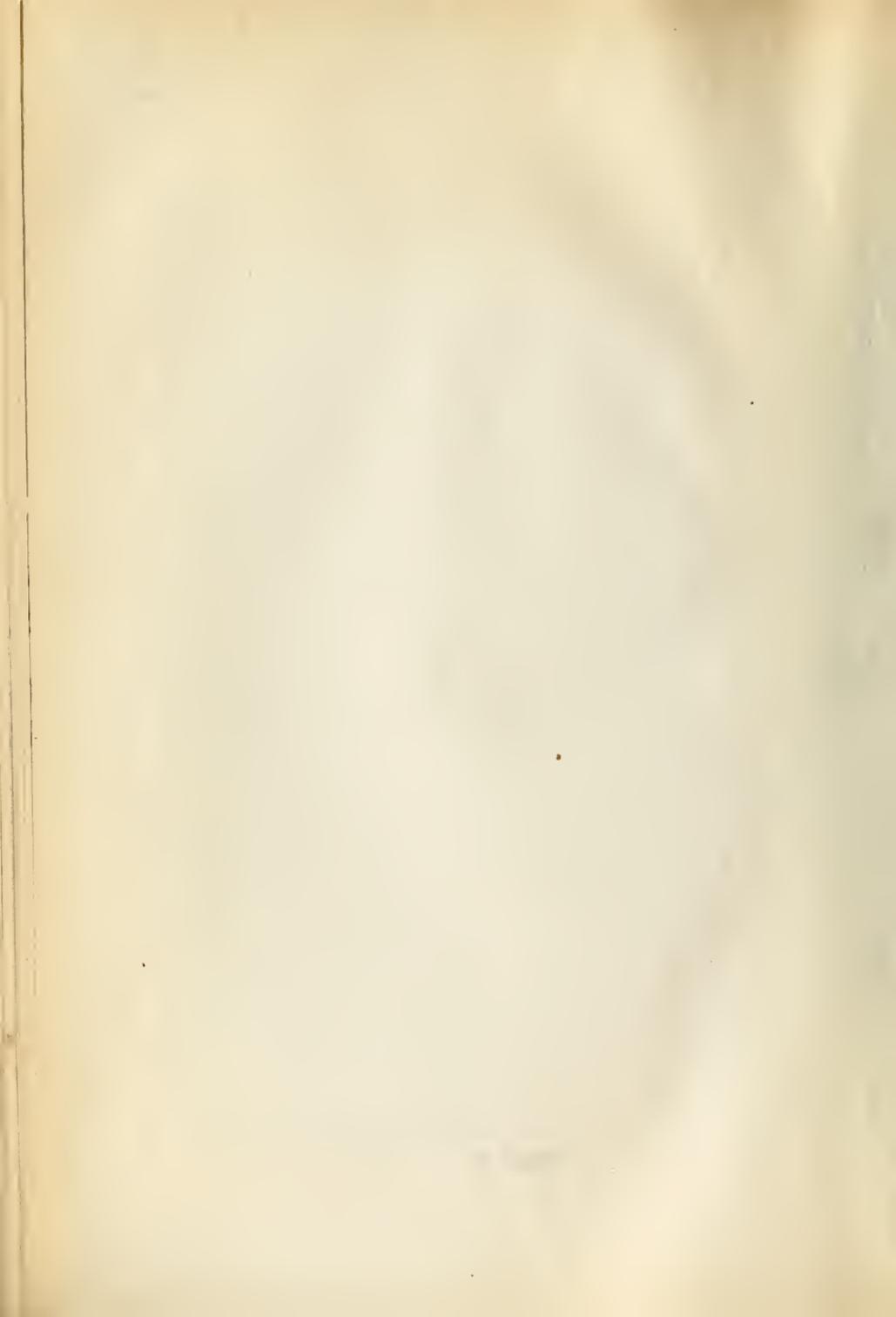


Test No. 1, Revised		Motor: Chalmers		Date of Test:											
Object: Economy		Model: 35 A, 3400 S. I. M. Barometer = in. Hg		Room Temperature °F.											
Constant: Suction		Number: 15339		Gasoline °B. at °F.											
No. Cyl. 5.															
Run No.	RPM	Torque lb.	Brake HP	Frict. HP	B. HP	F. HP	Mech. Eff. %	MEP Net lb/sq. in.	Jacket Temp.		Intake Suct. in. Hg	Gasoline lb. per Hr.	lb. per Hr. Therm. Deg. Cent.		
									In.	Out.					
1	600	24.0	3.6	1.9	5.5	65.4	21.3	16.0			16.0	5.4	1.502	8.4	27
2	800	28.5	5.7	2.7	8.4	67.9	25.3	16.0			16.0	5.7	1.000	12.6	39
3	1000	30.0	7.5	3.7	11.2	67.0	26.6	16.0			16.0	6.1	0.813	15.5	47
4	1200	30.2	9.1	4.3	14.0	65.0	26.8	16.0			16.0	7.2	0.791	16.0	51
5	1400	29.7	10.4	6.2	16.6	62.7	26.3	16.0			16.0	9.0	0.866	14.5	55
6	1600	28.7	11.5	7.5	19.3	59.6	25.5	16.0			16.0	11.0	0.957	13.2	57
7	1800	27.1	12.2	9.8	22.0	55.4	24.0	16.0			16.0	13.2	1.082	11.7	58
8	2000	25.4	12.7	12.0	24.7	51.4	22.5	16.0			16.0	15.7	1.238	10.2	60
9	2200	23.1	12.7	14.5	27.3	46.5	20.5	16.0			16.0	18.0	1.419	8.9	62
10	2380	20.2	12.0	17.1	29.1	41.2	17.9	16.0			16.0	20.0	1.569	7.6	63

Table 19

Test No. 1. Revised		Motor: Chalmers		Date of Test:							
Object: Economy		Model: 35 A. 3400 R. P. M. Barometer = in. Hg									
Constant: Suction		Number: 15039.		Room Temperature °F.							
		No. Cyl. 6.		Gasoline °B. at °F.							
Run No.	RPM.	Torque		Mech. Eff. %	MEP Net. lb/p ²	Jacket Temp.		Intake Suct. In. Hg	Gasoline lb per Hour	lb/HP	Therm. Deg. Eff. Sash
		Brake HP	Frict. HP			B. HP	F. HP				
1	640	8.8	1.4	2.1	3.5	40.0	7.8	18.0	5.3	3.790	3.3 38
2	800	14.0	2.8	2.8	5.6	50.0	12.4	18.0	6.0	2.146	5.9 43
3	1000	17.6	4.4	3.8	8.2	53.7	15.6	18.0	7.0	1.592	7.9 47
4	1200	19.4	5.8	5.0	10.8	53.7	17.1	18.0	8.0	1.382	9.1 51
5	1400	20.0	8.0	6.4	13.4	52.3	17.7	18.0	9.0	1.288	9.8 54
6	1600	20.0	8.0	8.1	16.1	49.5	17.7	18.0	10.0	1.250	10.1 57
7	1800	19.4	8.7	10.0	18.7	46.5	17.1	18.0	11.0	1.264	10.0 60
8	2000	18.0	9.0	12.3	21.3	42.3	15.9	18.0	12.0	1.333	9.5 62
9	2200	16.2	8.9	14.7	23.6	37.7	14.4	18.0	12.8	1.438	8.8 65
10	2400	13.3	8.0	17.4	25.4	31.5	11.8	18.0	13.0	1.625	7.8 67

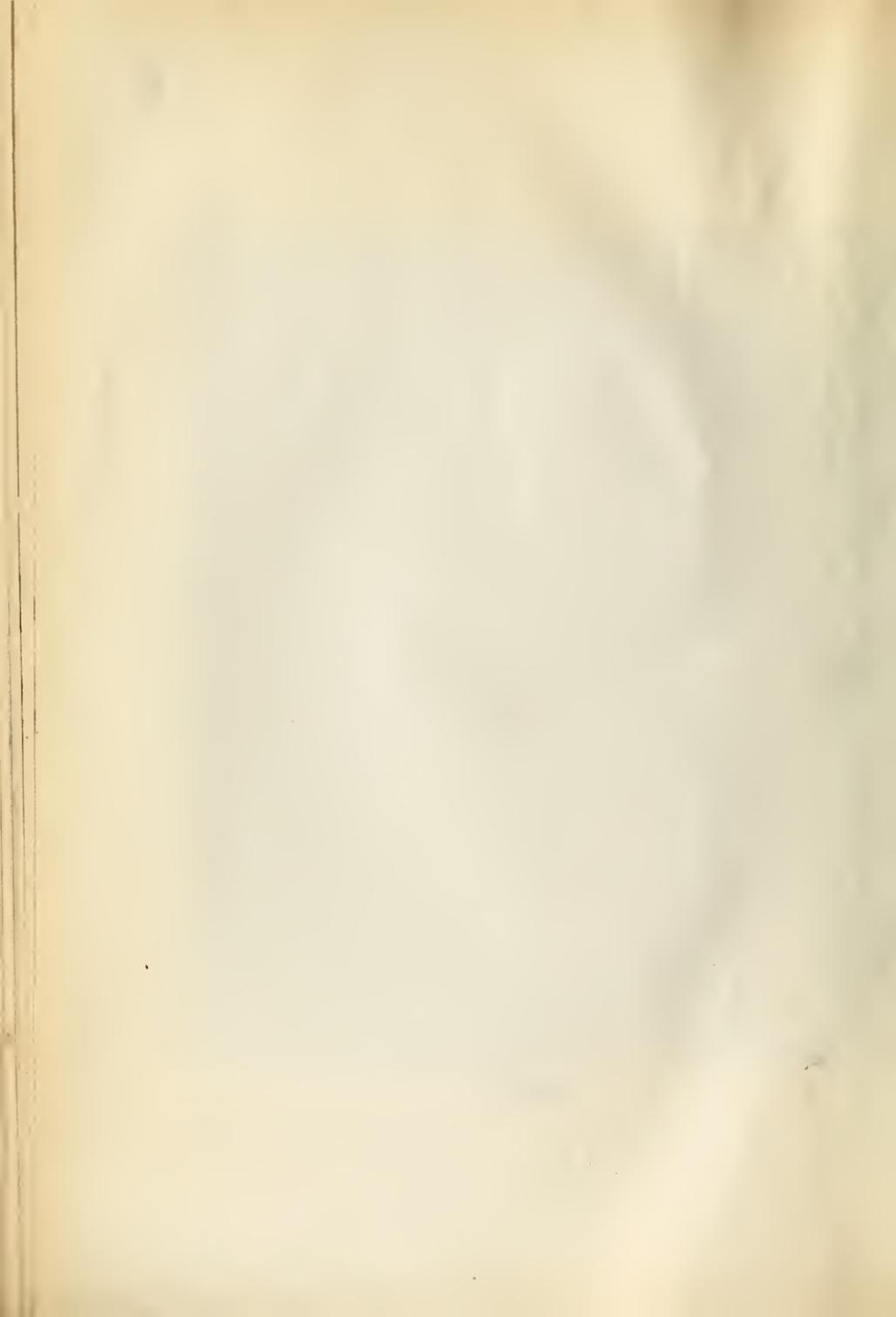
Table 20



CHARLES 3400 R.P.M. MOTOR
 Interpolation data for constant R.P.M. curves for
 lines of equal lbs. gasoline/3.H.P./hr.

600 R.P.M.				800 R.P.M.			
Suction W.O.	B.H.P.	Lbs. gas. per hr.	Lbs. gas. per B.H.P. per hr.	Suction W.O.	B.H.P.	Lbs. gas. per hr.	Lbs. gas. per B.H.P. per hr.
2	14.0	13.4	0.956	2	19.1	16.2	0.848
4	12.7	12.0	0.945	4	18.0	15.2	0.844
6	11.4	10.2	0.895	6	16.5	13.6	0.823
8	10.0	8.3	0.830	8	14.5	11.8	0.814
10	8.5	6.7	0.787	10	12.5	10.0	0.800
12	7.1	5.7	0.802	12	10.5	8.7	0.828
14	6.0	5.0	0.833	14	8.8	7.7	0.875
16	4.6	4.5	0.978	16	7.4	6.7	0.905
18	3.5	4.1	1.171	18	5.7	5.9	1.034
	1.0	3.8	3.800		2.7	5.1	1.890

Table 21



CHALLENGER 3400 R.P.M. MOTOR

Interpolation data for constant R.P.M. curves for
lines of equal lbs. gasoline/S.H.P./hr.

1000 R.P.M.				1200 R.P.M.			
Suction T.O.	B.H.P.	Lbs. gas. per hr.	Lbs. gas. per B.H.P. per hr.	Suction T.O.	B.H.P.	Lbs. gas. per hr.	Lbs. gas. per B.H.P. per hr.
2	24.0	19.0	0.791	2	28.2	21.7	0.720
4	23.1	18.4	0.736	4	27.7	21.4	0.744
6	21.1	17.0	0.806	6	25.6	20.2	0.789
8	18.8	15.1	0.803	8	22.9	18.4	0.804
10	16.6	13.3	0.801	10	20.6	16.5	0.801
12	13.8	11.6	0.840	12	17.0	14.6	0.853
14	11.5	10.3	0.895	14	13.9	12.8	0.920
16	9.8	8.8	0.899	16	12.0	11.0	0.917
18	7.5	7.5	1.012	18	9.1	9.3	1.022
20	4.4	6.3	1.432	20	5.8	7.5	1.292

Table 22

CHALMERS 3400 R.F.M. MOTOR
 Interpolation data for constant R.P.M. curves for
 lines of equal lbs. Gasoline/S.H.P./hr.

1400 R.P.M.					1600 R.P.M.		
Suction	B.H.P.	Lbs. Gas. per hr.	Lbs. Gas. per B.H.P. per hr.	Suction	B.H.P.	Lbs. Gas. per hr.	Lbs. Gas. per B.H.P. per hr.
W.O.	31.7	24.3	0.756	W.O.	34.8	26.9	0.773
2				2			
4	29.6	23.4	0.790	4	33.3	26.5	0.796
6	26.6	21.5	0.811	6	29.9	24.7	0.826
8	24.3	19.6	0.806	8	27.6	22.6	0.813
10	20.0	17.5	0.875	10	22.7	20.3	0.894
12	16.1	15.3	0.951	12	18.0	17.8	0.989
14	14.0	13.1	0.936	14	15.9	15.1	0.949
16	10.4	11.0	1.058	16	11.5	12.6	1.095
18	7.0	8.7	1.242	18	8.0	9.8	1.225

Table 23

CHALLENGER 3400 H.P.M. MOTOR
 Interpolation data for constant R.P.M. curves for
 lines of equal lbs. gasoline/H.P./hr.

1600 R.P.M.			2000 R.P.M.				
Suction	B.H.P.	Lbs. Gas. per hr.	Lbs. Gas. per B.H.P.	Suction	B.H.P.	Lbs. Gas. per hr.	Lbs. Gas. per B.H.P. per hr.
V.O.	37.2	29.4	0.791	V.O.	38.6	31.9	0.826
2				2			
4				4			
6	32.7	27.7	0.846	6	35.1	30.6	0.871
8	30.1	25.6	0.851	8	31.9	28.5	0.892
10	25.2	23.1	0.917	10	27.1	25.9	0.955
12	19.6	20.2	1.030	12	20.7	22.6	1.091
14	17.5	17.1	0.977	14	18.8	19.1	1.017
16	12.3	14.2	1.163	16	12.7	15.8	1.243
18	8.7	10.9	1.254	18	9.0	12.1	1.344

Table 24

CHALMERS 3400 R.P.M. MOTOR
 Interpolation data for constant R.P.M. curves for
 lines of equal lbs. gasoline/B.H.P./hr.

2200 R.P.M.						
Suction	B.H.P.	Lbs. Gas. per hr.	Lbs. Gas. per B.H.P.	Suction	B.H.P.	Lbs. Gas. per hr.
W.O.	38.6	34.4	0.891	W.O.		
2				2		
4				4		
6	36.8	53.6	0.913	6		
8	32.7	51.4	0.960	8		
10	27.5	28.6	1.040	10		
12	21.0	24.9	1.186	12		
14	18.5	21.1	1.140	14		
16	12.6	17.4	1.380	16		
18	8.9	13.2	1.487	18		

Table 25

CHALMERS 3400 R.P.M. MOTOR					
Interpolation data for lines of equal lbs. gasoline/B.H.P./hr.					
0.800			0.850		
Lbs. gas. per B.H.P. per hr.	B.H.P.	R.P.M.	Lbs. gas. per B.H.P. per hr.	B.H.P.	R.P.M.
0.800	7.4	600	0.850	6.0	600
0.800	8.9	600	0.850	9.1	800
0.800	12.2	800	0.850	10.5	600
0.800	13.2	800	0.850	12.4	1000
0.800	21.0	1000	0.850	16.6	1200
0.800	25.2	1200	0.850	13.1	800
0.800	28.5	1400	0.850	22.4	1400
0.800	33.0	1600	0.850	27.3	1600
0.800	36.5	1800	0.850	32.0	1800
			0.850	36.9	2000

Table 26

CHALMERS 3400 R.P.M. MOTOR					
Interpolation data for lines of equal lbs. gasoline/B.H.P./hr.					
0.900			1.000		
Lbs. gas. per B.H.P. per hr.	B.H.P.	R.P.M.	Lbs. gas. per B.H.P. per hr.	B.H.P.	R.P.M.
0.900	5.3	600	1.000	4.4	600
0.900	7.5	800	1.000	6.0	800
0.900	10.3	1000	1.000	7.6	1000
0.900	11.6	600	1.000	9.7	1200
0.900	14.1	1200	1.000	12.7	1400
0.900	18.4	1400	1.000	15.6	1600
0.900	22.8	1600	1.000	18.2	1800
0.900	25.8	1800	1.000	23.2	2000
0.900	31.0	2000	1.000	31.3	2200
0.900	37.7	2200			

Table 27

CHALMERS 3400 R.P.M. MOTOR					
Interpolation data for lines of equal lbs. gasoline/B.H.P./hr.					
1.100			1.200		
Lbs. gas. per B.H.P. per hr.	B.H.P.	R.P.M.	Lbs. gas. per B.H.P. per hr.	B.H.P.	R.P.M.
1.100	3.9	600	1.200	3.4	600
1.100	5.2	800	1.200	4.6	800
1.100	6.3	1000	1.200	5.5	1000
1.100	7.6	1200	1.200	6.6	1200
1.100	9.3	1400	1.200	7.6	1400
1.100	10.7	1600	1.200	8.4	1600
1.100	13.1	1800	1.200	9.8	1800
1.100	17.1	2000	1.200	12.3	2000
1.100	23.2	2200	1.200	17.1	2200

Table 28

CHALMERS 3400 R.P.M. MOTOR					
Interpolation data for lines of equal lbs. gasoline/B.H.P./hr.					
1.300			1.400		
Lbs. gas. per B.H.P. per hr.	B.H.P.	R.P.M.	Lbs. gas. per B.H.P. per hr.	B.H.P.	R.P.M.
1.300	3.0	600	1.400	2.6	600
1.300	4.1	800	1.400	3.7	800
1.300	5.0	1000	1.400	4.5	1000
1.300	5.9	1200	1.400	5.3	1200
1.300	6.6	1400	1.400	6.0	1400
1.300	7.2	1600	1.400	6.5	1600
1.300	8.1	1800	1.400	7.1	1800
1.300	9.2	2000	1.400	8.2	2000
1.300	13.2	2200	1.400	10.4	2200

Table 29

CHALMERS 3400 R.P.M. MOTOR

Interpolation data for lines of
equal lbs. gasoline/B.H.P./hr.

1.500					
lbs. gas. per B.H.P. per hr.	B.H.P.	R.P.M.	lbs. gas. per B.H.P. per hr.	B.H.P.	R.P.M.
1.500	2.4	600			
1.500	3.4	800			
1.500	4.2	1000			
1.500	4.9	1200			
1.500	5.4	1400			
1.500	5.9	1600			
1.500	6.4	1800			
1.500	7.2	2000			
1.500	8.7	2200			

Table 30

CHALMERS 3400 R.P.M. MOTOR					
Interpolation data for constant R.P.M. curves for lines of equal Mechanical Efficiency.					
600 R.P.M.			900 R.P.M.		
Suction	B.H.P.	Mech. Eff.	Suction	B.H.P.	Mech. Eff.
W.O.	13.9	93.0	W.O.	20.7	92.0
2	11.5	90.8	2	20.3	90.5
4	11.6	90.8	4	18.6	89.3
6	10.2	88.2	6	16.5	88.0
8	8.5	86.0	8	14.2	85.5
10	7.1	82.2	10	12.1	82.2
12	5.9	78.0	12	9.9	78.0
14	4.7	73.6	14	8.5	73.2
16	3.8	70.0	16	6.6	68.8
18	1.0	50.0	18	3.5	49.7

Table 31

CHALMERS 3400 R.P.M. MOTOR					
Interpolation data for constant R.P.M. curves for lines of equal Mechanical Efficiency.					
1200 R.P.M.			1500 R.P.M.		
Suction	B.H.P.	Mech. Eff.	Suction	B.H.P.	Mech. Eff.
W.O.	27.4	69.5	W.O.	38.3	86.2
2	27.4	86.8	2		
4	24.6	87.4	4	29.5	84.0
6	22.3	86.2	6	27.6	84.0
8	20.0	84.0	8	25.5	81.8
10	17.1	80.8	10	21.5	76.4
12	13.6	77.0	12	16.9	75.0
14	11.8	71.4	14	14.8	68.0
16	9.0	66.0	16	10.9	61.3
18	5.7	48.2	18	7.5	46.0

Table 32

CHALMERS 3400 R.P.M. MOTOR					
Interrelation data for constant R.P.M. curves for lines of equal Mechanical Efficiency.					
1600 R.P.M.			2100 R.P.M.		
Suction	B.H.P.	Mech. Eff.	Suction	B.H.P.	Mech. Eff.
m W.O.	37.4	82.0	W.O.	38.6	76.8
2			2		
4	32.2	80.2	4	32.2	77.5
6	32.4	79.5	6	35.5	74.8
8	29.8	78.0	8	32.4	74.0
10	25.2	75.8	10	28.1	70.6
12	19.7	71.0	12	21.2	65.8
14	17.3	63.2	14	16.7	57.5
16	12.3	55.5	16	12.8	46.8
18	8.7	42.2	18	9.2	38.0

Table 33

CHALMERS 3400 P.P.M. MOTOR					
Interpolation data for constant R.P.M. curves for lines of equal Mechanical Efficiency.					
2400 R.P.M.					
Suction	B.H.P.	Mech. Eff.	Suction	B.H.P.	Mech. Eff.
7.0.	36.5	71.5			
2					
4					
6	34.0	69.5			
8	32.6	69.0			
10	30.2	67.0			
12	19.5	58.0			
14	16.5	50.5			
16	12.2	41.0			
18	8.0	32.3			

Table 34

CHALMERS 3400 R.P.M. MOTOR					
Interpolation data for lines of equal Mechanical Efficiency.					
50 per cent			45 per cent		
Mech. Eff.	R.P.M.	B.H.P.	Mech. Eff.	R.P.M.	B.H.P.
50.0	600	1.0	45.0	1800	9.9
50.0	900	3.5	45.0	2100	12.0
50.0	1200	6.0	45.0	2400	14.1
50.0	1500	8.5	40 per cent		
50.0	1800	11.5	40.0	2100	10.0
50.0	2100	14.0	40.0	2400	11.9
50.0	2400	16.2			

Table 35

CHALMERS 3400 R.P.M. MOTOR					
Interpolation data for lines of equal Mechanical Efficiency.					
90 per cent			85 per cent		
Mech. Eff.	R.P.M.	B.H.P.	Mech. Eff.	R.P.M.	B.H.P.
90.0	600	11.2	85.0	600	8.1
90.0	900	19.2	85.0	900	13.6
			85.0	1200	20.6
			85.0	1500	31.5

Table 36

CHALMERS 3400 R.P.M. MOTOR					
Interpolation data for lines of equal Mechanical Efficiency.					
80 per cent			75 per cent		
Mech. Eff.	R.P.M.	B.H.P.	Mech. Eff.	R.P.M.	B.H.P.
80.0	600	6.2	75.0	600	5.0
80.0	900	10.6	75.0	900	8.5
80.0	1200	16.1	75.0	1200	13.1
80.0	1500	23.1	75.0	1500	17.9
80.0	1800	32.5	75.0	1800	24.8
			75.0	2100	34.8

Table 37

CEALMERS 3400 R.P.M. MOTOR					
Interpolation data for lines of equal Mechanical Efficiency.					
70 per cent			65 per cent		
Mech. Eff.	R.P.M.	B.H.P.	Mech. Eff.	R.P.M.	B.H.P.
70.0	600	4.0	65.0	600	3.1
70.0	900	6.6	65.0	900	5.2
70.0	1200	11.0	65.0	1200	9.2
70.0	1500	14.8	65.0	1500	12.9
70.0	1800	20.0	65.0	1800	17.4
70.0	2100	26.5	65.0	2100	21.5
70.0	2400	34.4	65.0	2400	26.6

Table 38

CHALMERS 3400 R.F.M. MOTOR					
Interpolation data for lines of equal Mechanical Efficiency.					
60 per cent			55 per cent		
Mech. Eff.	R.P.M.	B.H.P.	Mech. Eff.	R.P.M.	B.H.P.
60.0	600	2.5	55.0	600	1.6
60.0	900	4.5	55.0	900	4.0
60.0	1200	8.0	55.0	1200	7.0
60.0	1500	11.3	55.0	1500	9.8
60.0	1800	15.3	55.0	1800	13.4
60.0	2100	18.6	55.0	2100	16.2
60.0	2400	21.2	55.0	2400	18.2

Table 39

Test No. 1 & 2		Date: 10/10/16		Motor - CHALMERS		Temp. 58° F		Friction HP		Bar assembly		
Run No.	R.P.M.	Torque F.H.P.	Intake Suct. In. Hg.	Jacket Temp. In	Out	Run No.	R.P.M.	Torque F.H.P.	Intake Suct. In. Hg.	Jacket Temp. In	Out	
1	340	6.7	0.57	0.2		1	540	7.0	0.95	0.4		
2	350	7.0	0.58	2.0		2	535	7.3	0.97	2.0		
3	325	7.3	0.59	4.0		3	530	7.8	1.03	4.0		
4	320	8.3	0.66	6.0		4	525	8.5	1.12	6.0		
5	315	9.0	0.71	8.0		5	522	9.3	1.21	8.0		
6	312	9.8	0.76	10.0		6	520	10.3	1.33	10.0		
7	310	10.8	0.84	12.0		7	518	11.0	1.43	12.0		
8	305	11.5	0.88	14.0		8	515	11.8	1.51	14.0		
9	312	12.0	0.94	16.0		9	513	12.0	1.53	16.0		
10	295	12.5	0.92	16.0		10	510	12.8	1.63	18.0		
					Average				Average		126 deg. Fahr.	
					Average				Average		82 deg. Fahr.	

Table 40

Test No. 1 st 2		Date: 10/15/16		Motor - CHALMERS		Temp. 58°F		Friction HP		Barasensan, My	
Run No.	R.P.M.	Torque F.H.P.	Intake Suct. In. Hg.	Jacket Temp.		Intake Suct. In. Hg.	Jacket Temp.	In	Out	Average	82 deg. Fahr.
				In	Out						
1	795	8.5	1.69	0.8		1.3					
2	792	8.8	1.73	2.0		2.0					
3	790	9.0	1.78	4.0		4.0					
4	790	9.5	1.88	6.0		6.0					
5	790	10.0	1.98	8.0		8.0					
6	790	10.8	2.13	10.0		10.0					
7	795	11.5	2.28	12.0		12.0					
8	785	12.5	2.46	14.0		14.0					
9	790	13.5	2.66	16.0		16.0					
10	790	14.5	2.86	18.0		18.0					
										Average	126 deg. Fahr.
										Average	82 deg. Fahr.

Table 41

Test No. 1 1/2		Date: 10/15/16		Test No. 1 1/2		Date: 10/15/16	
Motor - CHALMERS		Temp. 53°F		Motor - CHALMERS		Temp. 53°F	
Friction HP		Bar. 23.5 in. Hg		Friction HP		Bar. 23.5 in. Hg	
Run No.	R.P.M.	Torque F.H.P.	Intake Suct. In. Hg.	Intake Jacket Temp. In	Intake Suct. In. Hg.	Jacket Temp. In	Out.
1	1210	11.3	3.40	1.8			
2	1200	11.5	3.45	2.0			
3	1200	11.8	3.53	4.0			
4	1200	12.5	3.75	5.0			
5	1200	13.0	3.90	8.0			
6	1200	13.5	4.05	10.0			
7	1210	14.5	4.38	12.0			
8	1200	15.5	4.65	14.0			
9	1200	16.3	4.88	16.0			
10	1200	16.8	5.03	18.0			
				Average 62 deg. Fahr.			
				Average 136 deg. Fahr.			

Table 42

CHALMERS 3400 R.P.M. MOTOR					
Interpolation data of friction torque above 1200 R.P.M. from extended curves, for determining friction horse power above 1200 R.P.M.					
W.O. throttle			4" vacuum		
R.P.M.	Frict. torque	Frict. H.P.	R.P.M.	Frict. torque	Frict. H.P.
1500	13.5	5.1	1500	14.5	5.4
1800	16.5	7.4	1800	17.5	7.8
2100	20.0	10.5	2100	21.2	11.1
2400	23.7	14.2	2400	25.0	15.0
2" vacuum			6" vacuum		
1500	14.0	5.3	1500	15.1	5.7
1800	17.4	7.8	1800	18.2	8.2
2100	21.0	11.0	2100	21.5	11.3
2400	25.0	15.0	2400	25.2	15.1

Table 43

CHALMERS 3400 R.P.M. MOTOR					
Interpolation data of friction torque above 1200 R.P.M. from extended curves, for determining friction horse power above 1200 R.P.M.					
8" vacuum			12" vacuum		
R.P.M.	Frict. torque	Frict. H.P.	R.P.M.	Frict. torque	Frict. H.P.
1500	15.2	5.7	1500	16.4	6.2
1800	18.1	6.1	1800	19.0	6.5
2100	21.5	11.3	2100	22.1	11.6
2400	25.1	15.1	2400	25.5	15.3
10" vacuum			14" vacuum		
1500	15.6	5.9	1500	18.1	6.8
1800	18.5	6.3	1800	21.1	9.5
2100	21.7	11.4	2100	24.5	12.8
2400	25.4	15.2	2400	27.6	16.7

Table 44

CHALMERS 3400 R.P.M. MOTOR.					
Interpolation data of friction torque above 1200 R.P.M. from extended curves, for determining friction horse power above 1200 R.P.M.					
16" vacuum			18" vacuum		
R.P.M.	Fric. torque	Fric. H.P.	R.P.M.	Fric. torque	Fric. H.P.
1500	18.6	7.00	1500	19.3	7.2
1800	21.7	9.6	1800	22.1	10.0
2100	25.2	13.2	2100	25.5	13.4
2400	29.1	17.5	2400	29.1	17.5

Table 45

Test No. 2. Motor: Chalmers Date of Test: 11 / 15 / 16
 Object: Economy Model: 35 A. 3400 S. P. V. Barometer = 29.68 in. Hg
 Constant: Suction Number: 15035 Room Temperature 58 °F
 No. Cyl. 6. Gasoline 57 °B. at 43 °F.

Run No.	R.P.M.	Brake H.P.	Frict. H.P.	S.H.P.	Mech. Eff. %	MEP Net. lb./sq. in.	Jacket Temp.		Intake Suct. in. Hg.	Gasoline		lb. per Hour	lb./HP	Therm. Ztg. Eff.	Sp. M.	
							In.	Out.		lb.	Time					
1	620	85.0	12.9	0.8	13.7	94.4	73.5	112	135	0.3	0.5	2:39.0	11.3	0.890	14.3	23
2	885	90.0	19.9	2.3	22.3	89.6	79.7	119	147	0.8	0.5	2:45.2	10.9	0.847	25.1	21
3	1114	89.8	25.0	3.0	26.0	89.3	79.5	122	156	1.3	1.0	2:59.7	20.0	0.802	15.8	20
4	1358	85.3	28.9	4.1	33.0	87.5	75.5	133	161	1.9	1.0	2:28.2	24.2	0.839	15.0	24
5	1685	80.3	33.8	5.4	40.2	84.5	71.1	137	164	2.9	1.0	2:14.5	26.7	0.790	16.0	30
6	1946	74.0	36.0	6.6	44.8	80.4	65.6	141	167	3.6	1.0	1:57.6	30.6	0.850	14.9	34
7	2143	70.6	37.8	11.0	48.6	77.5	62.6	127	151	4.6	1.5	2:47.3	32.2	0.853	14.8	36
8	2445	61.3	37.5	15.0	52.5	71.4	54.3	124	149	5.7	1.5	2:32.2	35.5	0.946	13.4	39

Table 46

Test No. 2.		Motor: Chalmers		Date of Test: 11 / 16 / 16												
Object: Economy		Model: 35 A. 3400 R. P. M.		Barometer = 29.46 in. Hg												
Constant: Suction		Number: 15039		Room Temperature 53°F.												
		No. Cyl. 6.		Gasoline 57° B. at 43°F.												
Run No.	RPM	Brake HP	Frict. HP	B. P. F. HP	Mech. Eff. %	MEP. $\frac{lb}{sq. in.}$	Jacket Temp. $\frac{In.}{Out.}$	Intake		Therm. Deg.						
								Suct. In. Hg	lb. Time							
								lb per Hour	per Hr.	Eff. Spent						
1	560	75.5	10.6	1.1	11.7	30.5	66.9	118	143	2.0	0.5	5:51.6	5.12	0.482	25.2	22
2	574	79.5	11.4	1.2	12.6	30.8	70.4	132	146	2.0	0.5	2:44.0	11.0	0.363	13.8	15
3	820	76.5	15.7	1.9	17.5	59.5	67.8	133	162	2.0	0.5	2:05.6	14.5	0.913	13.1	30
4	995	78.0	19.4	2.5	21.9	88.8	69.1	120	141	2.0	0.5	1:46.6	16.8	0.907	13.9	30
5	1235	85.0	25.6	3.6	29.2	87.6	73.5	125	153	2.0	0.5	1:26.6	20.7	0.810	15.6	29
6	1355	82.0	27.8	4.9	32.7	95.1	72.6	143	155	2.0	0.5	1:21.4	22.1	0.797	15.9	30

Table 47

Test No. 2. Motor: Chalmers Date of Test: 11 / 16 / 16
 Object: Economy Model: 35 A. 3400 P. M. Barometer = 29.45 in. Hg
 Constant: Suction Number: 15037 Room Temperature 58° F.
 No. Cyl. 6 Gasoline 57° B. at 45° F.

Run No.	RPM.	Torque		Brake Fict. HP	Fict. HP	B.H.P.	Mech. Eff.	M.E.P. Net lb/so.	Jacket Temp.		Intake Suct. In. Hg	Gasoline		lb. per Hour	Therm. Deg. Eff.	Spark
		lb.	HP						In.	Out.		lb.	Time			
1	537	67.0	9.0	1.1	10.1	89.2	59.4	116	129	4.0	0.5	4:44.0	6.4	0.705	17.9	18
2	684	76.0	13.0	1.5	14.5	89.6	67.4	118	130	4.0	0.5	3:12.0	9.4	0.722	17.5	18
3	867	76.0	16.5	2.1	18.6	88.8	67.4	126	158	4.0	0.5	2:29.4	12.1	0.730	17.3	20
4	990	76.5	18.9	2.6	21.5	88.0	67.8	138	171	4.0	0.5	1:50.4	16.3	0.861	14.7	25
5	1124	76.5	21.5	3.2	24.7	87.1	67.8	143	166	4.0	0.5	1:48.8	16.5	0.767	15.5	28
6	1300	75.5	24.9	4.1	29.0	86.0	67.8	145	160	4.0	0.5	1:34.4	19.1	0.765	15.5	32
7	1620	76.5	31.2	6.3	37.5	85.2	67.9	135	159	4.0	0.5	1:13.8	24.4	0.777	16.2	25
8	1820	74.0	33.7	8.0	41.7	80.7	66.6	137	157	4.0	0.5	1:06.2	27.2	0.807	15.6	27

Table 46

Test No. 2. Motor: Chalmers Date of Test: 11 / 16 / 16
 Object: Economy Model: 35 A. 3400 R. I. A. Barometer = 29.46 in. Hg
 Constant: Suction Number: 15039 Room Temperature 33°F
 No. Cyl. 6. Gasoline 57° B. at 45°F.

Run No.	R.P.M.	Torque		Frict.		Mech. Eff.	M.E.P. Net. lb/sq. in.	Jacket Temp.		Intake Suct. in. Hg.	Gasoline		lb. per Hour	lb./HP	Therm. Deg. Eff.	
		lb.	HP	HP	HP			In.	Out.		b.	Time				
1	719	53.7	11.4	1.7	13.1	86.8	56.5	105	118	6.0	0.5	5:37.2	8.3	0.726	17.4	15
2	872	54.0	14.1	2.3	16.4	86.0	57.2	107	126	6.0	0.5	2:49.8	10.6	0.753	16.6	20
3	1100	53.3	19.1	3.3	22.4	85.5	61.4	114	137	6.0	0.5	1:53.4	15.9	0.832	15.2	25
4	1305	58.0	22.2	4.3	26.5	83.7	60.2	120	149	6.0	1.0	3:16.2	18.3	0.821	15.4	25
5	1440	58.0	24.4	5.1	29.5	82.7	60.2	131	157	6.0	1.0	2:55.2	20.5	0.842	15.0	28
6	1325	68.5	33.0	9.4	42.4	77.8	60.7	120	143	6.0	1.0	2:13.8	26.9	0.816	15.5	30
7	2340	64.0	36.4	14.4	50.8	71.7	56.7	126	158	6.0	1.0	1:47.0	33.6	0.923	13.7	32

Table 49

Test No. 2. Motor: Chalmers Date of Test: 11 / 16 / 16
 Object: Economy Model: 35 A. 3400 R. F. M. Barometer = 29.46 in. Hg.
 Constant: Suction Number: 15039 Room Temperature 53 °F.
 No. Cyl. 6. Gasoline 57 °B. at 43 °F.

Run No.	Torque Brake		Frict. HP	Mech. Eff.		Jacket Temp.		Intake Suct. In. Hg.	Gasoline		Therm. Deg.			
	lb.	HP		Brake HP	Net HP	In.	Out.		lb. per Hour	Time				
1	59.0	7.4	1.3	8.7	44.3	120	132	8.0	1.0	4:26.2	13.5	1.885	6.9	15
2	92.0	56.0	12.2	2.6	14.9	103	123	8.0	0.5	3:07.8	9.6	0.787	16.2	28
3	116.7	56.8	16.6	3.8	20.4	114	126	8.0	0.5	2:27.9	12.2	0.735	17.2	30
4	136.7	59.5	20.3	4.8	25.1	124	134	8.0	1.0	2:20.9	17.9	0.862	14.3	30
5	197.4	58.5	28.9	9.9	38.8	131	148	8.0	1.0	2:21.0	25.5	0.894	14.3	32
6	237.0	52.5	31.1	14.7	45.8	131	158	8.0	1.0	1:51.8	32.2	1.035	12.2	33

Table 50

Test No. 2, Motor: Chalmers Date of Test: 11 / 15 / 16
 Object: Economy Model: 35 A, 34.0 R. P. U. Barometer = 29.43 in. Hg
 Constant: Suction Number: 15039 Room Temperature 53° F.
 No. Cyl. 6 Gasoline 57° B. at 43° F.

Run No.	RPM	Torque lb.	Brake HP	Frict. HP	5 HP Fr. HP	Mech. Eff.	MEP Net. lb./sq. in.	Jacket Temp.		Intake Suct. In. Hg	Gasoline		lb. per Hour	lb./HP Hour	Therm. Deg. F.	Sp. Ht. ft.
								In.	Out.		In. Hg	lb.				
1	515	43.0	5.5	1.0	6.5	84.7	38.1	104	140	10.0	1.0	10:35	5.7	1.020	12.4	19
2	710	47.0	8.3	2.0	10.3	81.1	41.6	131	152	10.0	1.0	8:21.4	7.2	0.862	14.8	19
3	916	47.0	10.8	2.8	13.6	79.8	41.6	126	145	10.0	1.0	6:37.3	9.1	0.840	15.0	22
4	1210	49.5	15.0	4.1	19.1	78.5	43.8	121	143	10.0	1.0	1:11.2	11.6	0.774	16.3	25
5	1650	50.5	20.4	6.9	27.3	74.9	44.7	126	154	10.0	1.0	5:03.0	19.1	0.935	15.5	30
6	1920	49.0	23.5	9.5	33.0	71.2	43.4	137	153	10.0	1.0	2:38.2	22.8	0.969	15.0	28
7	2400	47.3	28.4	15.2	43.6	65.1	41.8	116	144	10.0	1.0	2:03.0	29.3	1.052	12.2	35

Table 51

Test No. 2. Motor: Chalmers Date of Test: 11 / 20 / 16
 Object: Economy Model: 35 A. 3400 R. P. V. Barometer = 29.43 in. Hg
 Constant: Suction Number: 15039 Room Temperature 53 °F.
 No. Cyl. 5. Gasoline 57 °B. at 43°F.

Run No.	R.P.M.	Brake hp	Frict. hp	F. hp	Mach. Eff. %	Mach. Net Eff. %	Jacket Temp.		Intake Suct. In. Hg	Gasoline lb. per Hr.	Therm. Deg					
							In.	Out.								
1	778	36.0	7.0	2.3	9.3	74.9	31.9	126	148	12.0	0.5	4:42.6	6.4	0.909	15.9	19
2	842	38.0	8.0	2.6	10.6	75.3	33.6	130	156	12.0	0.3	2:30.4	7.2	0.897	14.1	23
3	845	37.0	7.8	2.5	10.4	74.8	32.8	126	154	12.0	0.5	4:11.8	7.2	0.915	15.8	22
4	1056	39.0	10.6	3.5	14.2	74.6	34.5	126	153	12.0	0.5	3:10.8	9.4	0.892	14.1	24
5	1345	39.0	13.1	5.1	18.2	72.1	34.5	128	155	12.0	0.5	2:40.4	11.2	0.856	14.7	26
6	1539	40.5	15.6	6.4	22.0	70.9	35.9	129	154	12.0	0.5	2:18.4	13.0	0.832	15.2	28
7	1825	42.5	19.4	8.7	28.1	69.0	37.6	129	155	12.0	1.0	2:08.6	19.1	0.985	12.3	33
8	1950	40.0	18.3	8.8	27.1	67.5	35.4	132	160	12.0	1.0	2:50.9	21.1	1.150	11.0	30
9	2566	38.0	22.7	15.2	37.9	59.9	35.6	130	155	12.0	1.0	2:21.9	25.4	1.119	11.3	37

Table 52

Test No. 2. Motor: Chalmers Date of Test: 11 / 20 / 16
 Object: Economy Model: 35 A. 3400 R. P. M. Barometer = 29.43 in. Hg
 Constant: Suction Number: 15039 Room Temperature 53°F
 No. Cyl. 6 Gasoline 57° B. at 48°F.

Rev No.	Torque Brake		Frict.		BHP FHP	Mech. Eff. Net 1/2%	Jacket Temp.		Intake Suct. In. Hg	Gasoline		Therm Deg				
	lb.	HP	HP	HP			In.	Out.		lb.	Time		per Hr.	Eff.		
1	625	26.0	4.1	1.6	5.7	71.0	23.0	120	145	14.0	0.5	5:23.4	5.6	1.368	9.2	22
2	862	31.5	6.8	2.9	9.7	70.3	27.9	130	155	14.0	0.3	2:35.5	7.0	1.024	12.3	25
3	1340	32.0	10.7	5.6	16.3	65.5	28.3	124	142	14.0	0.5	3:09.2	9.5	0.888	14.2	32
4	1973	33.0	16.3	11.4	27.7	58.9	29.2	121	144	14.0	0.5	2:11.8	13.7	0.841	15.0	36
5	2242	28.0	17.7	14.6	32.3	54.8	24.8	125	151	14.0	1.0	2:55.6	20.5	1.160	10.9	40

Table 53

Test No. 2. Motor: Chalmers Date of Test: 11 / 20 / 16
 Object: Economy Model: 35 A. 3400 S. P. M. Barometer = 29.43 in. Hg
 Constant: Suction Number: 15039 Room Temperature 55 °F.
 No. Cyl. 6 Gasoline 57 °B. at 45 °F.

Run No.	RPM.	Torque Brake Frict.		B. HP	F. HP	Mech. Eff.	MEP. Net. lb./sq. In.	Jacket Temp.		Intake Suct. In. Hg	Gasoline		lb. per Hour	lb./HP	Therm. Deg. per Hr.	Est. Spent
		HP	HP					In.	Out.		lb.	Time				
1	380	12.0	1.1	0.9	2.0	57.3	10.6	131	153	16.0	0.3	5:36.2	3.2	2.820	4.5	23
2	1020	20.0	5.1	3.8	8.9	57.2	17.7	118	150	16.0	0.5	4:43.2	6.4	1.248	10.1	26
3	1210	25.0	7.6	5.0	12.6	50.4	22.2	136	160	16.0	0.5	3:14.2	9.3	1.250	10.3	32
4	1730	24.5	10.6	9.1	19.7	53.9	21.7	125	145	16.0	0.5	2:31.9	12.9	1.120	11.3	36
5	2275	21.0	11.9	15.6	27.5	43.4	13.6	122	144	16.0	1.0	2:54.7	15.3	1.285	9.6	42

Table 54

CHALLENGER 3400 R.P.M. MOTOR

Interpolation data for constant R.T.M. curves for
lines of equal lbs. gasoline/R.H.P./hr.

600 R.P.M.					800 R.P.M.						
Suction W.O.	B.H.P.	Lbs. gas. per hr.	Lbs. gas. per B.H.P. per hr.	Suction W.O.	B.H.P.	Lbs. gas. per hr.	Lbs. gas. per B.H.P. per hr.	Suction W.O.	B.H.P.	Lbs. gas. per hr.	Lbs. gas. per B.H.P. per hr.
2	13.2	12.3	0.930	2	17.6	15.1	0.855	2	15.0	13.8	0.918
4	10.2	8.1	0.794	4	14.7	11.5	0.787	4	14.7	11.5	0.787
6	9.3	5.9	0.635	6	13.1	9.4	0.717	6	13.1	9.4	0.717
8	9.0	4.9	0.544	8	12.0	8.2	0.679	8	12.0	8.2	0.679
10	6.7	4.4	0.656	10	9.4	7.3	0.771	10	9.4	7.3	0.771
12				12	7.3	6.5	0.890	12	7.3	6.5	0.890
14				14	6.0	5.8	0.957	14	6.0	5.8	0.957
16				16	4.3	5.1	1.174	16	4.3	5.1	1.174

Table 55

CHARLES 3400 I.P.M. MOTOR

Interpolation data for constant R.P.M. curves for
lines of equal lbs. gasoline/S.H.P./hr.

1000 R.P.M.				1200 R.P.M.			
Suction W.O.	B.H.P.	Lbs. Gas. per hr.	Lbs. gas. per B.H.P. per hr.	Suction W.O.	B.H.P.	Lbs. gas. per hr.	Lbs. gas. per B.H.P. per hr.
2	21.8	17.9	0.821	2	25.8	20.6	0.800
4	20.2	17.1	0.845	4	24.8	20.3	0.816
6	18.8	15.0	0.798	6	22.9	18.4	0.804
8	17.0	12.8	0.753	8	20.8	15.2	0.775
10	15.0	11.3	0.754	10	17.0	14.5	0.855
12	12.0	10.2	0.843	12	14.6	13.1	0.894
14	9.6	9.1	0.942	14	11.9	11.5	0.971
16	7.9	7.9	0.992	16	9.8	9.9	1.009
	5.8	6.7	1.151		7.2	8.3	1.155

Table 56

CHALLENGER 340C I. E. M. MOTOR

Interpolation data for constant R.P.M. curves for lines of equal lbs. Gasoline/B.H.P./hr.

1400 R.P.M.				1600 R.P.M.			
Suction	B.H.P.	Lbs. Gas. per hr.	Lbs. Gas. per B.H.P. per hr.	Suction	B.H.P.	Lbs. Gas. per hr.	Lbs. Gas. per B.H.P. per hr.
W.O.	29.5	23.3	0.790	W.O.	32.6	25.9	0.794
2	28.5	23.3	0.817	2	30.3	24.9	0.820
4	26.7	21.7	0.813	4	27.8	22.7	0.821
6	24.5	19.5	0.796	6	23.5	20.7	0.878
8	20.8	17.6	0.848	8	19.6	18.6	0.950
10	17.2	15.9	0.925	10	16.5	16.5	1.000
12	14.2	14.1	0.991	12	13.4	13.9	1.038
14	11.6	12.0	1.050	14	9.8	11.5	1.177
15	8.6	9.9	1.152				

Table 57

CHAMBERS 5400 R.F.M. MOTOR

Interpolation data for constant R.P.M. curves for
lines of equal lbs. gasoline/B.H.P./hr.

1800 R.F.M.			2000 R.F.M.				
Suction W.G.	B.H.P.	Lbs. gas. per hr.	Lbs. gas. per B.H.P. per hr.	Suction W.G.	B.H.P.	Lbs. gas. per hr.	Lbs. gas. per B.H.P. per hr.
7.0.	35.1	26.5	0.611	7.0.	37.0	31.0	0.637
2	33.8	28.0	0.828	2	36.8	31.0	0.656
4	31.0	25.8	0.831	4	33.5	28.8	0.660
6	26.0	23.5	0.903	6	28.1	26.4	0.904
8	22.0	21.3	0.957	8	24.2	23.9	0.986
10	18.8	16.9	1.004	10	21.0	21.2	1.007
12	15.0	15.3	1.056	12	16.4	17.6	1.082
14	10.6	13.1	1.210	14	11.4	14.6	1.280

Table 58

CHILLER 3450 R.F.M. MOTOR

Interpolation data for constant R.F.M. curves for lines of equal lbs. gasoline/B.H.P./hr.

2200 R.P.M.				2400 R.F.M.			
Suction	B.H.P.	Lbs. gas. per hr.	Lbs. gas. per B.H.P. per hr.	Suction	B.H.P.	Lbs. gas. per hr.	Lbs. gas. per B.H.P. per hr.
W.O.	37.8	33.5	0.887	W.O.	37.6	36.0	0.955
2			.	2			
4				4			
6	35.6	31.7	0.891	6	36.5	34.6	0.947
8	30.0	29.2	0.973	8	31.3	31.9	1.0200
10	26.3	26.5	1.010	10	28.1	29.0	1.052
12	23.2	23.5	1.052	12	25.5	25.7	1.007
14	17.5	19.7	1.123	14	18.2	21.5	1.180
16	11.8	16.1	1.332	16	11.3	17.5	1.555

Table 59

CHALMERS 3400 R.P.M. MOTOR					
Interpolation data for lines of equal lbs. gasoline/B.H.P./hr.					
0.650			0.750		
Lbs. gas. per B.H.P. per hr.	B.H.P.	R.P.M.	Lbs. gas. per B.H.P. per hr.	B.H.P.	R.P.M.
0.650	6.8	600	0.750	8.6	600
0.650	9.2	600	0.750	9.8	600
	0.700		0.750	9.9	600
0.700	6.2	600	0.750	14.1	600
0.700	9.5	600	0.750	15.1	1000
0.700	11.0	800	0.750	17.3	1000
0.700	13.0	800			

Table 60

CHALMERS 3400 R.P.M. MOTOR					
Interpolation data for lines of equal lbs. gasoline/B.H.P./hr.					
0.800			0.850		
Lbs. gas. per B.H.P. per hr.	B.H.P.	R.P.M.	Lbs. gas. per B.H.P. per hr.	B.H.P.	R.P.M.
0.800	5.1	600	0.850	4.7	600
0.800	8.9	600	0.850	8.0	800
0.800	10.3	600	0.850	10.9	600
0.800	13.1	1000	0.850	12.0	1000
0.800	14.7	800	0.850	15.2	800
0.800	16.9	1000	0.850	17.6	800
0.800	20.0	1200	0.850	18.1	1200
0.800	22.8	1200	0.850	20.9	1400
0.800	23.1	1400	0.850	24.2	1600
0.800	25.1	1400	0.850	28.5	1800
0.800	25.7	1200	0.850	37.0	2000
0.800	29.1	1400			
0.800	32.4	1600			

Table 61

CHALMERS 3400 R.F.M. MOTOR					
Interpolation data for lines of equal lbs. gasoline/B.H.P./hr.					
0.900			1.000		
Lbs. gas. per B.H.P. per hr.	B.H.P.	R.P.M.	Lbs. gas. per B.H.P. per hr.	B.H.P.	R.P.M.
0.900	4.3	600	1.000	3.7	600
0.900	7.0	800	1.000	5.6	800
0.900	10.7	1000	1.000	8.2	1000
0.900	12.1	600	1.000	10.3	1200
0.900	14.6	1200	1.000	13.7	1400
0.900	18.4	1400	1.000	16.8	1500
0.900	21.5	1600	1.000	19.4	1800
0.900	26.1	1800	1.000	21.8	2000
0.900	34.5	2000	1.000	27.3	2200
0.900	35.0	2200	1.000	33.3	2400

Table 62

CHALMERS 3400 R.P.M. MOTOR					
Interpolation data for lines of equal lbs. gasoline/l.H.P./hr.					
1.100			1.150		
Lbs. gas. per B.H.P. per hr.	B.H.P.	R.P.M.	Lbs. gas. per B.H.P. per hr.	B.H.P.	R.P.M.
1.100	3.2	800	1.150	4.4	800
1.100	4.7	800	1.150	5.6	1000
1.100	6.5	1000	1.150	7.2	1200
1.100	8.0	1200	1.150	8.6	1400
1.100	10.0	1400	1.150	10.2	1600
1.100	11.2	1600	1.150	11.3	1800
1.100	13.1	1800	1.150	13.9	2000
1.100	15.5	2000	1.150	16.0	2200
1.100	18.6	2200	1.150	19.5	2400
1.100	22.0	2400			

Table 63



CHALLERS 3400 R.P.M. MOTOR					
Interpolation data for constant R.P.M. curves for lines of equal Mechanical Efficiency.					
600 R.P.M.			900 R.P.M.		
Suction	B.H.P.	Mech. Eff.	Suction	B.H.P.	Mech. Eff.
W.O.	13.2	95.0	W.O.	19.7	90.0
2	9.2	90.0	2	17.6	89.2
4	10.2	89.3	4	16.9	88.6
6			6	15.0	86.0
8	9.0	84.0	8	13.5	82.5
10	6.6	82.8	10	10.7	79.6
12			12	8.5	74.5
14	4.0	71.0	14	7.0	70.0
16	2.8	58.0	16	5.1	58.8

Table 64

CHALMERS 3400 R.P.M. MOTOR					
Interpolation data for constant R.P.M. curves for lines of equal Mechanical Efficiency.					
1200 R.P.M.			1500 R.P.M.		
Suction	B.H.P.	Mech. Eff.	Suction	B.H.P.	Mech. Eff.
W.O.	25.8	87.5	W.O.	31.1	86.0
2	24.8	87.2	2		
4	22.9	86.7	4	28.5	84.2
6	20.8	85.0	6	26.2	83.4
8	18.0	80.5	8	22.0	78.2
10	14.6	78.0	10	18.4	75.8
12	11.9	74.0	12	15.3	72.0
14	9.8	67.8	14	12.5	64.4
16	7.3	58.5	16	9.3	57.0

Table 65

CHALMERS 3400 R.P.M. MOTOR					
Interpolation data for constant R.P.M. curves for lines of equal Mechanical Efficiency.					
1800 R.P.M.			2100 R.P.M.		
Suction	B.H.P.	Mech. Eff.	Suction	B.H.P.	Mech. Eff.
W.O.	35.1	63.5	W.O.	37.6	76.8
2			2		
4	33.7	61.3	4		
6	31.0	61.9	6	34.6	79.0
8	26.0	76.0	8	29.2	76.0
10	22.0	73.1	10	25.3	69.4
12	18.8	67.8	12	22.0	60.0
14	15.0	61.2	14	17.0	56.8
16	10.7	54.0	16	11.6	47.5

Table 66

CHALMERS 3400 R.P.M. MOTOR					
Interpolation data for lines of equal Mechanical Efficiency.					
90 per cent			85 per cent		
Mech. Eff.	R.P.M.	B.H.P.	Mech. Eff.	R.P.M.	B.H.P.
90.0	600	9.8	65.0	600	7.7
90.0	900	19.7	85.0	900	13.7
			85.0	1200	20.3
			85.0	1500	28.6

Table 67

CHALMERS 3400 R.P.M. MOTOR					
Interpolation data for lines of equal Mechanical Efficiency.					
80 per cent			75 per cent		
Mech. Eff.	R.P.M.	B.H.P.	Mech. Eff.	R.P.M.	B.H.P.
80.0	600	7.0	75.0	600	4.9
80.0	900	10.8	75.0	900	8.6
80.0	1200	15.8	75.0	1200	12.9
80.0	1500	22.5	75.0	1500	16.5
80.0	1800	29.0	75.0	1800	24.4
			75.0	2100	28.7

Table 68

CHALMERS 3400 H.P.M. MOTOR					
Interpolation data for lines of equal Mechanical Efficiency.					
70 per cent			65 per cent		
Mech. Eff.	R.P.M.	B.H.P.	Mech. Eff.	R.P.M.	B.H.P.
70.0	600	4.0	65.0	600	3.4
70.0	900	7.3	65.0	900	6.2
70.0	1200	10.7	65.0	1200	9.0
70.0	1500	15.5	65.0	1500	12.8
70.0	1800	20.5	65.0	1800	17.1
70.0	2100	25.0	65.0	2100	22.0

Table 69

CHALMERS 3400 R.P.M. MOTOR					
Interpolation data for lines of equal Mechanical Efficiency.					
60 per cent			55 per cent		
Mech. Eff.	R.P.M.	B.H.P.	Mech. Eff.	R.P.M.	B.H.P.
60.0	600	3.0			
60.0	900	5.3			
60.0	1200	7.5			
60.0	1500	10.5			
60.0	1800	14.0	55.0	1800	11.2
60.0	2100	19.0	55.0	2100	16.0

Table 70

Test No. 3. Motor: Chalmers Date of Test: 11 / 9 / 16
 Object: Economy Model: 35 A. 3400 R. P. M. Barometer = 29.87 in. Hg
 Constant: Torque Number: 15333 Room Temperature 58 °F.
 No. Cyl. 6. Gasoline °B. at °F.

Run No.	RPM.	Torque lb.	Brake HP	Frict. HP	B. F. HP	Mech. Eff. %	M.E.P. lb./sq. in.	Jacket Temp.		Intake Suct. In. Hg	Gasoline		lb. per Hour	Therm. Deg.
								In.	Out.		lb.	Time		
1	596	83.5	12.4	0.8	13.2	93.9	74.0	120	141	.7	.3	2:23.0	7.6	0.609 20.7 19
2	870	82.5	16.1	1.6	17.7	90.9	73.1	121	138	1.0	.5	2:56.4	10.2	0.636 19.8 19
3	1024	85.5	21.9	2.6	24.5	89.6	75.8	122	141	1.3	.5	1:47.7	16.7	0.762 16.6 22
4	1291	84.5	27.3	3.8	31.1	87.8	74.9	123	144	1.8	1.0	3:42.5	16.2	0.592 21.3 26
5	1831	80.0	36.6	7.7	44.3	82.7	70.9	132	157	3.1	.5	1:11.6	25.1	0.685 18.4 34
6	1990	77.5	38.6	9.3	47.9	80.6	68.7	123	143	3.7	1.0	2:21.6	25.4	0.658 19.2 32
7	2325	67.5	43.2	13.3	52.5	74.7	59.8	125	146	4.6	1.0	2:26.4	24.6	0.628 20.1 35

Table 71

Test No. 3. Motor: Chalmers Date of Test: 11 / 9 / 16
 Object: Economy Model: 35 A. 3400 R. P. M. Barometer = 29.37 in. Hg
 Constant: Torque Number: 15039 Room Temperature 58 °F.
 No. Cyl. 6 Gasoline °B. at °F.

Run No.	RPM.	Torque lb.	Brake HP	Frict. HP	B. HP	F. HP	Mech. Eff.	MEP Net. lb./sq. in.	Jacket Temp. °F.		Intake Suct. In. Hg	Gasoline		lb. per Hour	lb./HP	Therm. Deg. per Hr.	Eff. Spark
									In.	Out.		lb.	Time				
1	465	80.0	9.3				70.8	129	152	2.2	.3	2:14.3	8.0	0.865	14.6	9	
2	530	80.0	10.6				70.8	127	147	1.0	.3	2:12.6	8.2	0.769	15.4	14	
3	857	80.0	17.1				70.8	127	155	1.1	.3	1:17.2	14.0	0.817	15.5	18	
4	1091	80.0	21.6				70.8	129	154	1.8	.5	2:15.2	13.3	0.615	20.5	21	
5	1301	80.0	26.0				70.8	131	155	2.1	.5	1:52.4	15.0	0.615	20.5	27	
6	1492	80.0	29.8				70.8	131	155	2.2	.5	1:32.6	18.3	0.612	20.6	34	
7	1891	80.0	36.6				70.8	132	157	3.1	.5	1:11.6	25.1	0.685	18.4	34	

Table 72

Test No. 3. Motor: Chalmers Date of Test: 11 / 9 / 16
 Object: Economy Model: 35 A. P. H. Barometer = 29.37 in. Hg
 Constant: Torque Number: 16039 Room Temperature 56 °F.
 No. Cyl. 6 Gasoline °B. at °F.

Run No.	R.P.M.	Torque lb.	Brake HP	Frict. HP	B.H.P. F.H.P.	Mech. Eff. %	M.E.P. Net. lb./sq. in.	Jacket Temp. In. Out.	Intake Suct. In. Hg.	Gasoline		lb. per Hr.	lb./HP	Therm. Deg. Eff. Span.		
										lb. per Hr.	Time					
1	651	70.0	11.4				62.0	112	137	4.5	.3	1:51.7	11.8	1.033	12.2	18
2	659	70.0	11.6				62.0	121	152	4.5	.3	2:13.3	8.1	0.702	18.0	18
3	821	70.0	14.4				62.0	114	127	5.0	.3	1:54.3	9.4	0.657	19.2	23
4	1082	70.0	19.0				62.0	122	143	5.1	.3	1:20.3	13.5	0.709	20.5	30
5	1416	70.0	24.8				62.0	124	141	4.5	.3	1:15.4	14.4	0.578	21.9	33
6	1757	70.0	30.8				62.0	127	147	4.4	.5	1:30.1	20.0	0.649	19.5	35
7	1972	70.0	34.6				62.0	131	150	4.7	.5	1:14.7	24.1	0.636	18.1	39
8	2345	70.0	40.8				62.0	132	167	4.7	.5	0:59.6	30.2	0.740	17.1	45

Table 73

Test No. 3		Motor: Chalmers		Date of Test: 11 / 9 / 16												
Object: Economy		Model: 35 s. 3400 R. P. M. Barometer = 29.37 in. Hg		Room Temperature 58°F.												
Constant: Torque		No. Cyl. 6		Gasoline °B. at °F.												
Run No.	RPM	Torque lb.	Brake HP	Fric. HP	B. HP	Mach. Eff. %	MEP Net lb./sq. in.	Jacket Temp.		Intake Suct. in. Hg	Gasoline lb. per Hour	lb. /HP Hour	Therm. Deg. per Hr. Err. Span			
								In.	Out.							
1	639	60.0	9.5			53.2	53.2	127	154	7.2	.3	2:24.4	7.5	0.763	15.1	18
2	865	60.0	13.0			53.2	53.2	125	151	7.2	.5	3:04.8	9.7	0.752	16.8	20
3	1065	60.0	16.0			53.2	53.2	126	145	7.2	.5	3:35.6	11.6	0.725	17.5	23
4	1319	60.0	19.8			53.2	53.2	126	144	6.7	.3	0:54.4	19.9	1.003	12.6	27
5	1750	60.0	26.3			53.2	53.2	129	150	7.8	.5	2:15.6	13.3	0.506	24.9	32
6	2091	60.0	31.4			53.2	53.2	117	150	7.6	.5	0:52.2	20.7	0.660	19.1	36
7	2168	60.0	32.5			53.2	53.2	127	156	7.0	.5	1:13.8	24.4	0.751	16.8	41

Table 74

Test No. 3. Motor: Chalmers Date of Test: 11 / 7 / 16
 Object: Economy Model: 35 A. 3400 R. P. M. Barometer = 29.86 in. Hg
 Constant: Torque Number: 15039 Room Temperature 58 °F.
 No. Cyl. 6 Gasoline °B. at °F.

Run No.	RPM	Torque lb.	Brake Frict.		BHP	FHP	Mech. Eff. %	MEP Net. lb./sq. in.	Jacket Temp.		Intake Suct. in. Hg	Gasoline		Therm Deg. F.		
			HP	HP					In.	Out.		lb. per Hour	Time			
1	618	50.0	7.7				44.3	125	149	8.8	.3	3:37.6	5.0	0.643	19.6	18
2	995	50.0	12.5				44.3	124	139	9.9	.3	2:29.0	7.3	0.582	21.6	22
3	1486	50.0	18.6				44.3	120	145	10.3	.3	1:28.8	12.2	0.656	19.2	30
4	1914	50.0	23.9				44.3	122	147	10.4	.3	1:10.1	15.4	0.644	19.6	35
5	2134	50.0	27.4				44.3	125	152	10.6	.3	0:47.6	22.7	0.600	15.8	40

Table 75

Test No. 3 Motor: Chalmers Date of Test: 11 / 7 / 16
 Object: Economy Model: 35 A. 3400 R. P. M. Barometer = 29.36 in. Hg
 Constant: Torque Number: 35 A. 3400 R. P. M. Room Temperature 58 °F.
 No. Cyl. 6 Gasoline °B. at °F.

Run No.	RPM	Torque lb.	Brake HP	Frict. HP	B.T. F. HP	Mech. Eff.	MEP Net. lb/sq. in.	Jacket Temp.		Intake Suct. In. Hg	Gasoline lb/HP Hour	Therm. Deg. Eff.				
								In.	Out.							
1	574	30.0	4.3				26.6	132	165	14.0	.3	3:49.2	4.7	1.090	11.6	22
2	1021	30.0	7.7				26.6	136	164	15.0	.3	2:56.0	6.1	0.801	15.7	28
3	1447	30.0	10.8				26.6	132	153	15.1	.3	2:09.8	8.3	0.767	16.4	38
4	1693	30.0	12.7				26.6	125	147	15.0	.3	1:28.4	12.2	0.962	13.1	40
5	2004	30.0	15.0				26.6	125	148	15.0	.3	1:25.5	12.9	0.960	14.7	45
6	2340	30.0	17.6				26.6	125	148	14.2	.3	1:06.4	16.5	0.939	13.4	51

Table 77

Test No. 3		Motor: CHalmers		Date of Test: 11 / 7 / 16												
Object: Economy		Model: 35 L. 3400 I. P. L.		Barometer = 29.86 in. Hg												
Constant: Torque		Number: 15039		Room Temperature 83°F												
		No. Cyl. 6		Gasoline °B. at °F.												
Run No.	RPM	Torque lb.	Brake HP	Frict. HP	B. HP F. HP	Mech. Eff. %	MEP per in. Hg	Jacket Temp. In. Out.	Intake Suct. In. Hg	Gasoline		lb. per Hour	Therm. Deg. Eff. %			
										lb. Time	lb. per Hour					
1	545	20.0	2.7				17.7	125	148	16.1	.3	4:22.5	4.1	1,510	8.4	21
2	955	20.0	4.8				17.7	126	152	17.1	.3	2:56.8	6.1	1,280	9.9	26
3	1138	20.0	5.7				17.7	127	151	17.0	.3	3:03.8	5.9	1,034	12.2	33
4	1492	20.0	7.5				17.7	126	152	17.2	.3	2:25.4	7.4	0,995	12.7	38
5	2089	20.0	10.4				17.7	126	152	17.0	.3	1:33.0	11.6	1,110	11.4	41
6	2350	20.0	11.8				17.7	126	152	16.3	.3	1:26.4	12.5	1,063	11.9	48

Table 78

Test No. 30		Motor: Chalant		Date of Test: 11/7/16										
Object: Secondary		Model: 30 A. 3400 N. 2. 2.		Barometer = 29.86 In. Hg										
Constant: Torque		Number: 14839		Room Temperature 68°F.										
No. Cyl. 6		Gasoline		°B. at °F.										
Run No.	RRM lb.	Torque HP	Brake Frict. HP	5 HP F ² HP	Mech. Eff. %	MEP Net 10% ²	Jacket Temp. In. Out.	In. In. Hg	Stroke	Gasoline lb. per Hour	Therm. Deg. Eff. Spark			
1	803	10.0	2.0			8.9	102 143	17.4	.3	3:54.9	4.6	2.290	5.5	20
2	1123	10.0	2.8			8.9	126 158	13.0	.3	3:03.0	5.9	2.100	6.0	26
3	1357	10.0	3.4			8.9	117 133	17.3	.3	2:58.1	6.8	2.020	6.2	38
4	1629	10.0	4.1			8.9	137 158	18.4	.3	2:09.6	8.3	2.040	6.2	35
5	2069	10.0	5.2			8.9	130 156	18.3	.3	1:52.5	9.6	1.970	6.4	42

Table 79

CHEALERS 3400 R.P.M. MOTOR

Interpolation data for constant R.P.M. curves for
lines of equal lbs. gasoline/S.E.P./hr.

600 R.P.M.		900 R.P.M.					
Torque	B.H.P.	Lbs. gas. per hr.	Lbs. gas. per B.H.P. per hr.	Torque	B.H.P.	Lbs. gas. per hr.	Lbs. gas. per B.H.P. per hr.
10	1.5	3.9	2.600	-10	2.3	5.0	2.220
20	3.0	4.1	1.360	20	4.5	5.5	1.230
30	4.5	4.4	0.987	30	6.8	6.1	0.904
50	7.5	4.8	0.644	50	11.3	6.7	0.538
60	9.0	7.2	0.798	60	13.5	10.1	0.749
70	10.5	7.4	0.705	70	15.6	10.5	0.665
W.O.	12.5	7.7	0.613	W.O.	19.1	11.4	0.595

Table 80

CHALMERS 3400 R.P.M. MOTOR						
Interpolation data for constant R.P.M. curves for lines of equal lbs. Gasoline/B.H.P./hr.						
1200 R.P.M.			1500 R.P.M.			
Torque	B.H.P.	Lbs. Gas. per hr.	Lbs. Gas. per B.H.P.	Torque	B.H.P.	Lbs. Gas. per hr.
10	3.0	6.2	2.060	10	3.6	7.4
20	6.0	7.0	1.162	20	7.5	8.5
30	9.0	7.7	0.857	30	11.3	9.6
50	15.0	9.1	0.608	50	18.6	11.3
60	18.0	12.9	0.715	60	22.5	15.3
70	21.0	13.6	0.647	70	26.3	16.9
W.O.	25.5	15.2	0.599	W.O.	31.5	19.0
						1.975
						1.130
						0.849
						0.636
						0.707
						0.644
						0.604

Table 81

CHALLENGE 3400 R.P.M. MOTOR

Interpolation data for constant R.P.M. curves for
 lines of equal lbs. gasoline/S.H.P./hr.

1800 R.P.M.				2100 R.P.M.			
Torque	B.H.P.	Lbs. gas. per hr.	Lbs. gas. per B.H.P.	Torque	B.H.P.	Lbs. gas. per hr.	Lbs. gas. per B.H.P.
10	4.5	8.6	1.912	10	5.3	9.8	1.870
20	9.0	10.0	1.111	20	10.5	11.7	1.110
30	13.5	11.5	0.855	30	15.8	13.8	0.878
50	22.5	15.2	0.676	50	26.3	18.3	0.721
60	27.0	19.0	0.705	60	32.0	22.4	0.701
70	31.5	20.6	0.654	70	37.3	24.7	0.563
W.O.	36.3	22.9	0.530	W.O.	39.2		

Table 62

CHALISES 3400 R.P.M. MOTOR

Interpolation data for constant R.P.M. curves for
lines of equal lbs. gasoline/B.H.P./hr.

2400 R.P.M.

	Torque	B.H.P.	Lbs. gas. per hr.	Lbs. Gas. per B.H.P. per hr.	Torque	B.H.P.	Lbs. gas. per hr.	Lbs. Gas. per B.H.P. per hr.
10	6.0	11.1	1.850					
20	12.0	13.5	1.121					
30	18.0	16.3	0.506					
50	30.0	22.8	0.758					
60	36.0	26.2	0.728					
70	42.0	29.9	0.711					
W.O.	38.5							

Table 83

CHAMBERS 3400 R.P.M. MOTOR					
Interpolation data for lines of equal lbs. gasoline/B.H.P./hr.					
0.650			0.700		
Lbs. gas. per B.H.P. per hr.	B.H.P.	R.P.M.	Lbs. gas. per B.H.P. per hr.	B.H.P.	R.P.M.
0.650	8.1	600	0.700	7.1	600
0.650	9.6	800	0.700	8.1	600
0.650	10.1	600	0.700	9.1	1000
0.650	11.2	1000	0.700	10.2	1200
0.650	11.4	800	0.700	11.6	1400
0.650	11.6	1000	0.700	11.8	600
0.650	13.4	1400	0.700	13.3	1600
0.650	14.6	1600	0.700	14.4	800
0.650	22.3	1200	0.700	17.3	1800
0.650	27.0	1800	0.700	18.8	1000
0.650	29.5	1400	0.700	27.0	2000
0.650	33.5	1800	0.700	31.1	1400
0.650			0.700	37.1	2200
0.650			0.700	39.5	2000

Table 84



CHALMERS 3400 R.P.M. MOTOR					
Interpolation data for lines of equal lbs. gasoline/B.H.P./hr.					
0.800			0.900		
Lbs. gas. per B.H.P. per hr.	B.H.P.	R.P.M.	Lbs. gas. per B.H.P. per hr.	B.H.P.	R.P.M.
0.800	6.0	600	0.900	5.3	600
0.800	6.8	800	0.900	6.1	800
0.800	7.6	1000	0.900	7.1	1000
0.800	8.8	1200	0.900	8.0	1200
0.800	10.2	1400	0.900	9.4	1400
0.800	11.7	1600	0.900	10.6	1600
0.800	13.8	1800	0.900	12.3	1800
0.800	16.6	2000	0.900	14.1	2000
0.800	27.2	2200	0.900	17.0	2200

Table 85

CHALMERS 3400 R.P.M. MOTOR					
Interpolation data for lines of equal lbs. gasoline/B.H.P./hr.					
1.000			1.200		
Lbs. gas. per B.H.P. per hr.	B.H.P.	R.P.M.	Lbs. gas. per B.H.P. per hr.	B.H.P.	R.P.M.
1.000	4.8	600	1.200	4.2	600
1.000	5.7	800	1.200	5.0	800
1.000	6.5	1000	1.200	5.8	1000
1.000	7.5	1200	1.200	6.6	1200
1.000	8.7	1400	1.200	7.6	1400
1.000	9.8	1600	1.200	8.5	1600
1.000	11.3	1800	1.200	9.8	1800
1.000	12.7	2000	1.200	10.8	2000
1.000	14.5	2200	1.200	12.1	2200

Table 86

CHALMERS 3400 R.P.M. MOTOR					
Interpolation data for lines of equal lbs. gasoline/B.H.P./hr.					
1.400			1.600		
Lbs. gas. per B.H.P. per hr.	B.H.P.	R.P.M.	Lbs. gas. per B.H.P. per hr.	B.H.P.	R.P.M.
1.400	3.7	600	1.600	3.4	600
1.400	4.5	800	1.600	4.0	800
1.400	5.1	1000	1.600	4.8	1000
1.400	5.8	1200	1.600	5.0	1200
1.400	6.5	1400	1.600	5.5	1400
1.400	7.4	1600	1.600	6.3	1600
1.400	8.4	1800	1.600	7.1	1800
1.400	9.3	2000	1.600	7.8	2000
1.400	10.4	2200	1.600	8.8	2200

Table 87

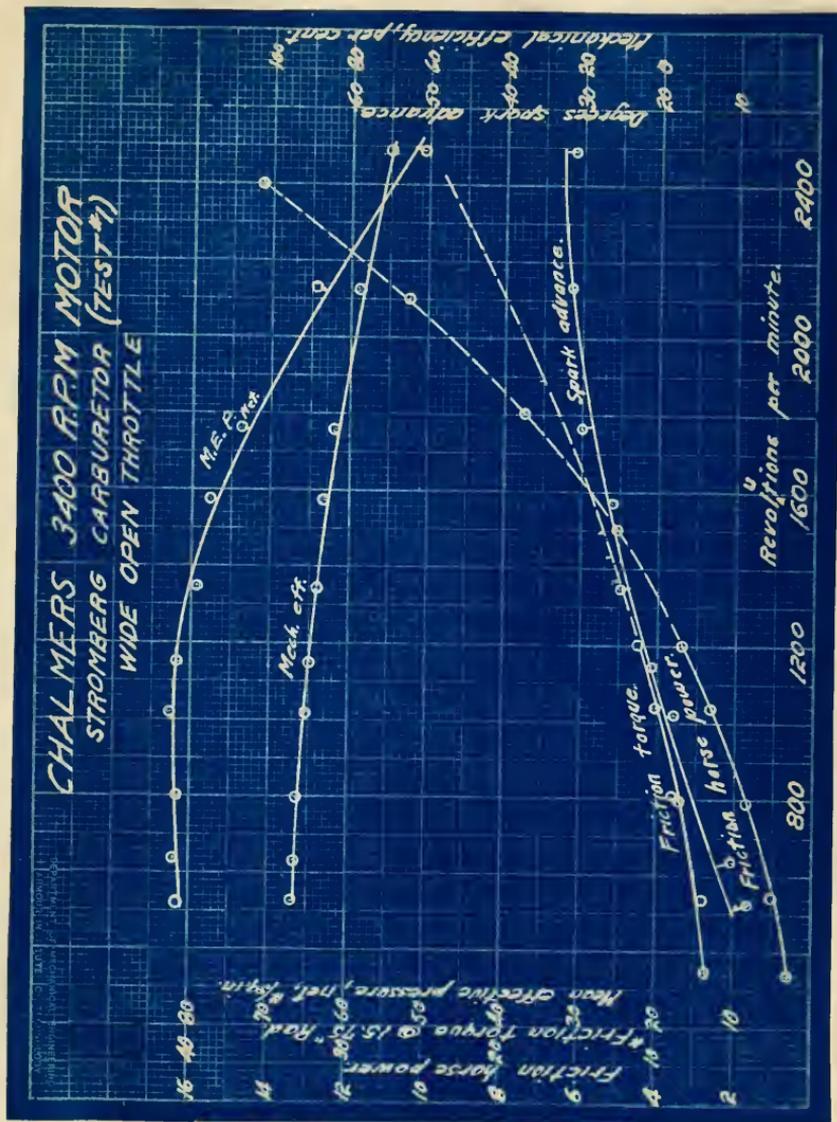


Fig. 2

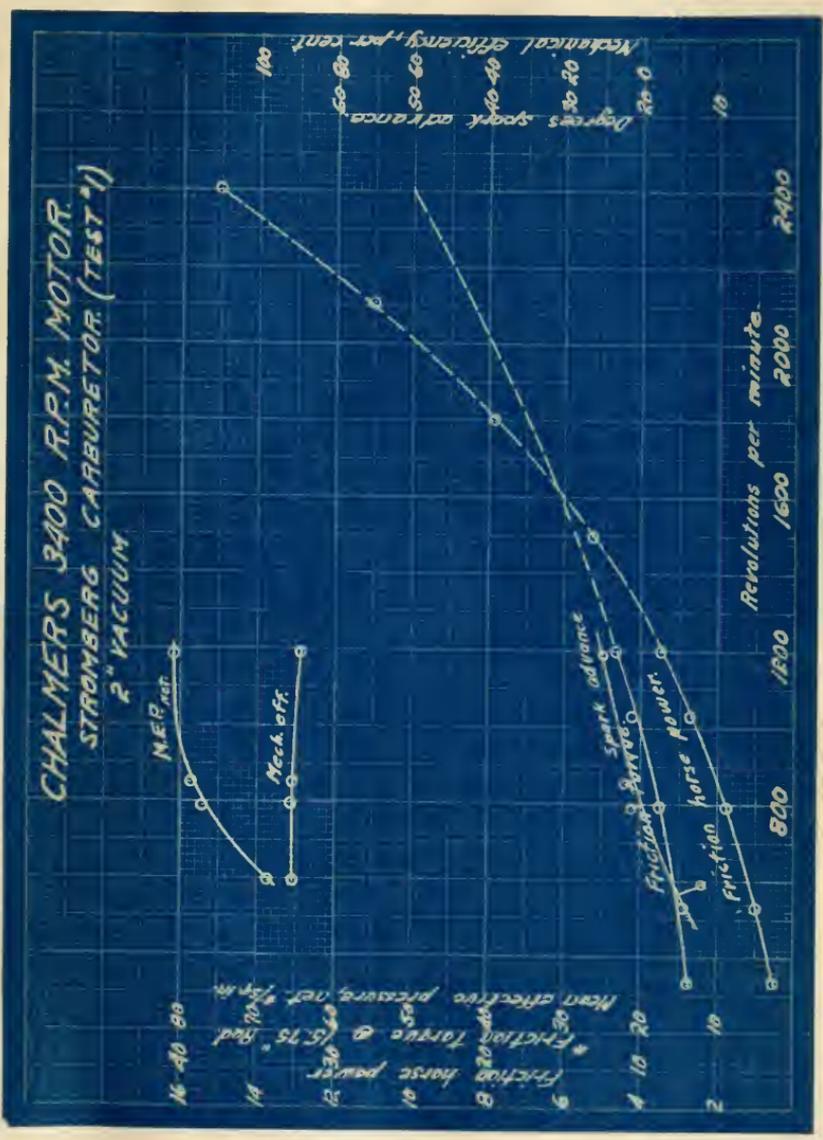


Fig. 4

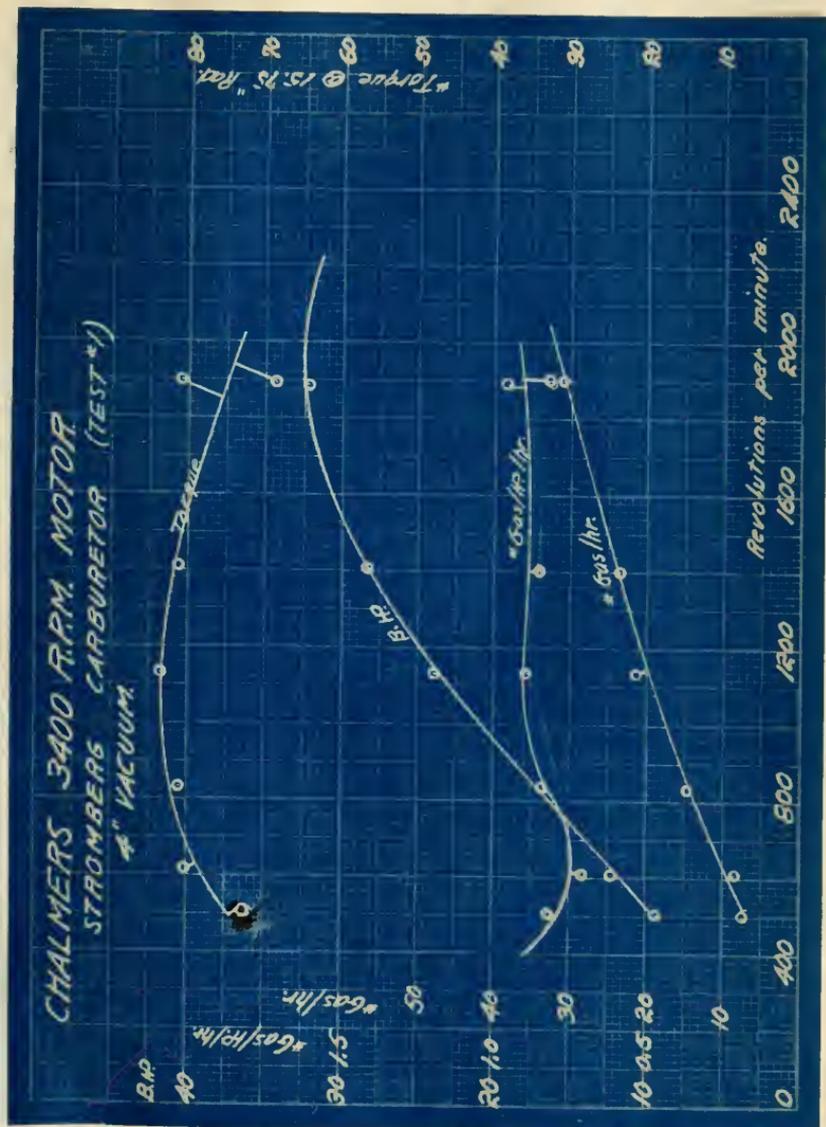


Fig. 5

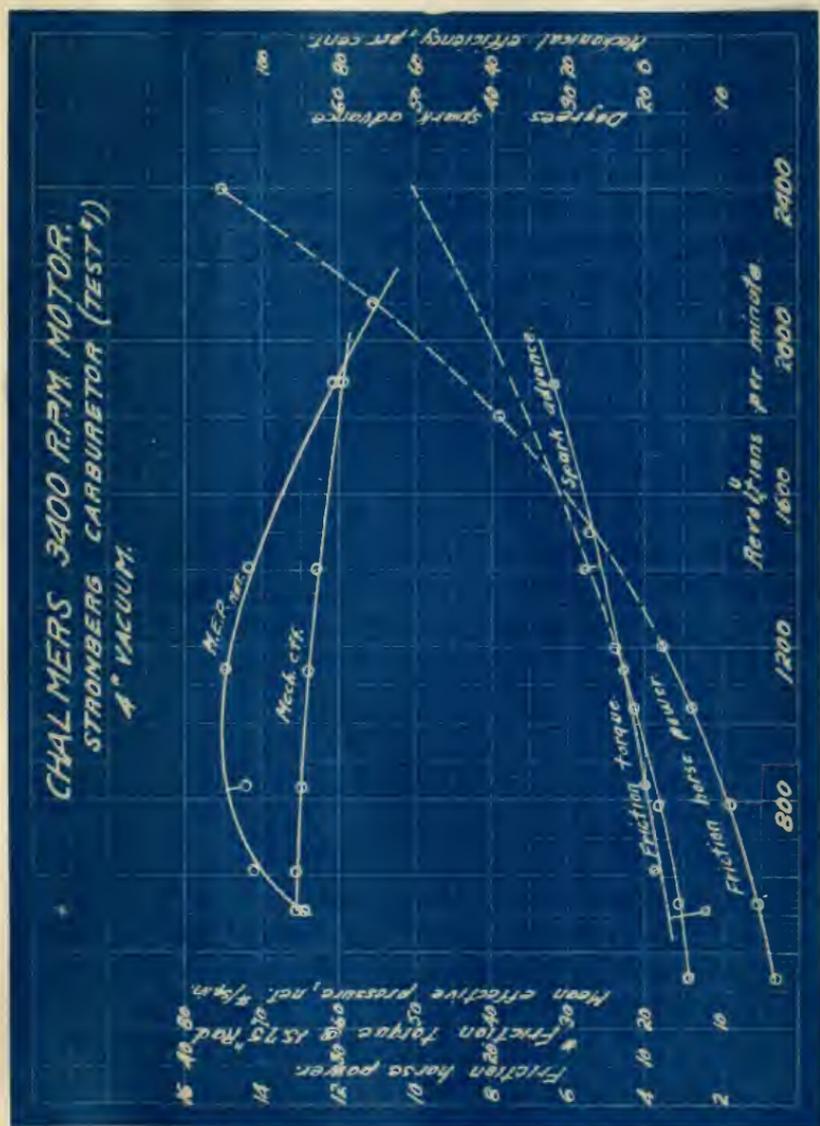


Fig. 6

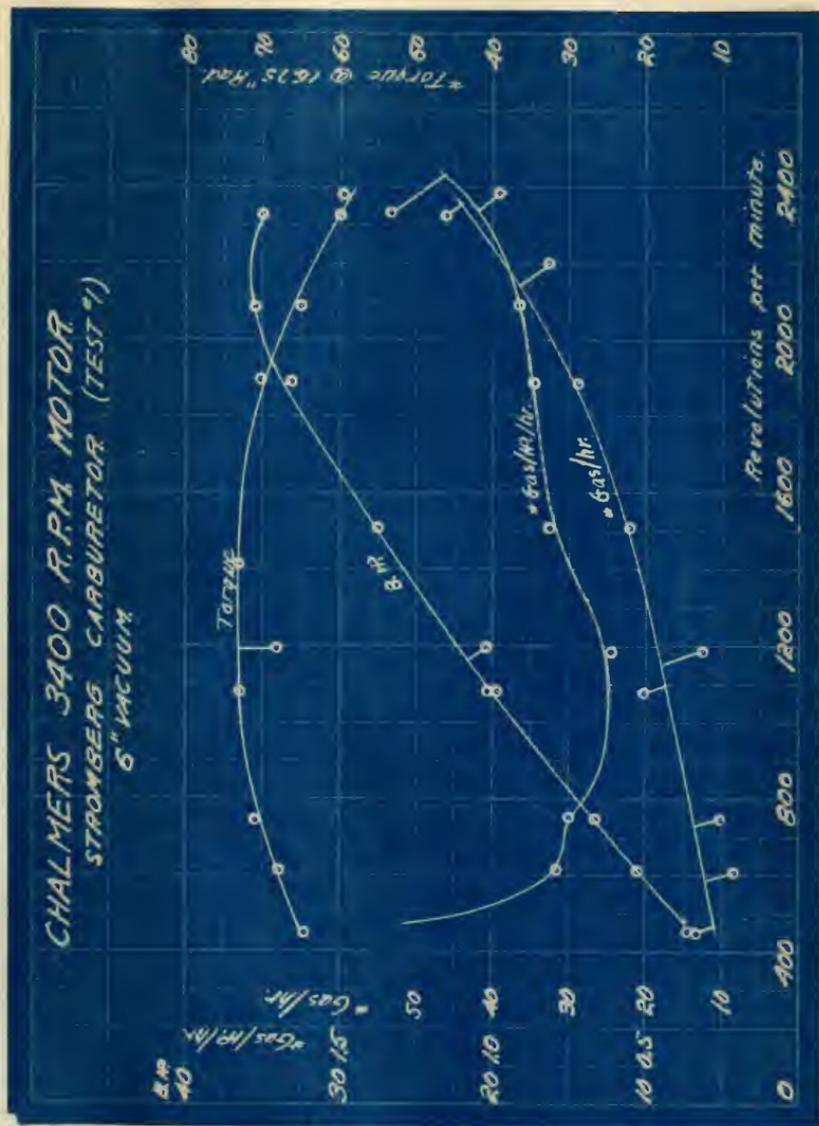


Fig. 7

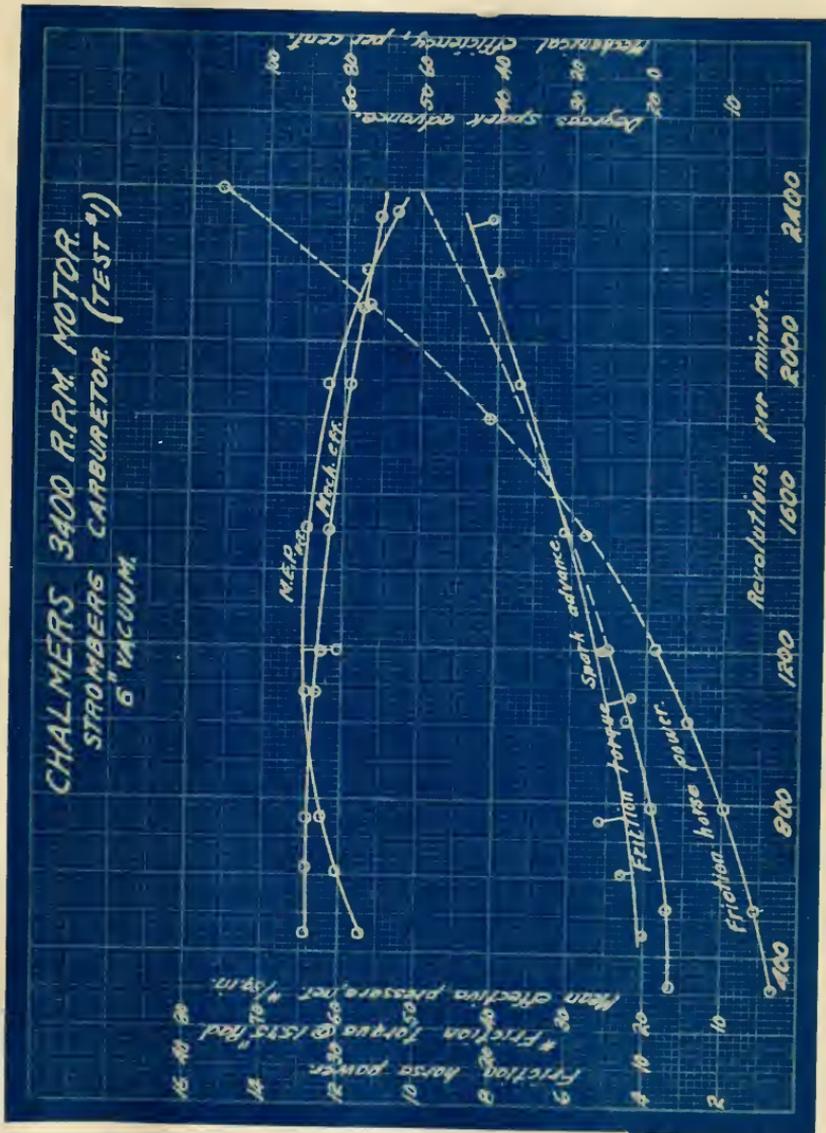


Fig. 8

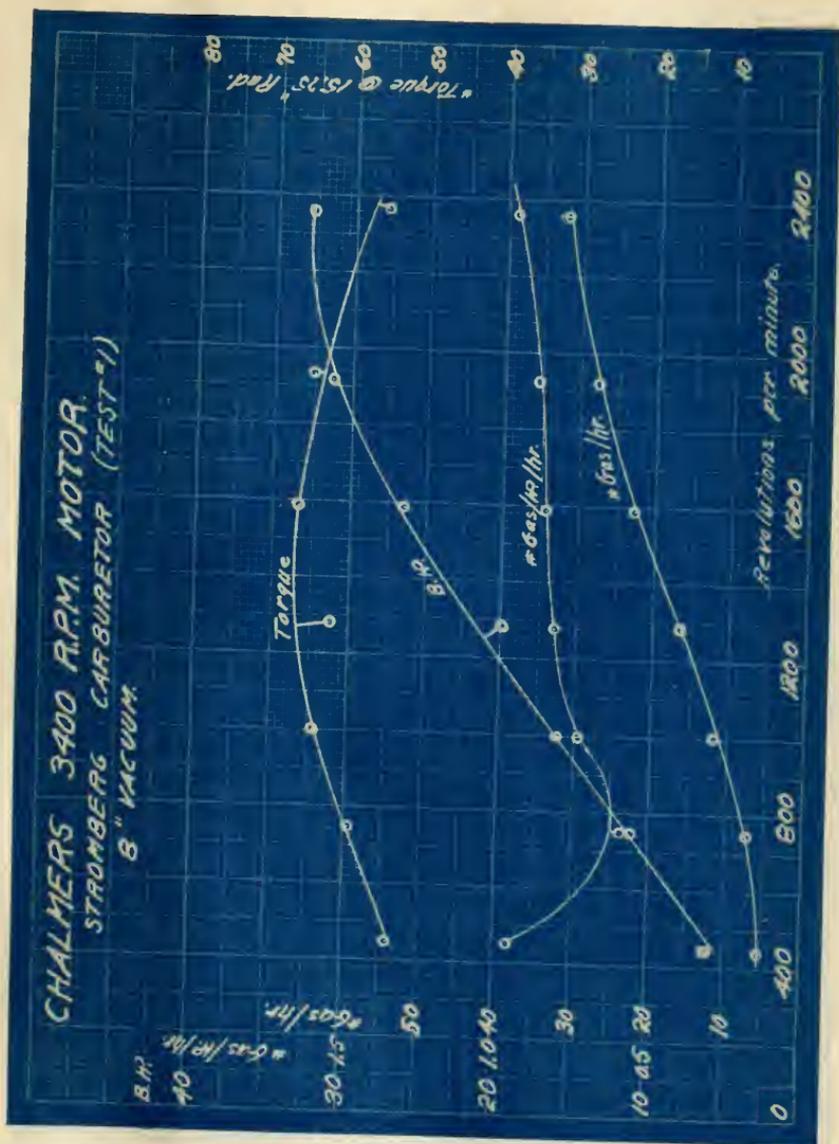


Fig. 9

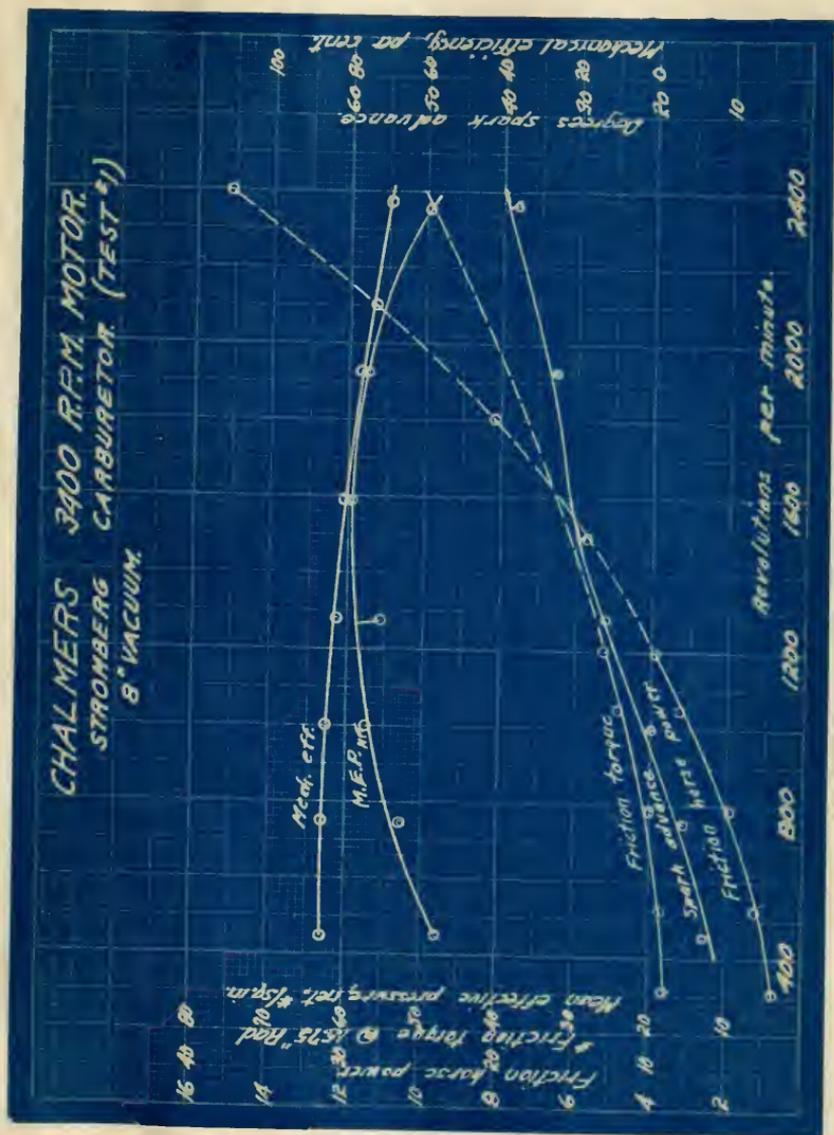


Fig. 10



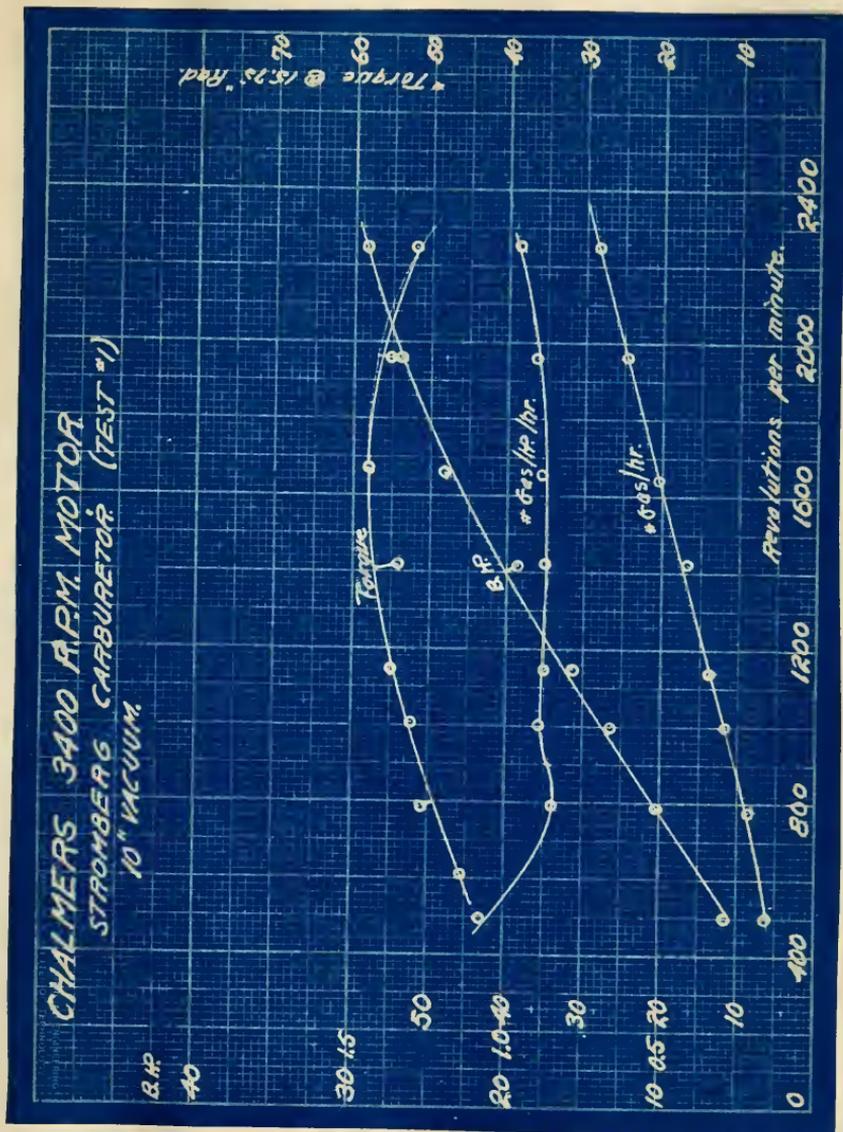


Fig. 11

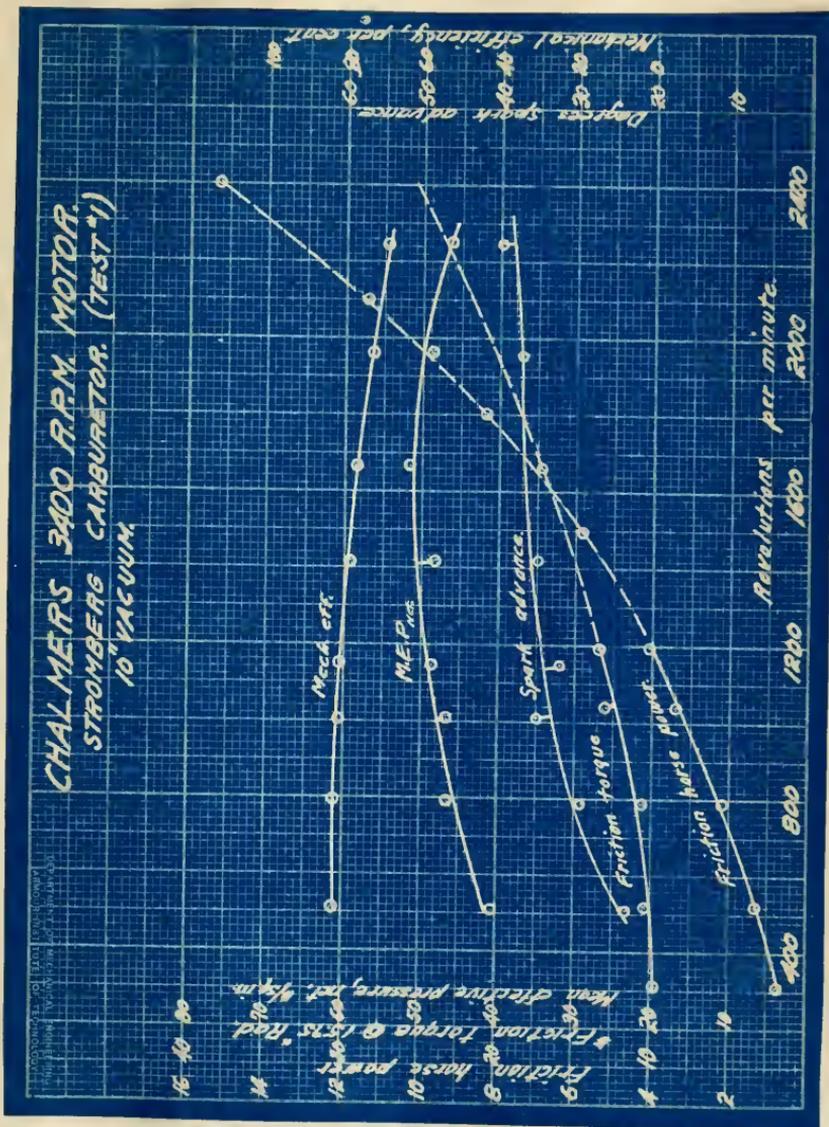


Fig. 12

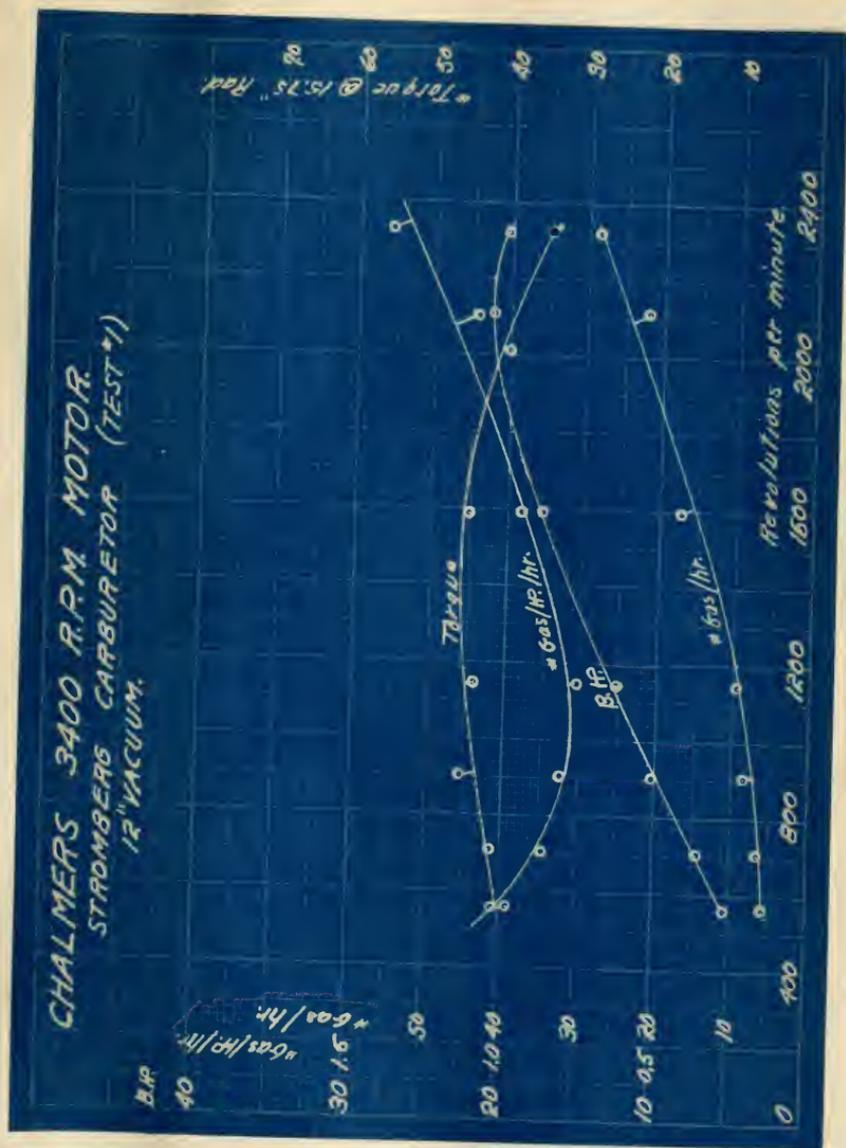


Fig. 13

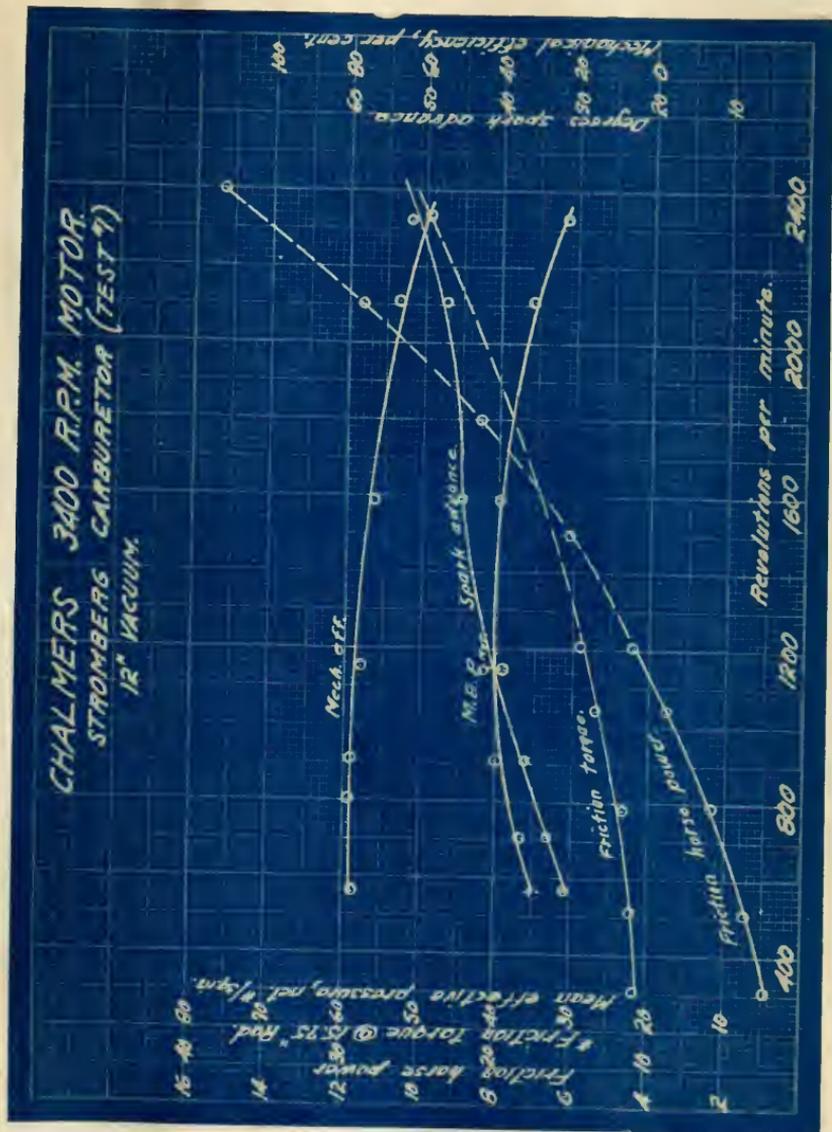


Fig. 14

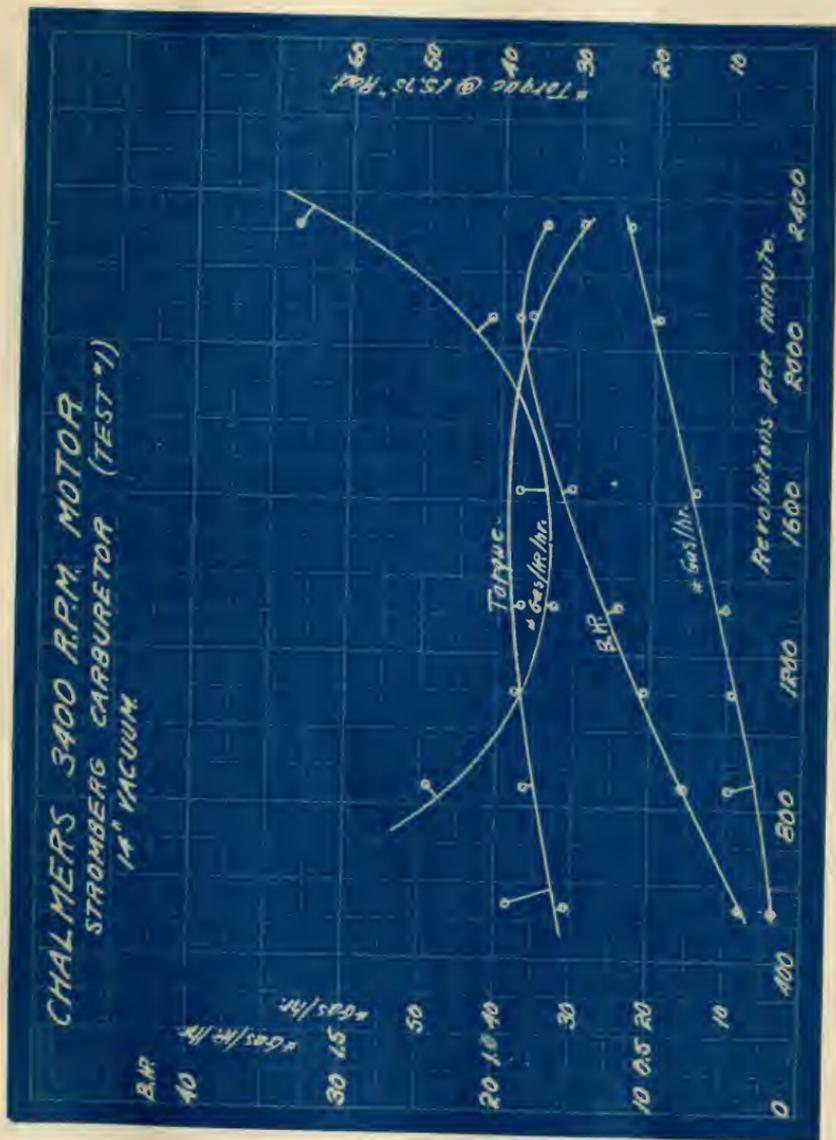


Fig. 15

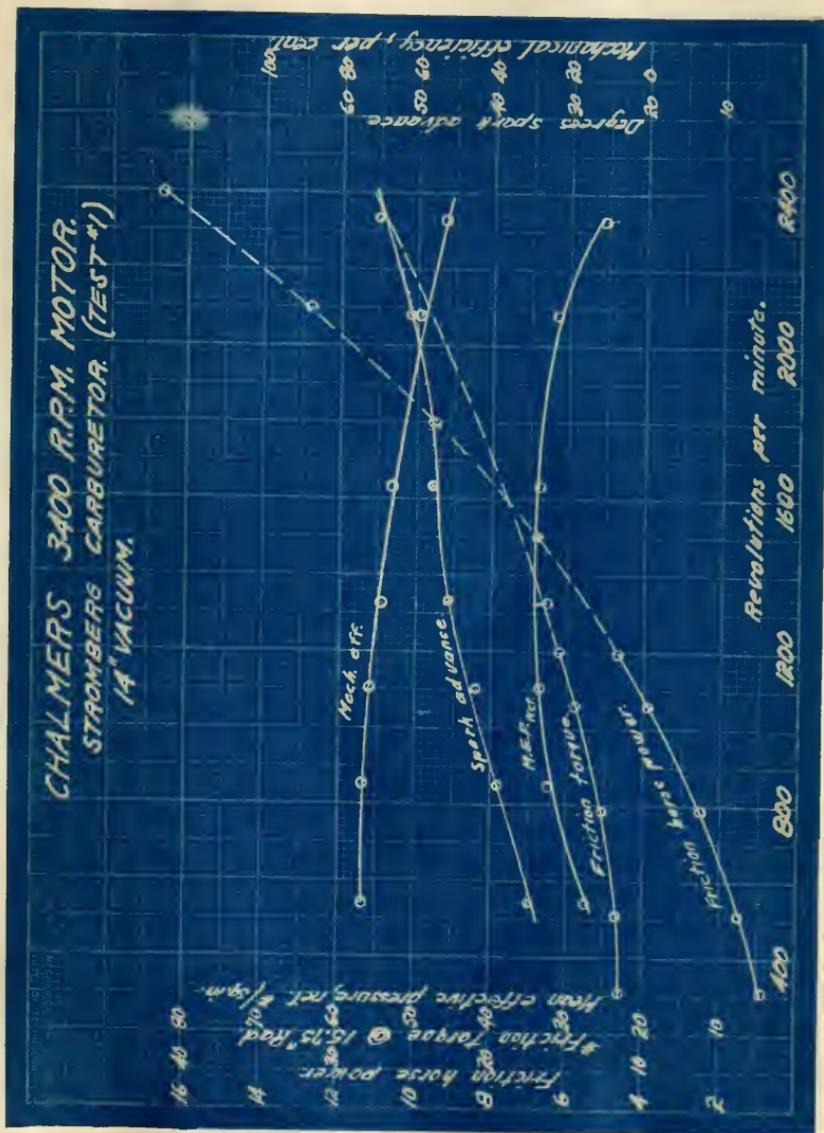
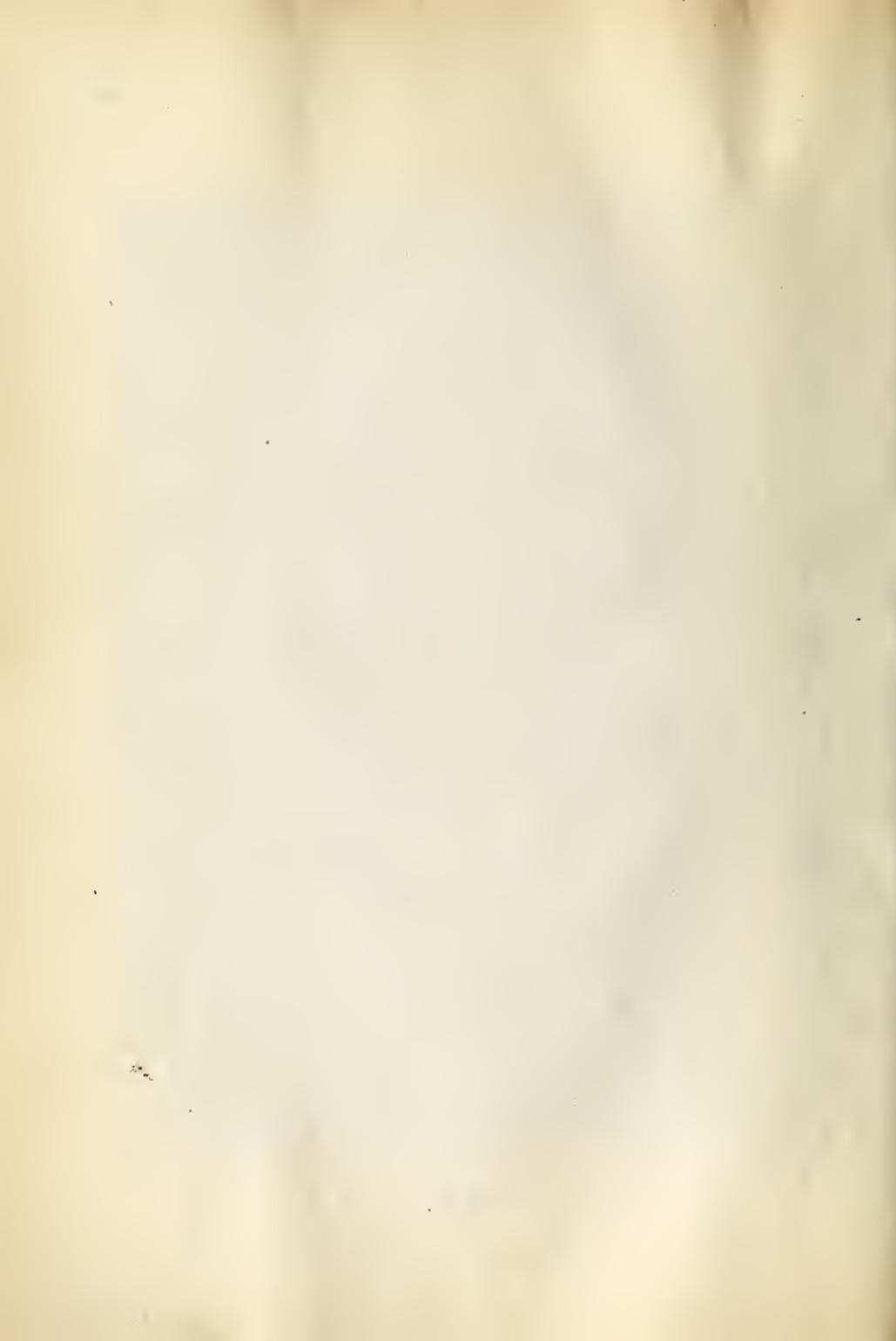


Fig. 16



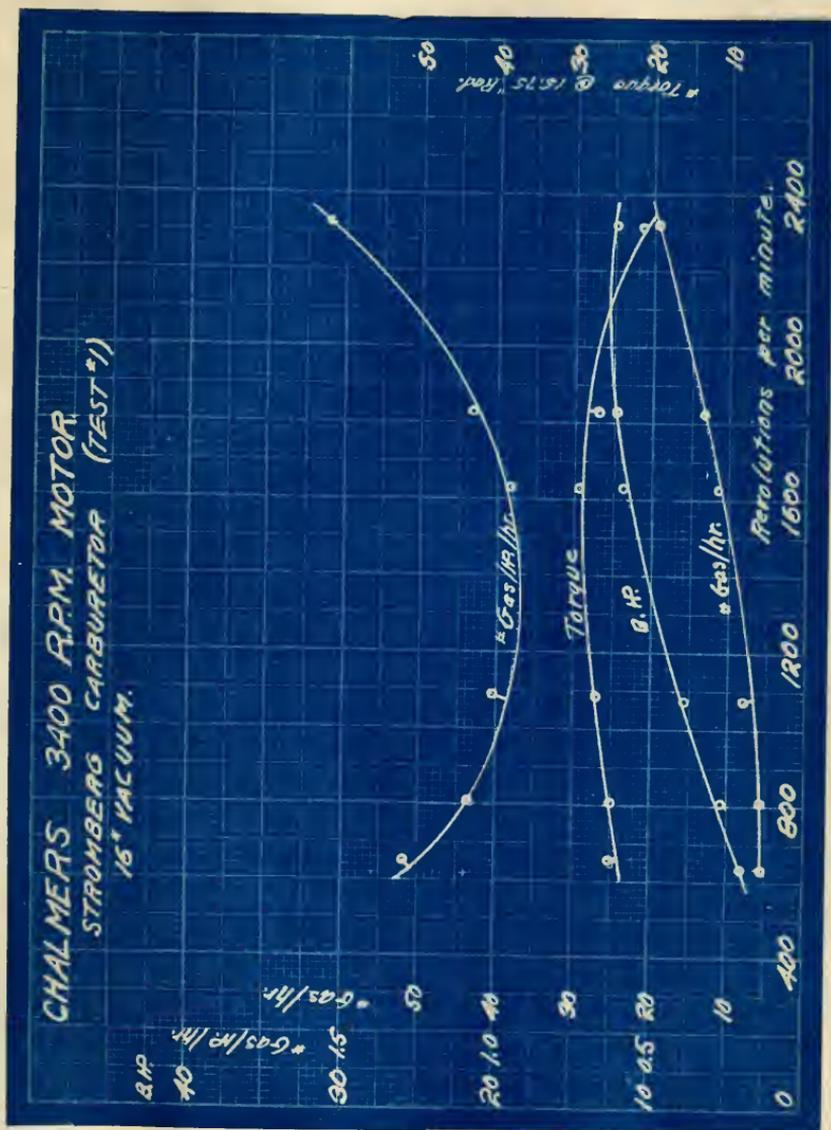


Fig. 17

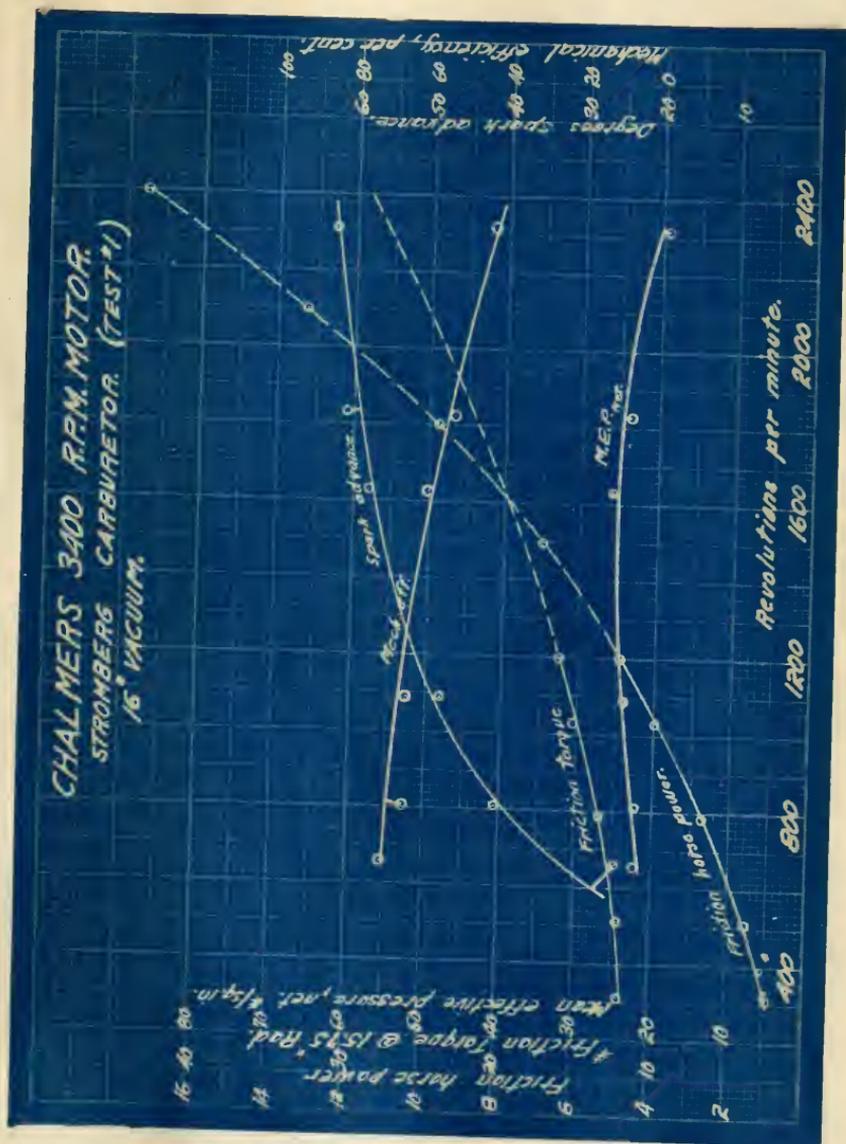


Fig. 18

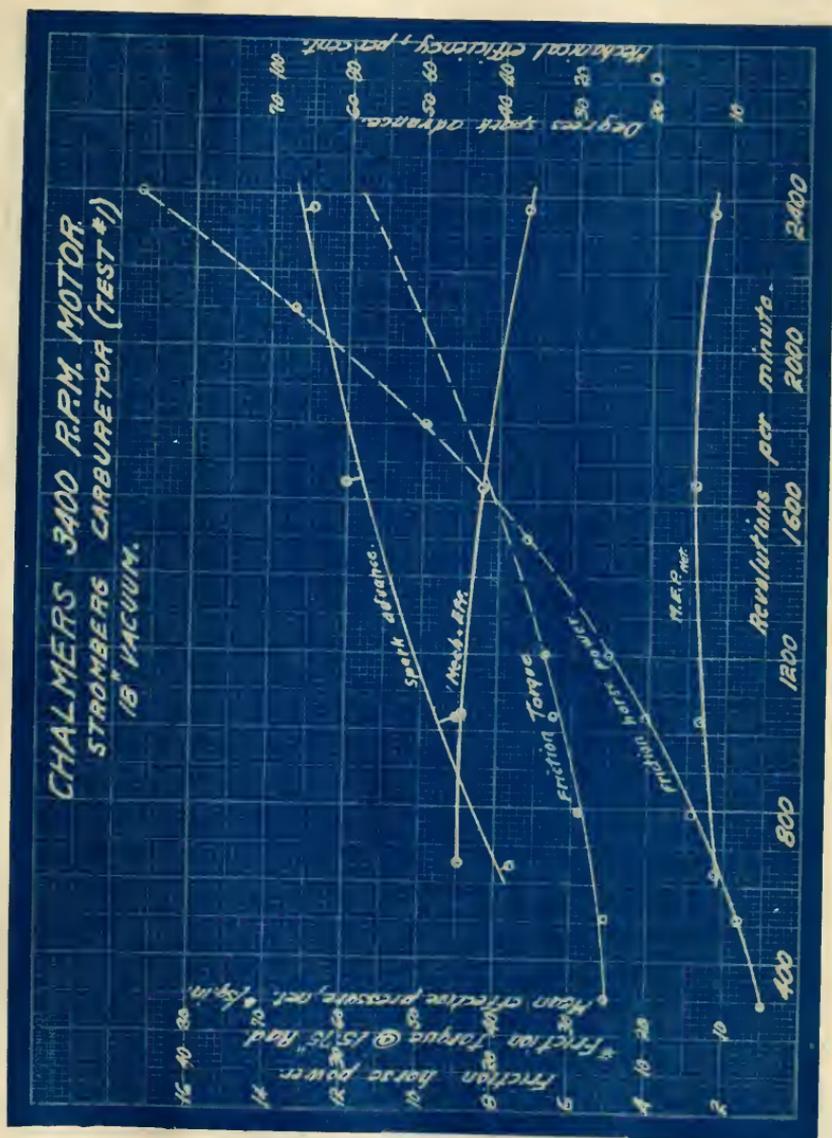


Fig. 20

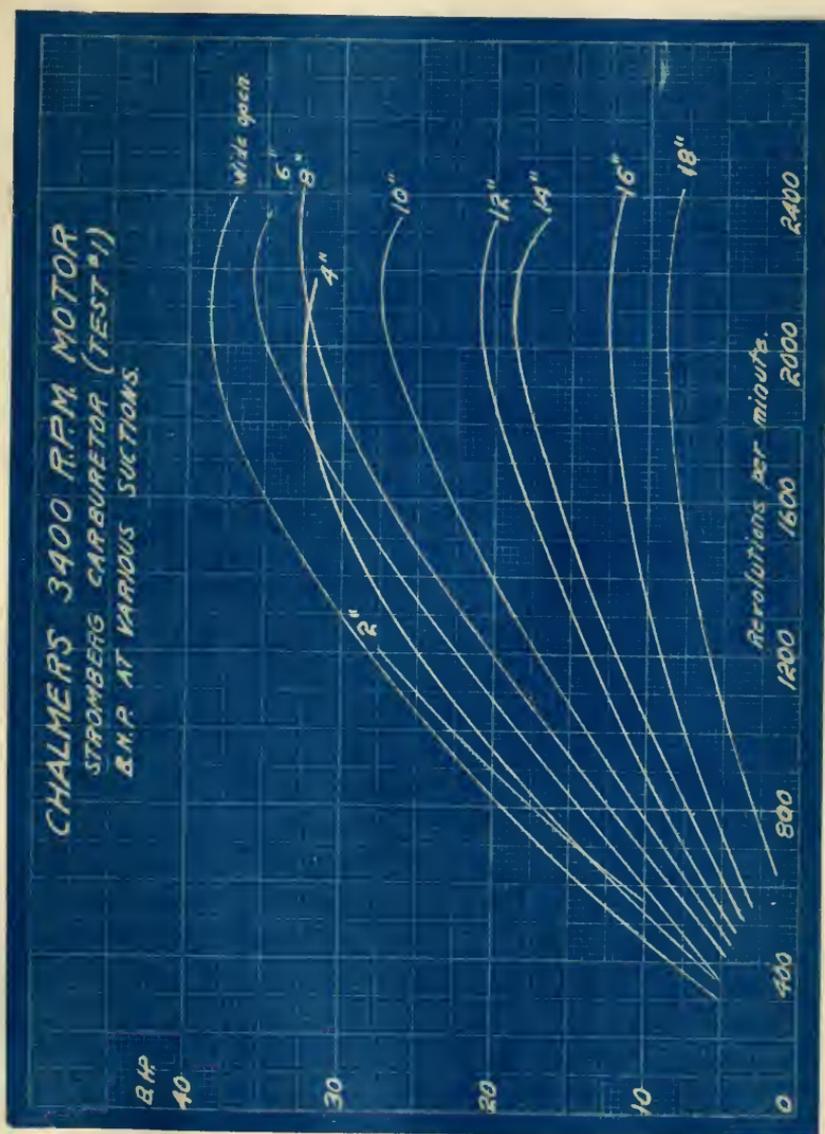


Fig. 21

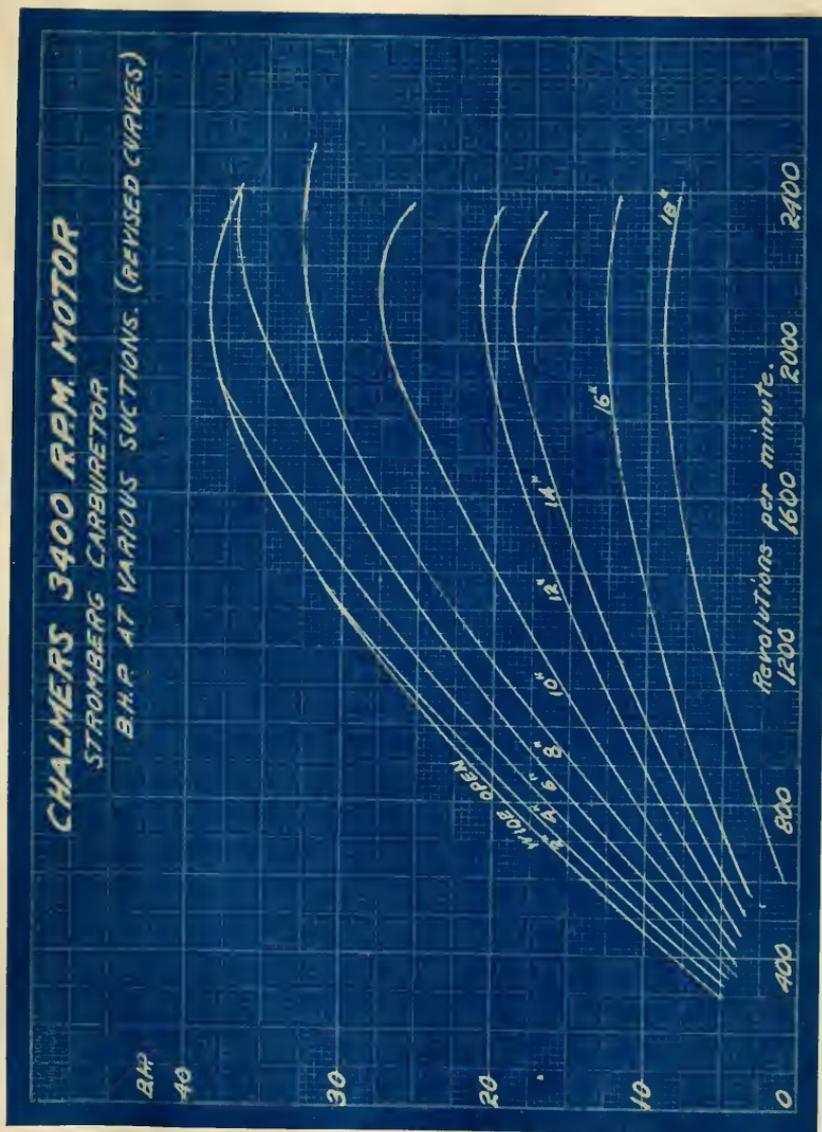


Fig. 22

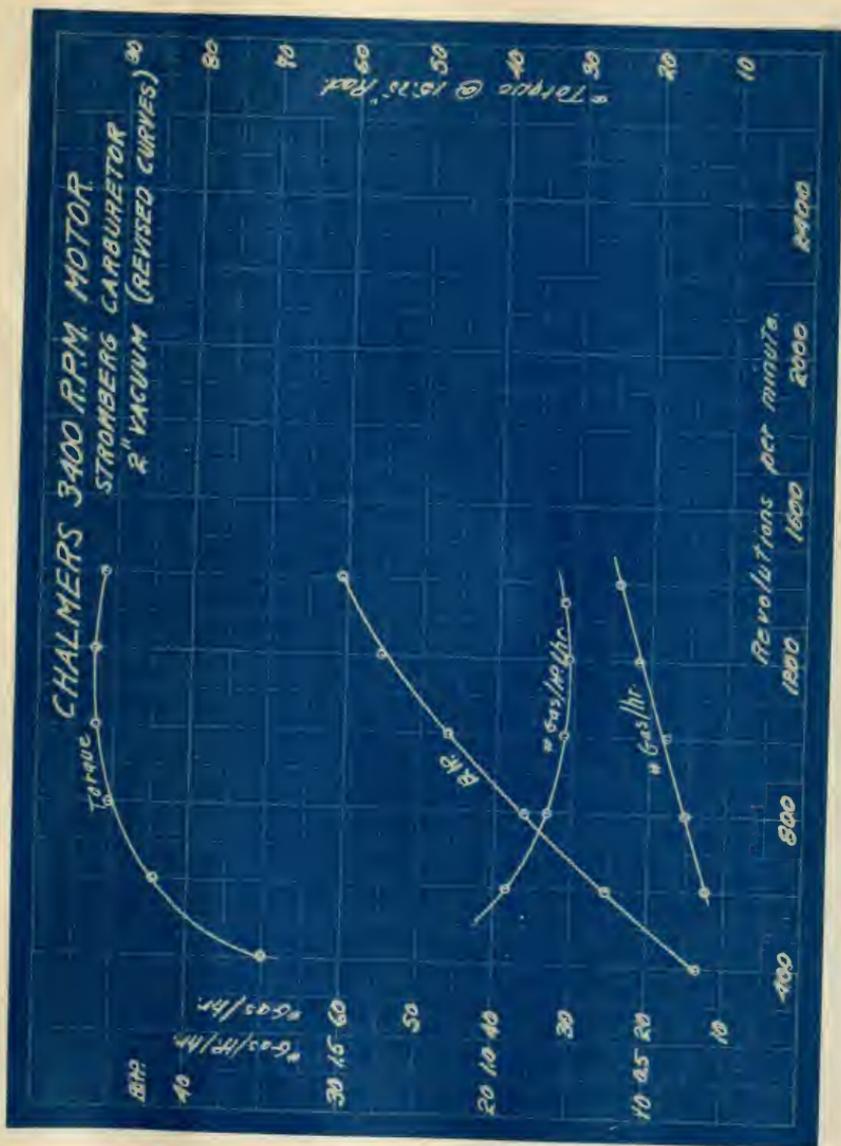


Fig. 24

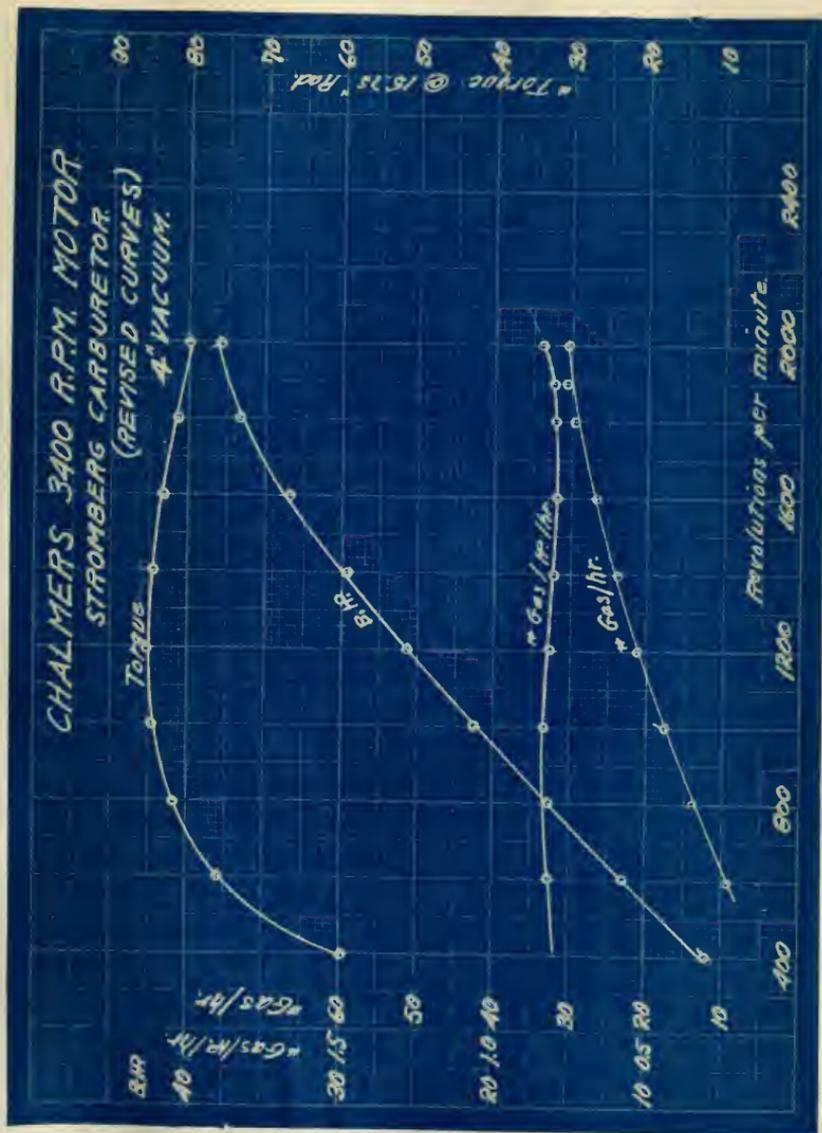


Fig. 25

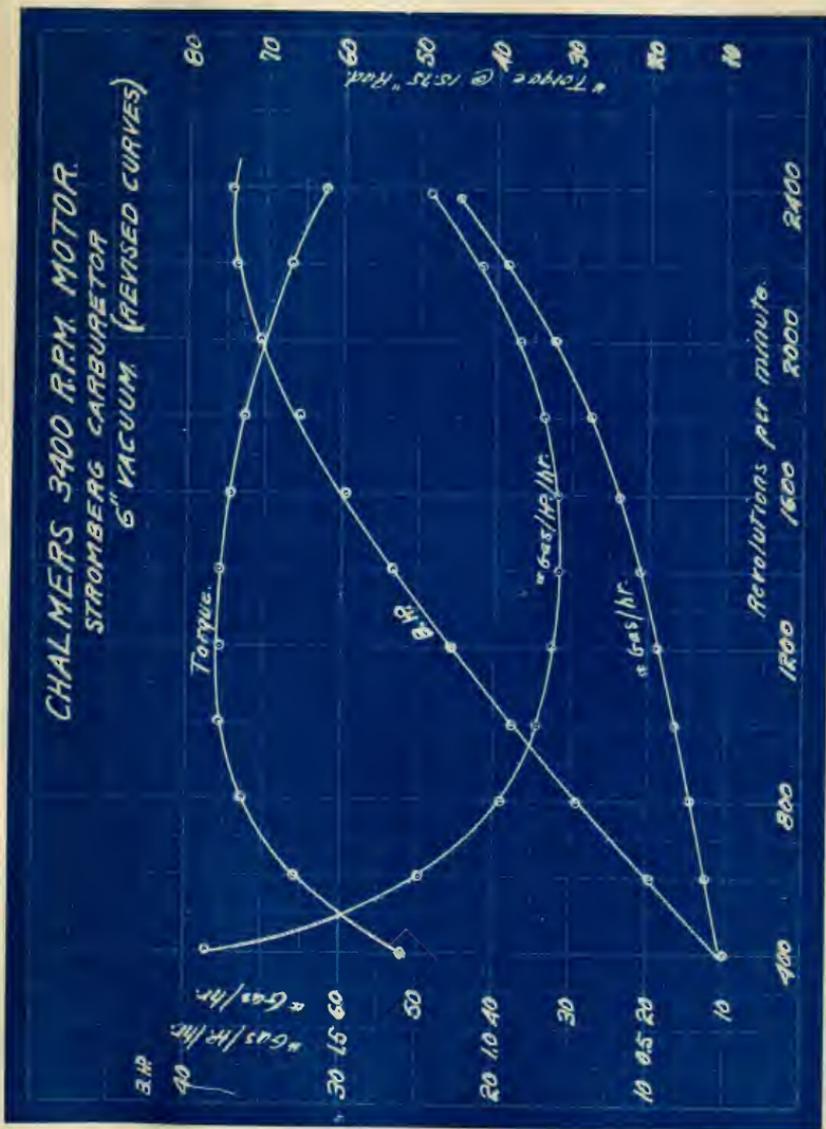


Fig. 26

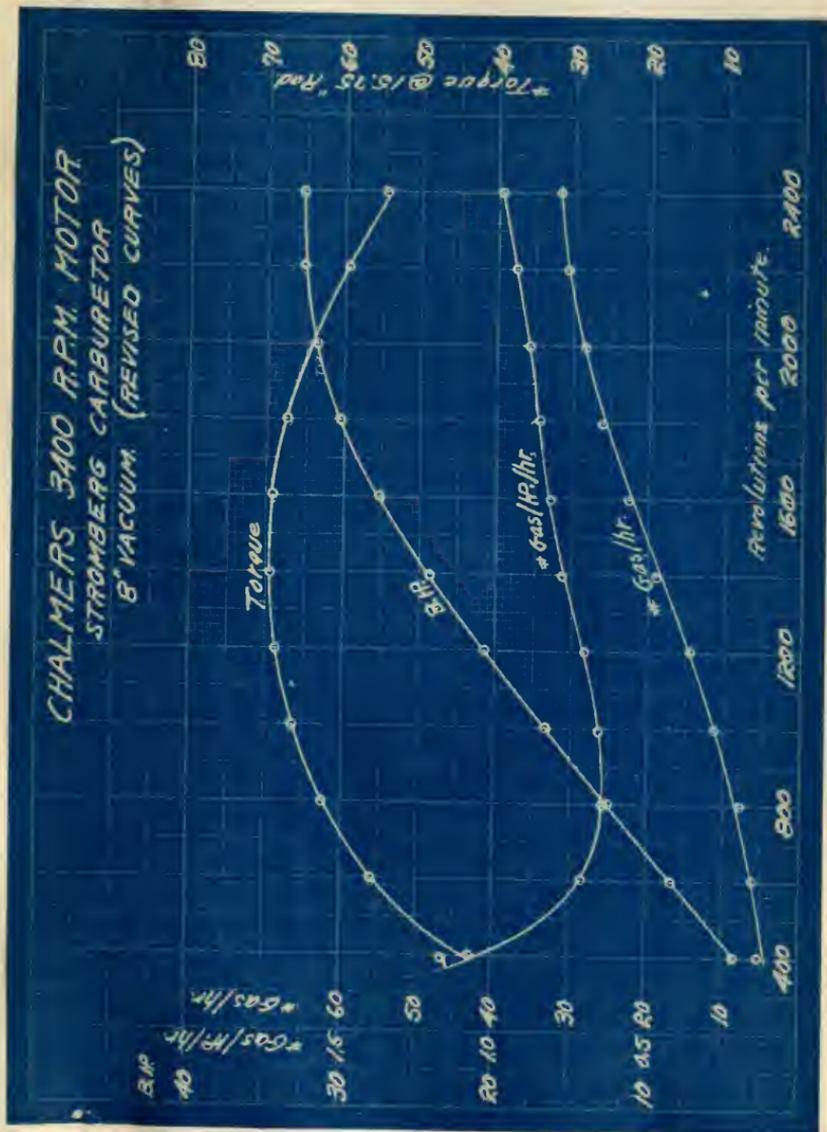


Fig. 27

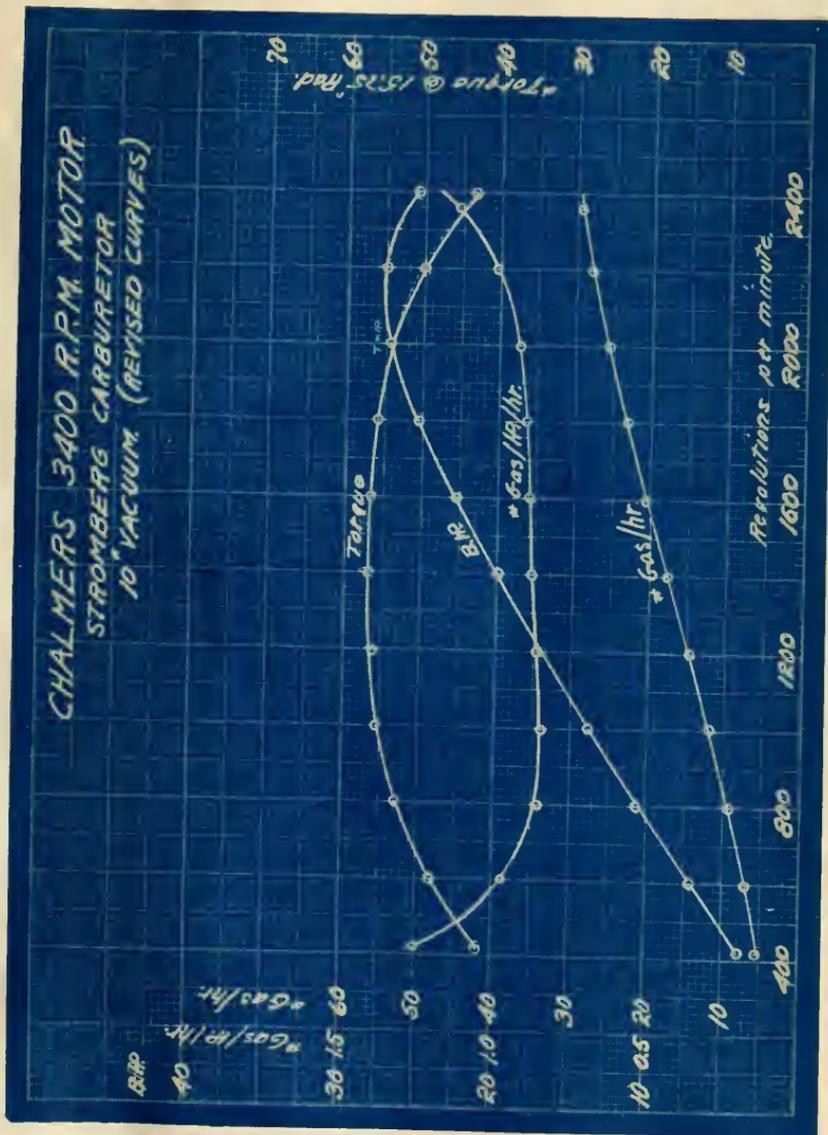


Fig. 28

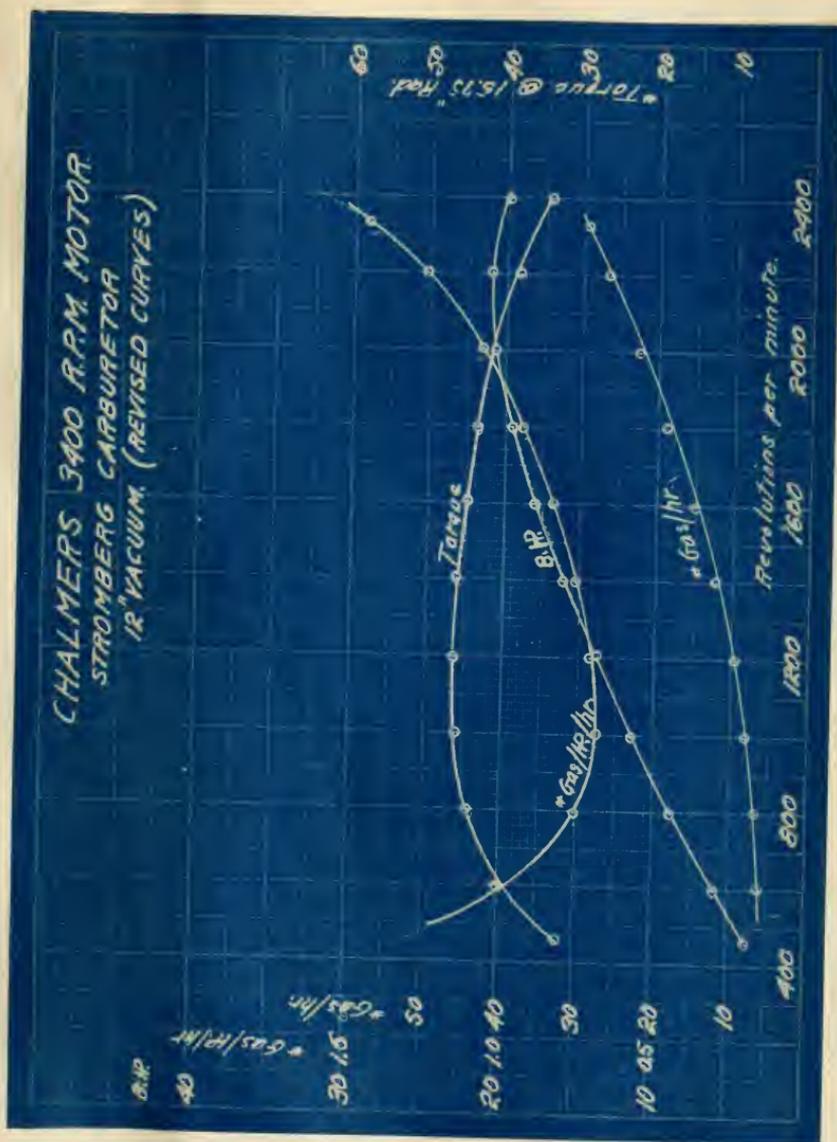


Fig. 29

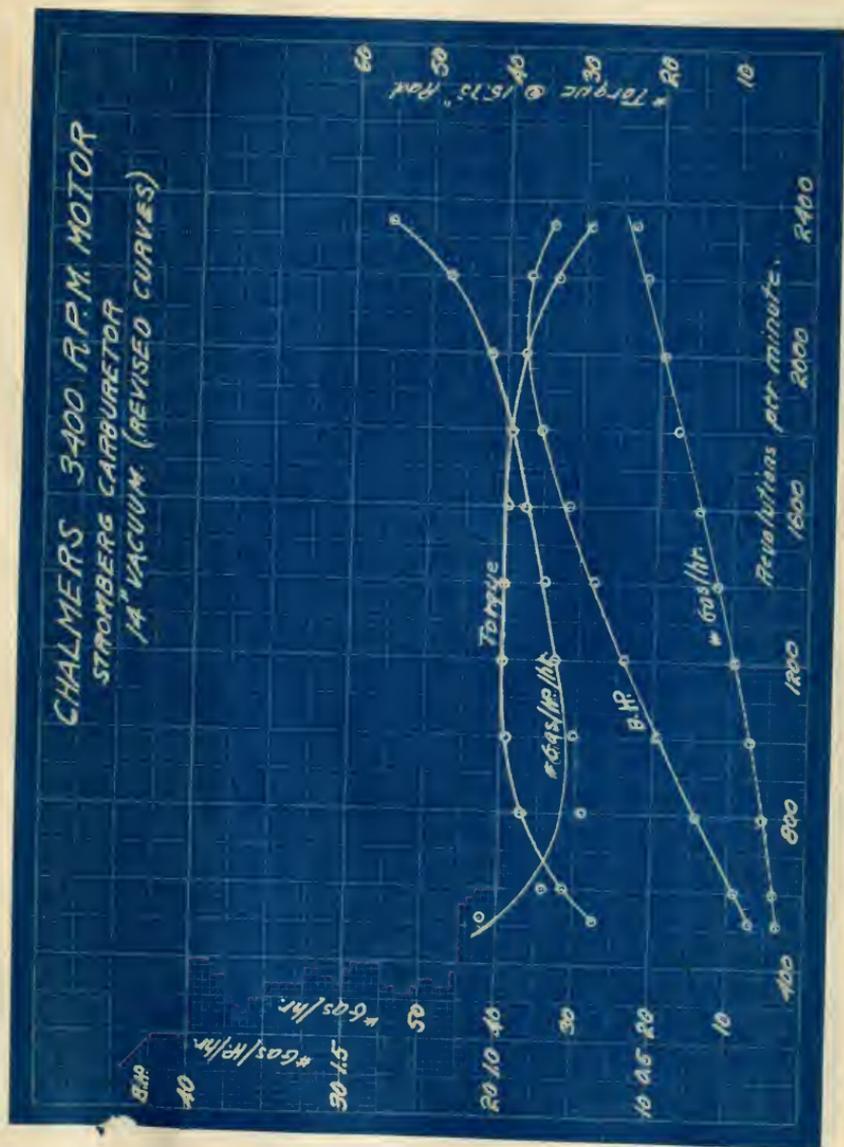


Fig. 30

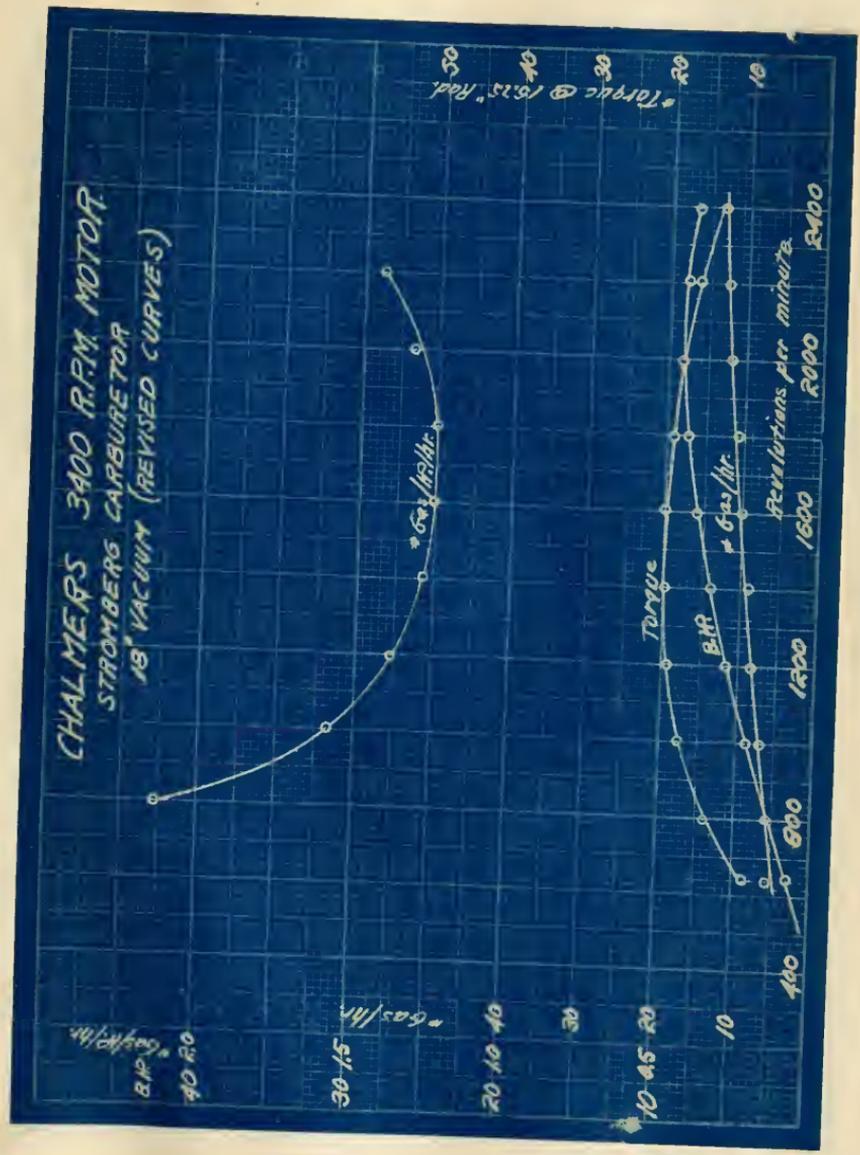


Fig. 32

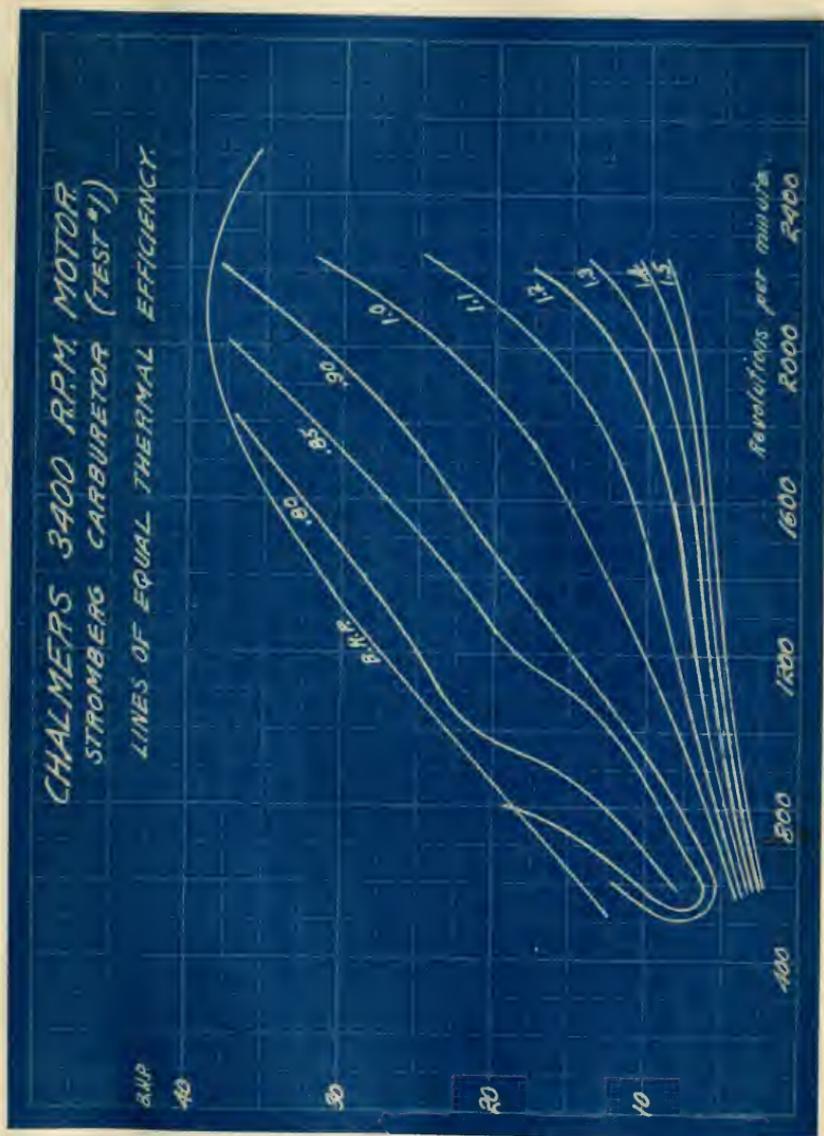


Fig. 33

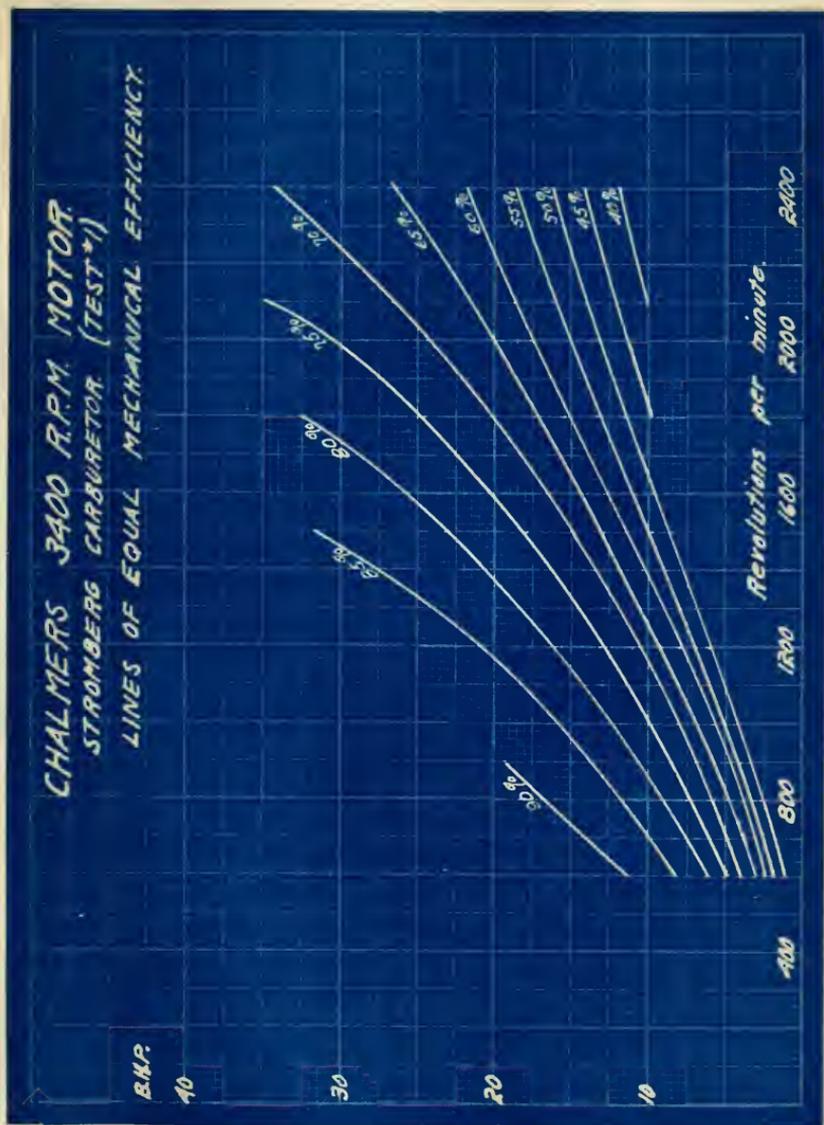


Fig. 34

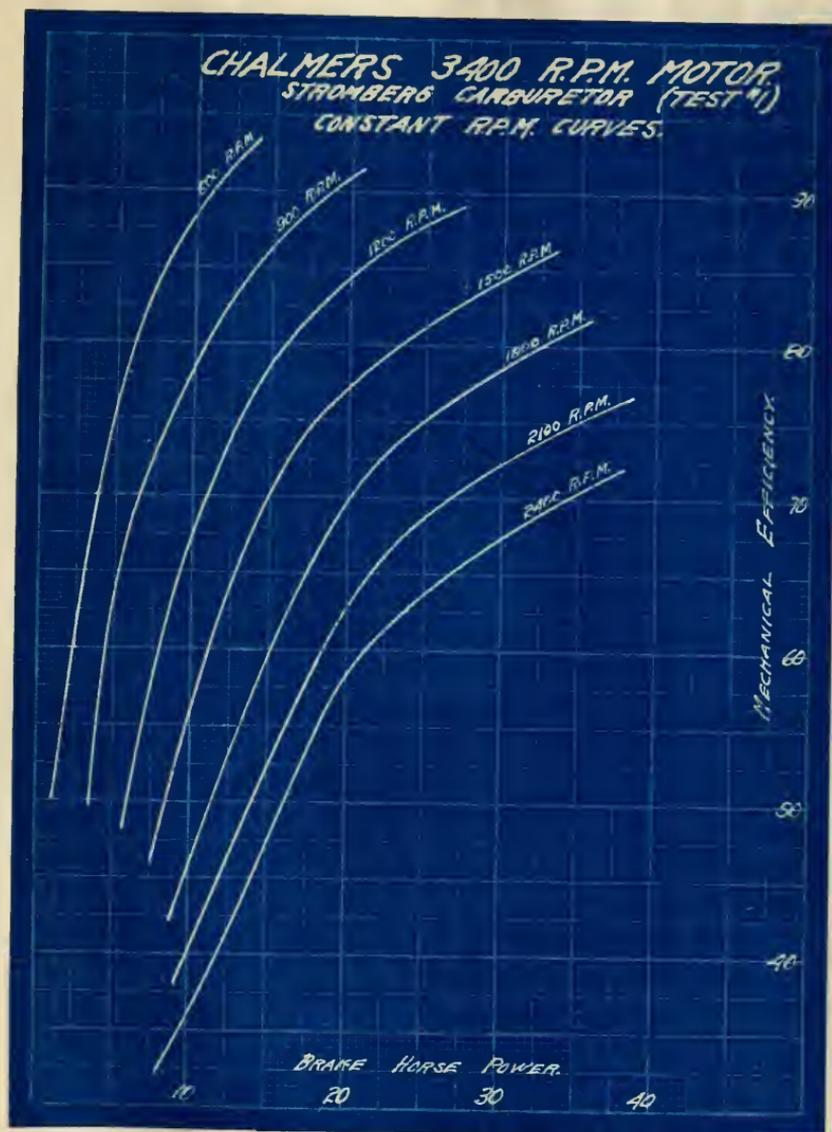


Fig. 35

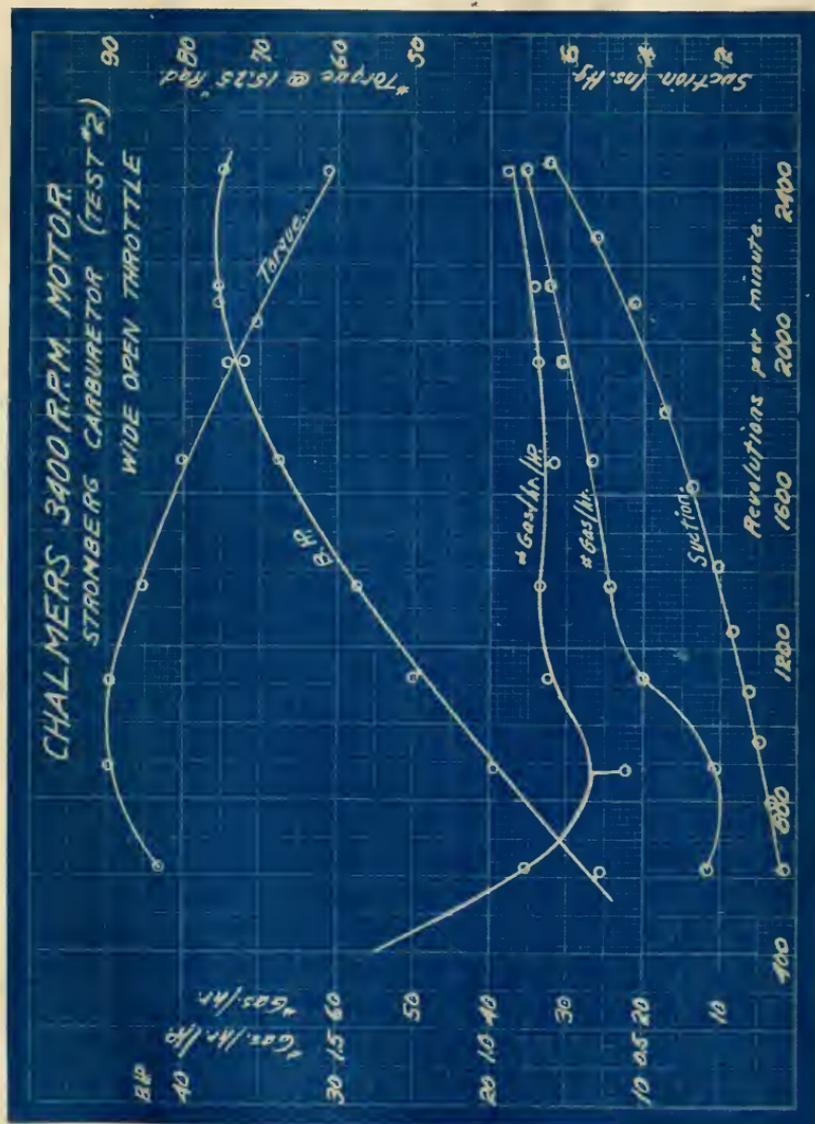


Fig. 36

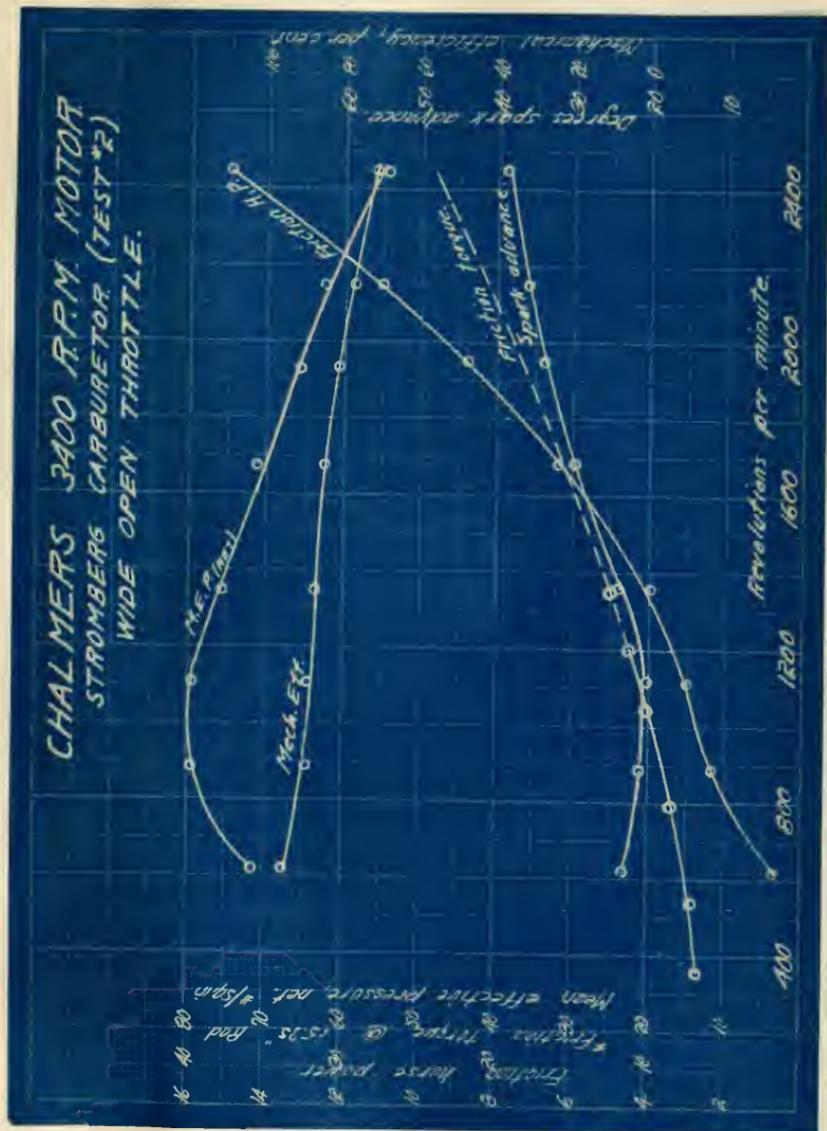


Fig. 37

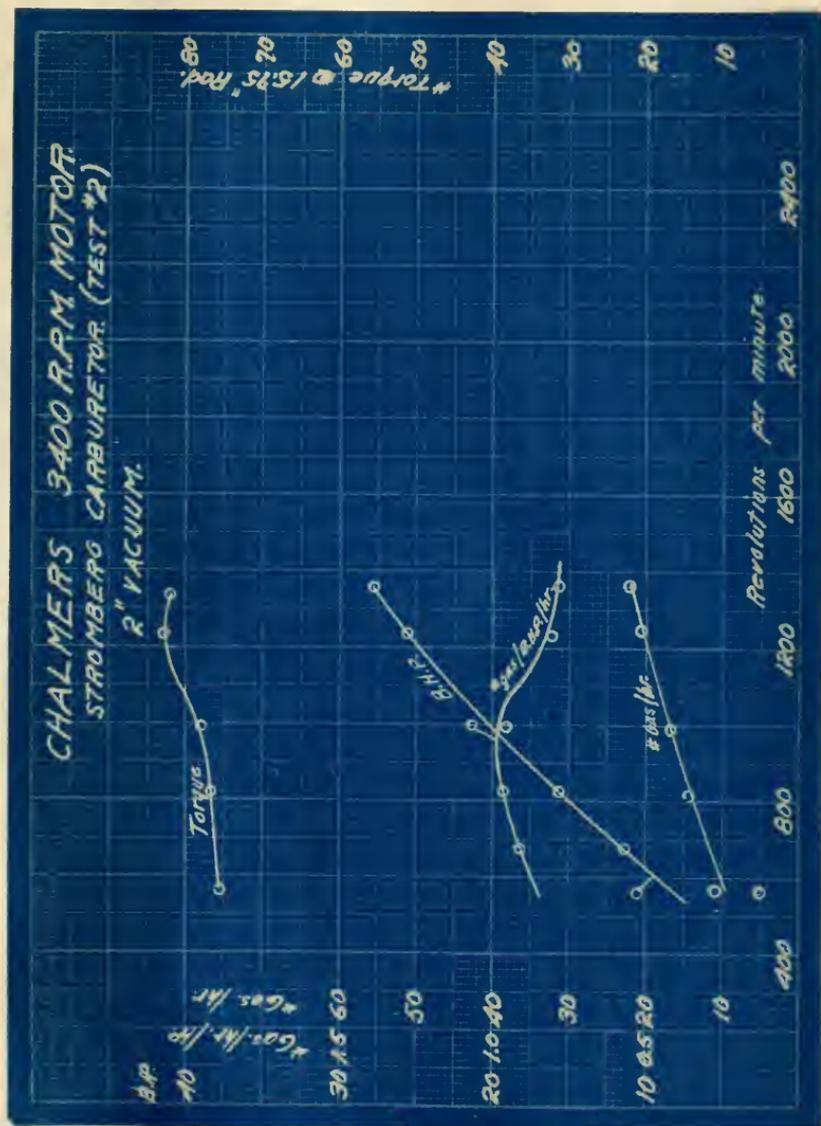


Fig. 36

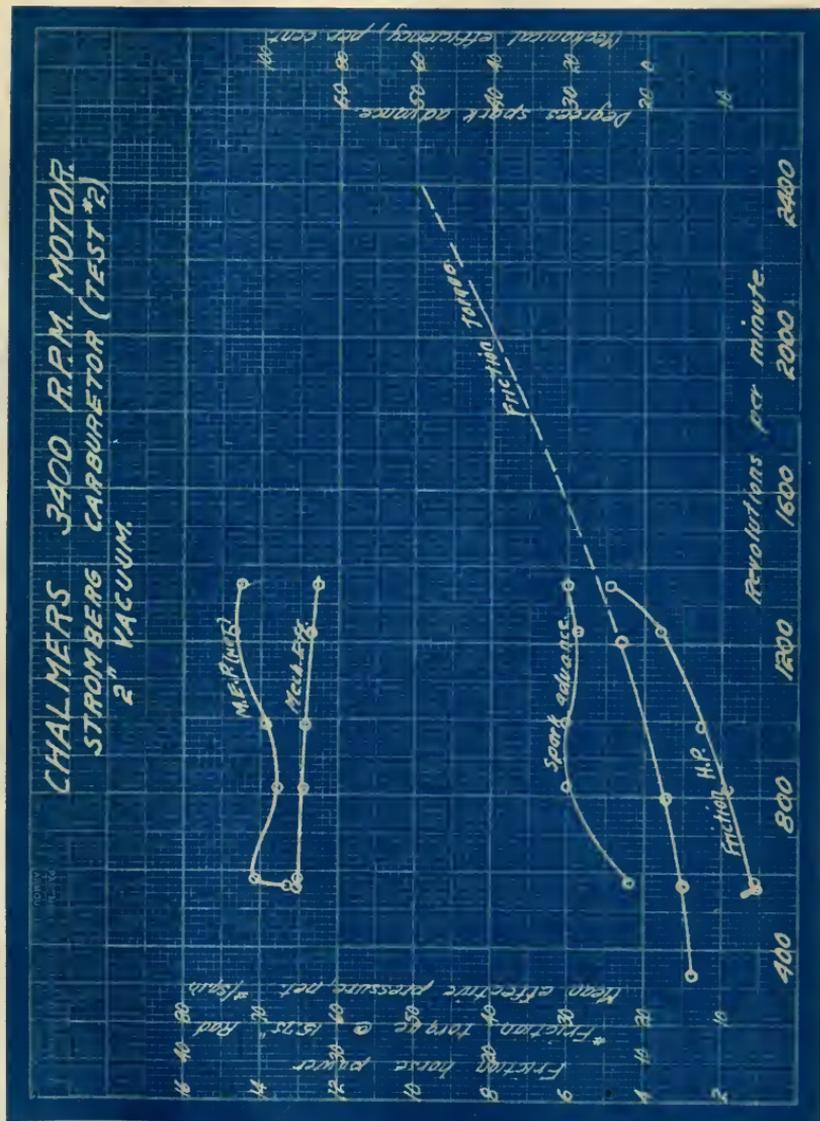


Fig. 39

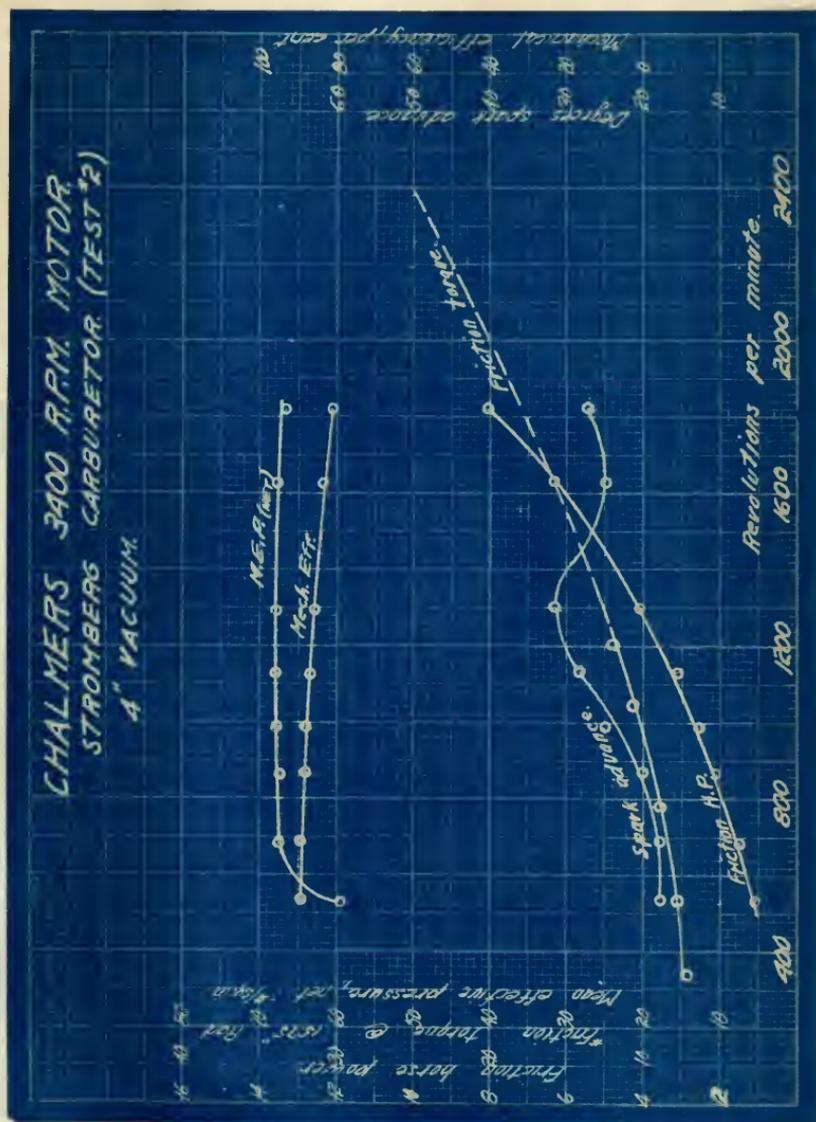


Fig. 41

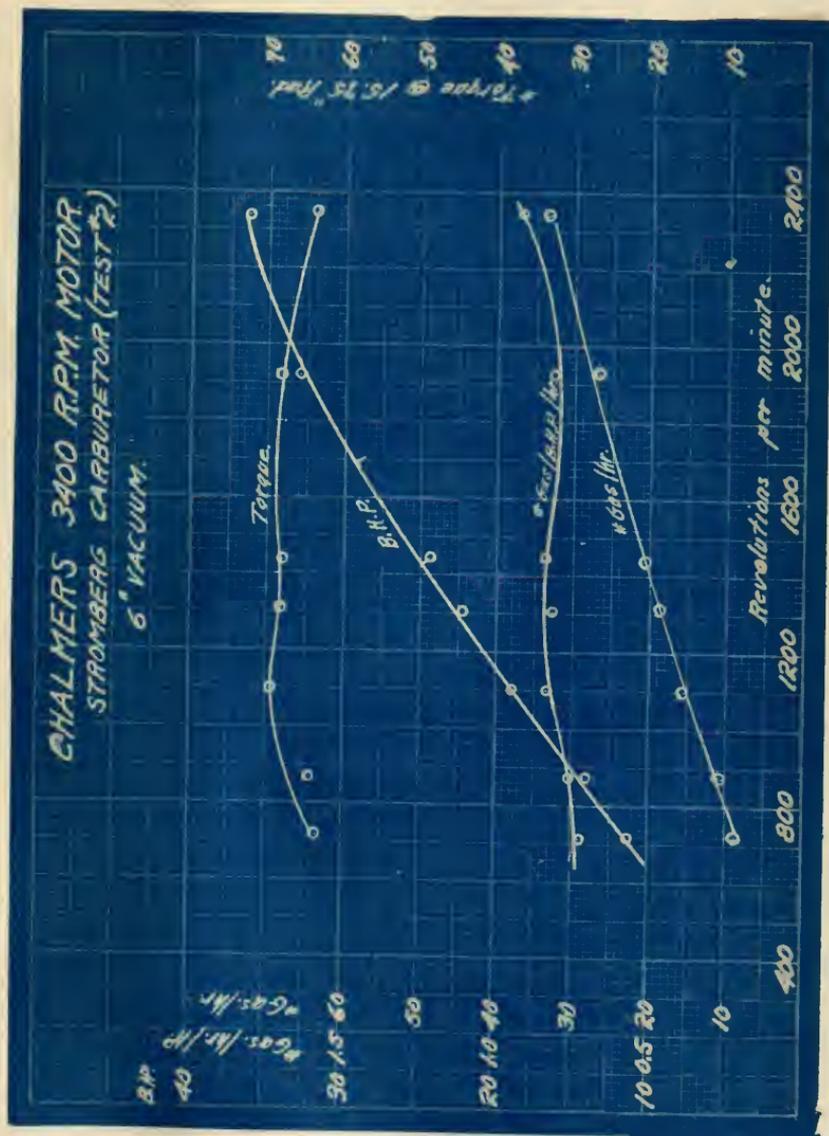


Fig. 42

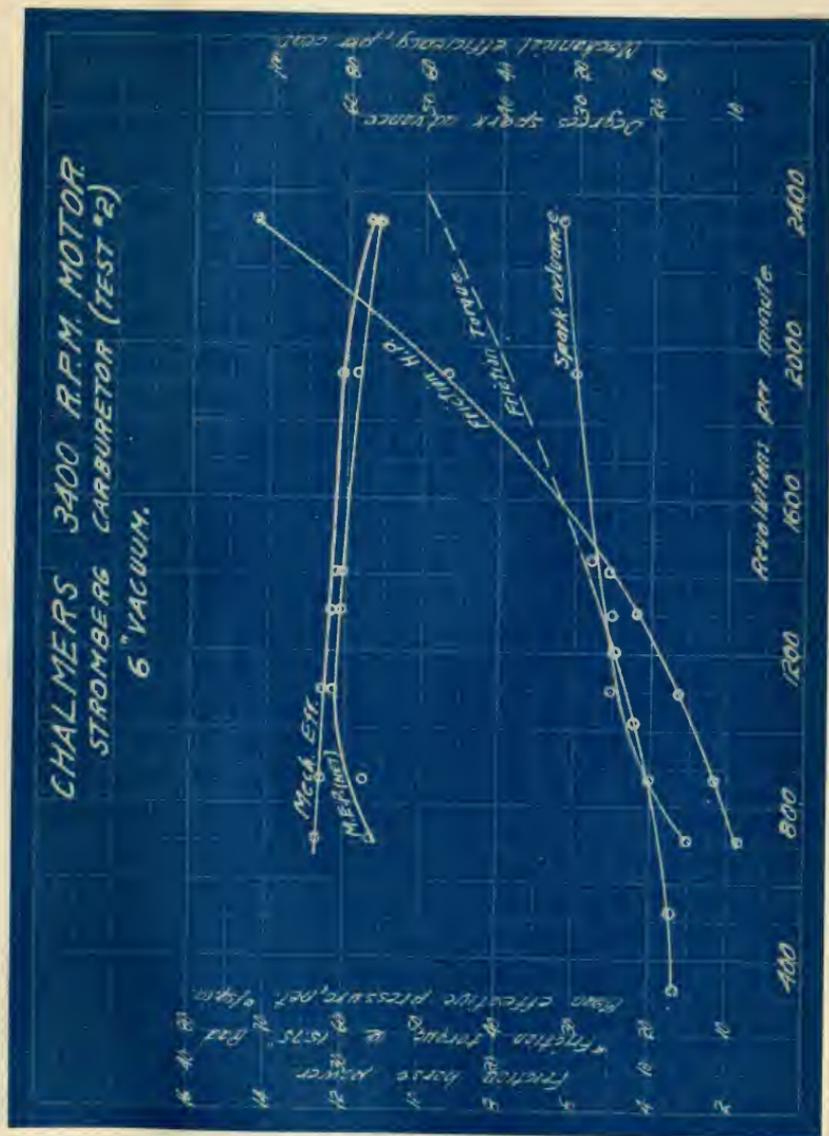


Fig. 43

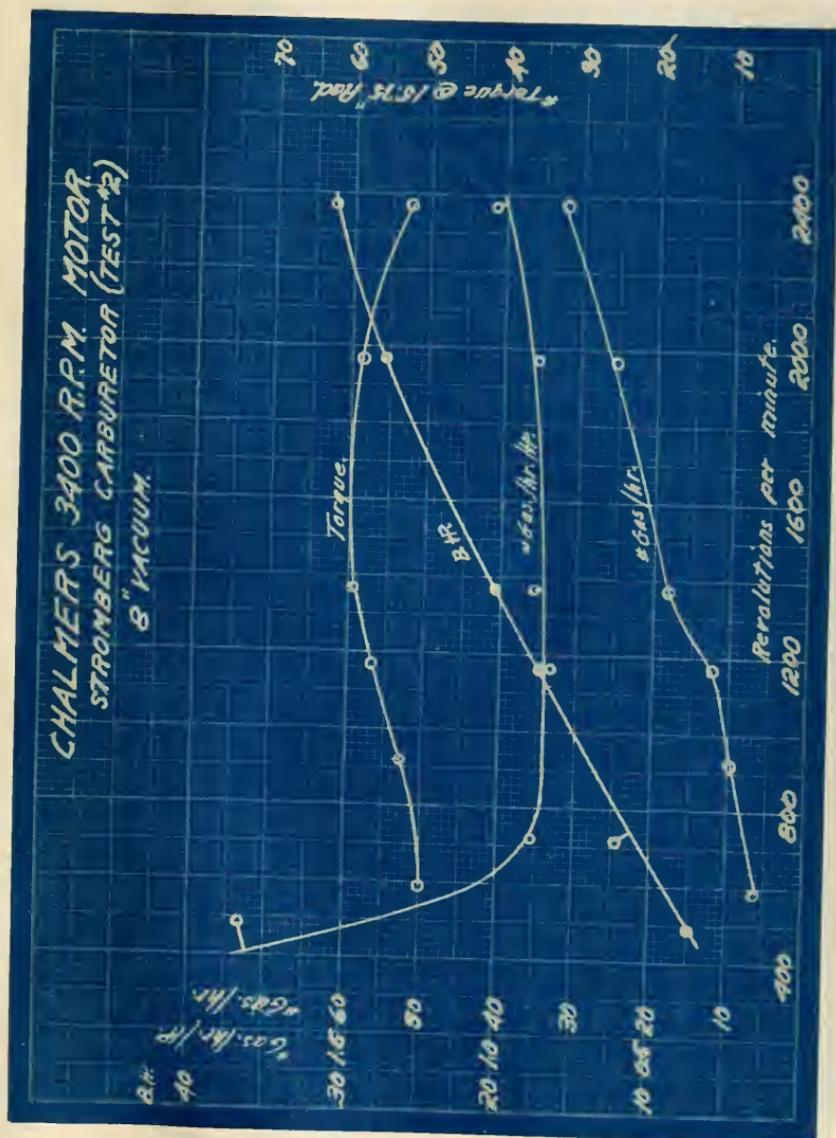


Fig. 44

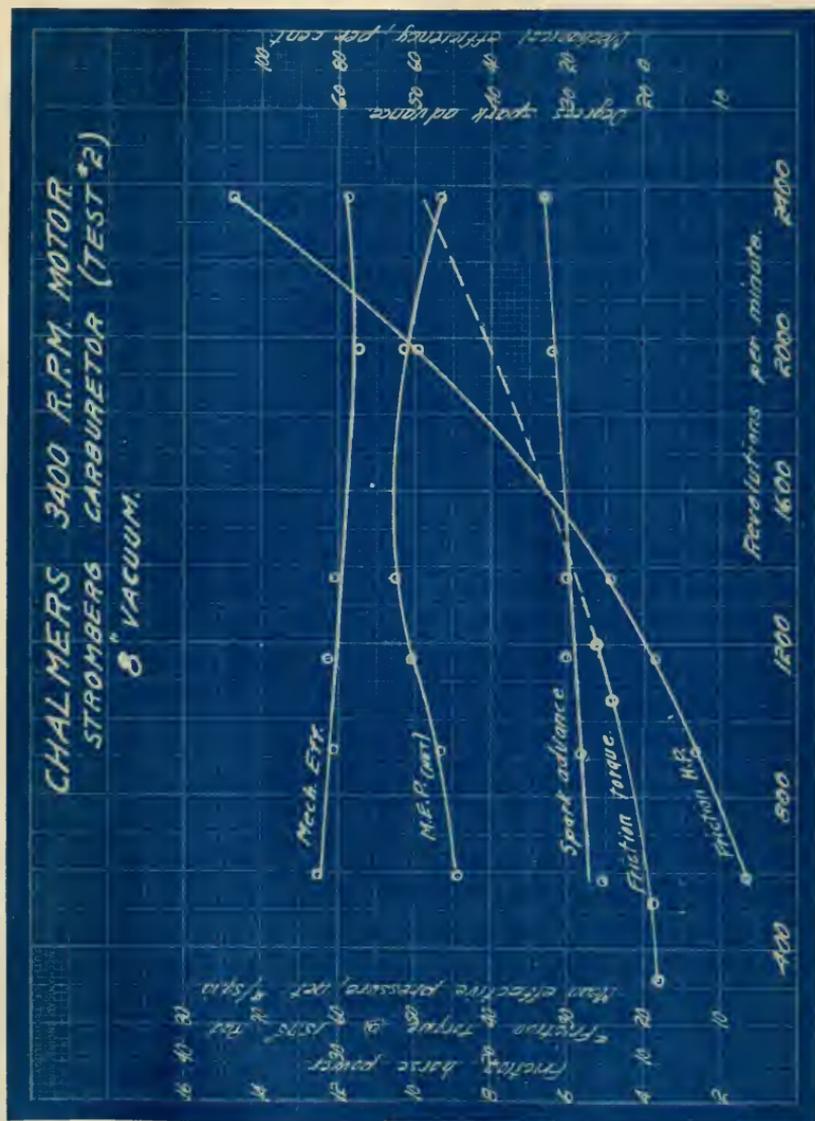


Fig. 45

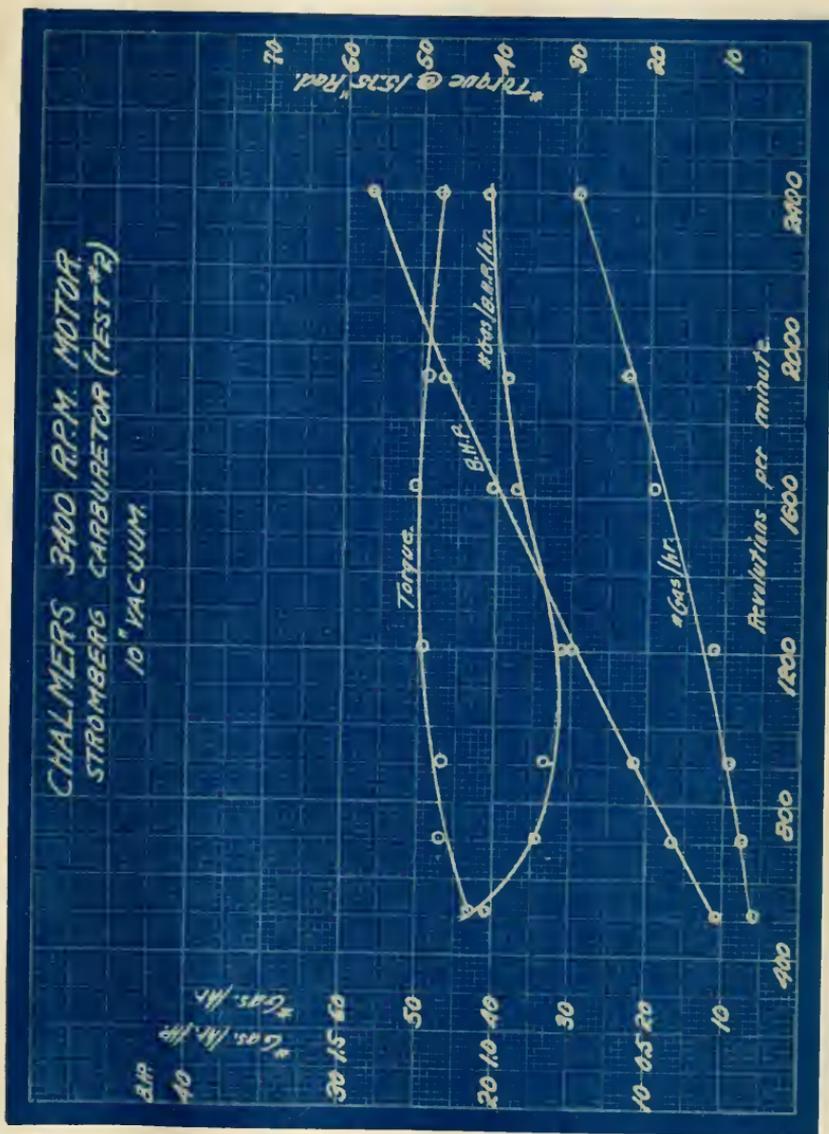


Fig. 46

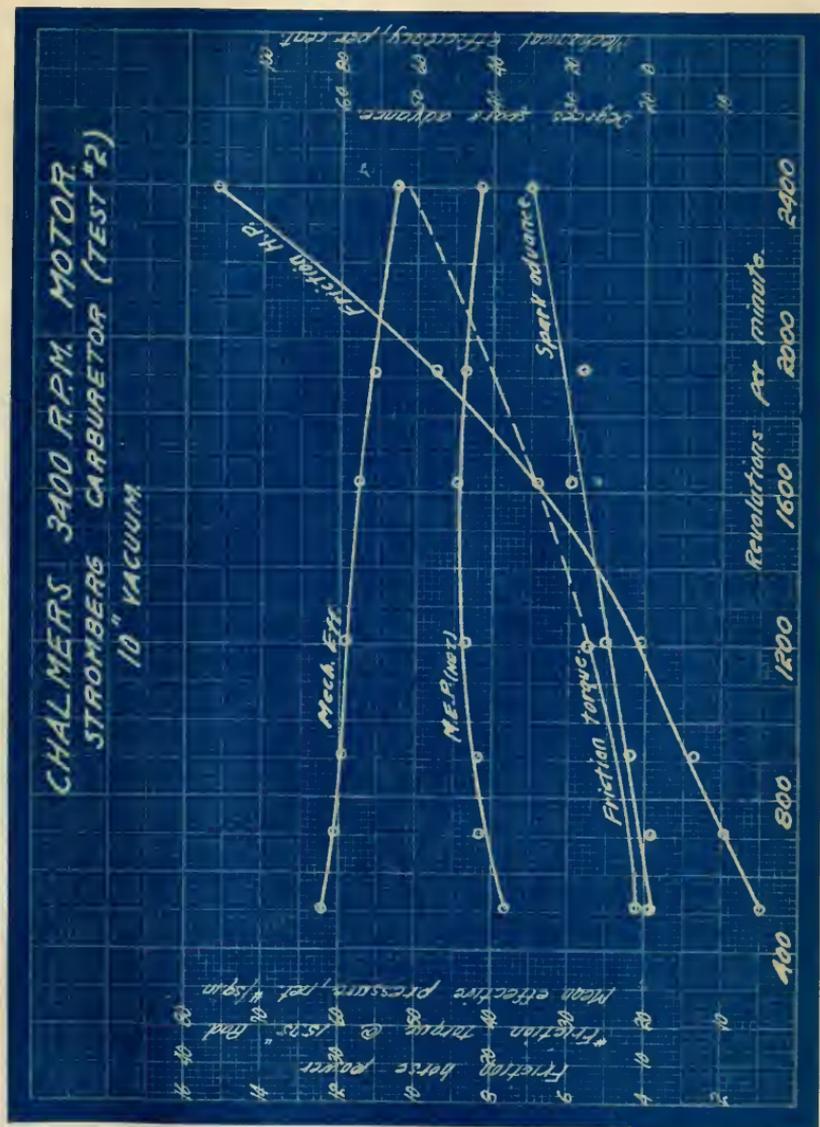


Fig. 47

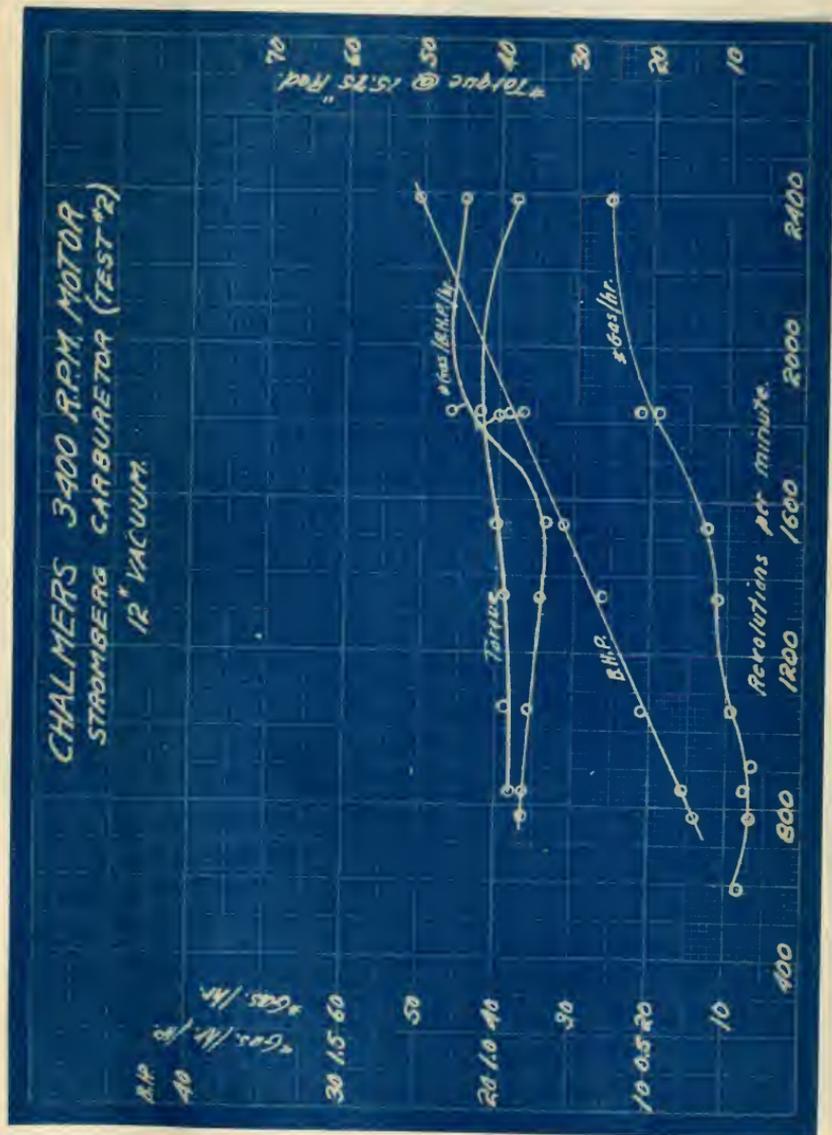


Fig. 48

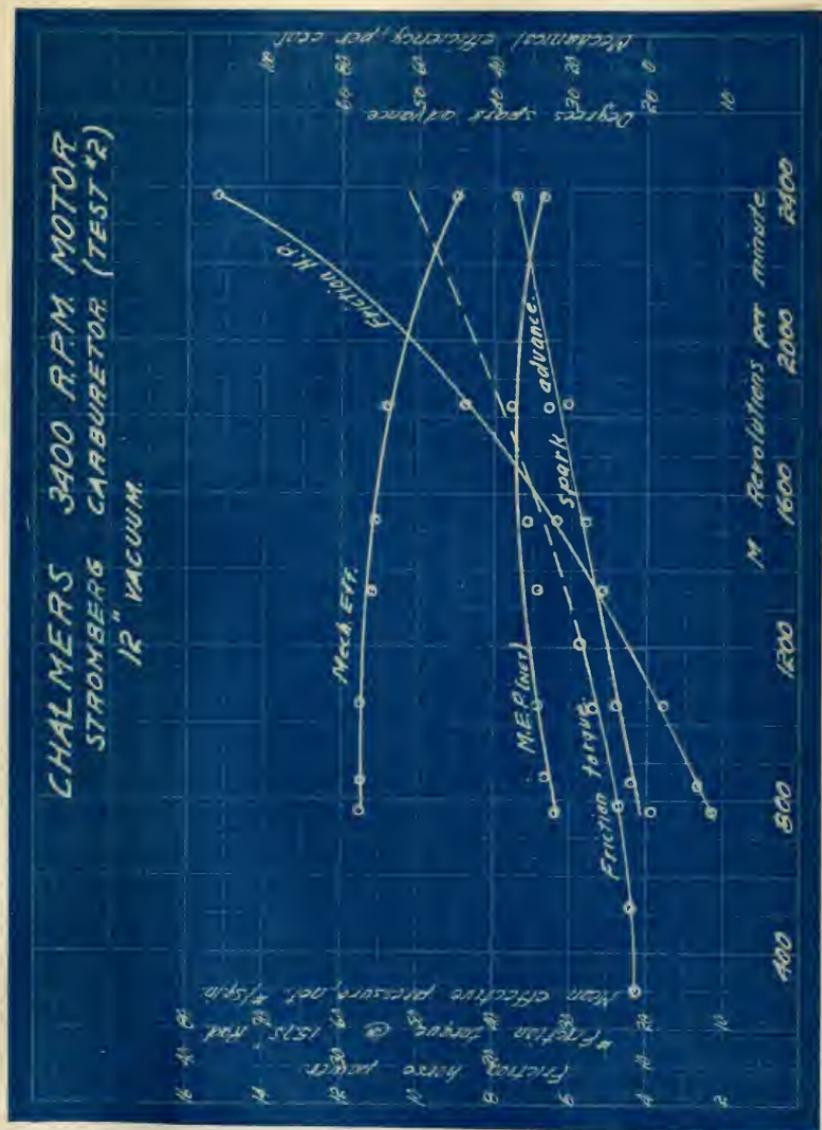


Fig. 49

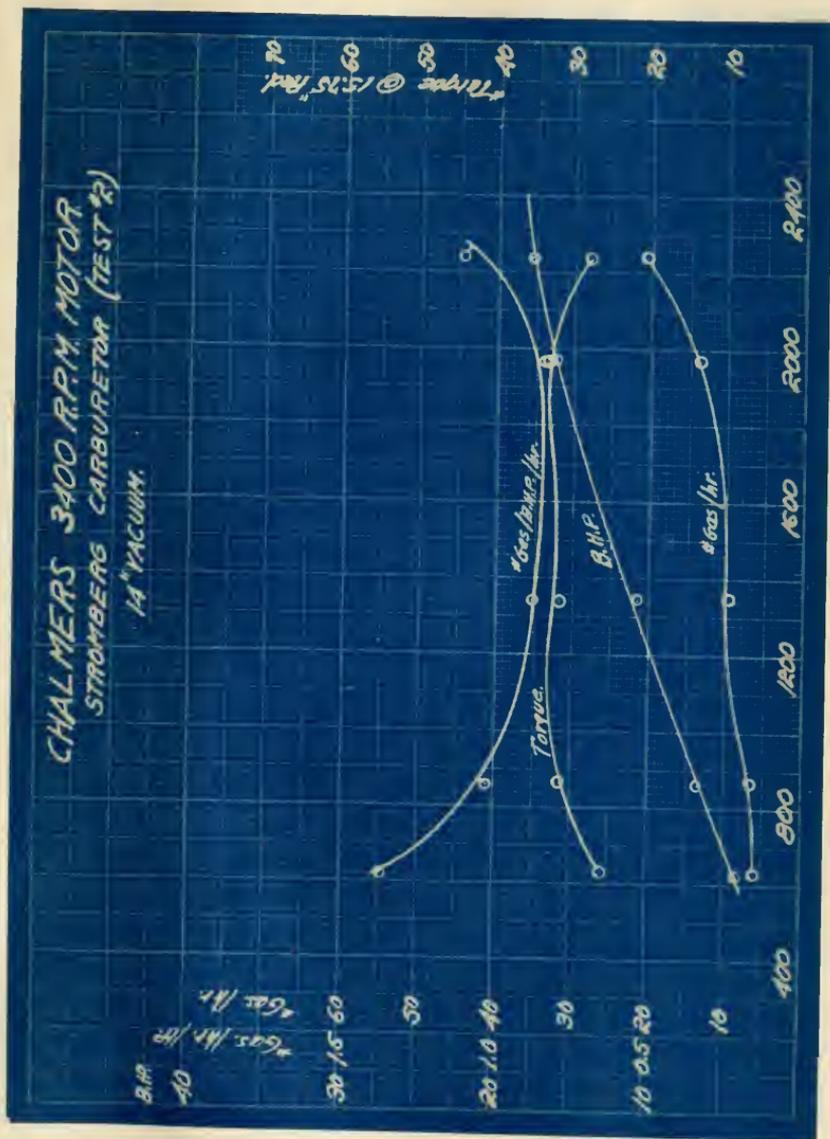


Fig. 50

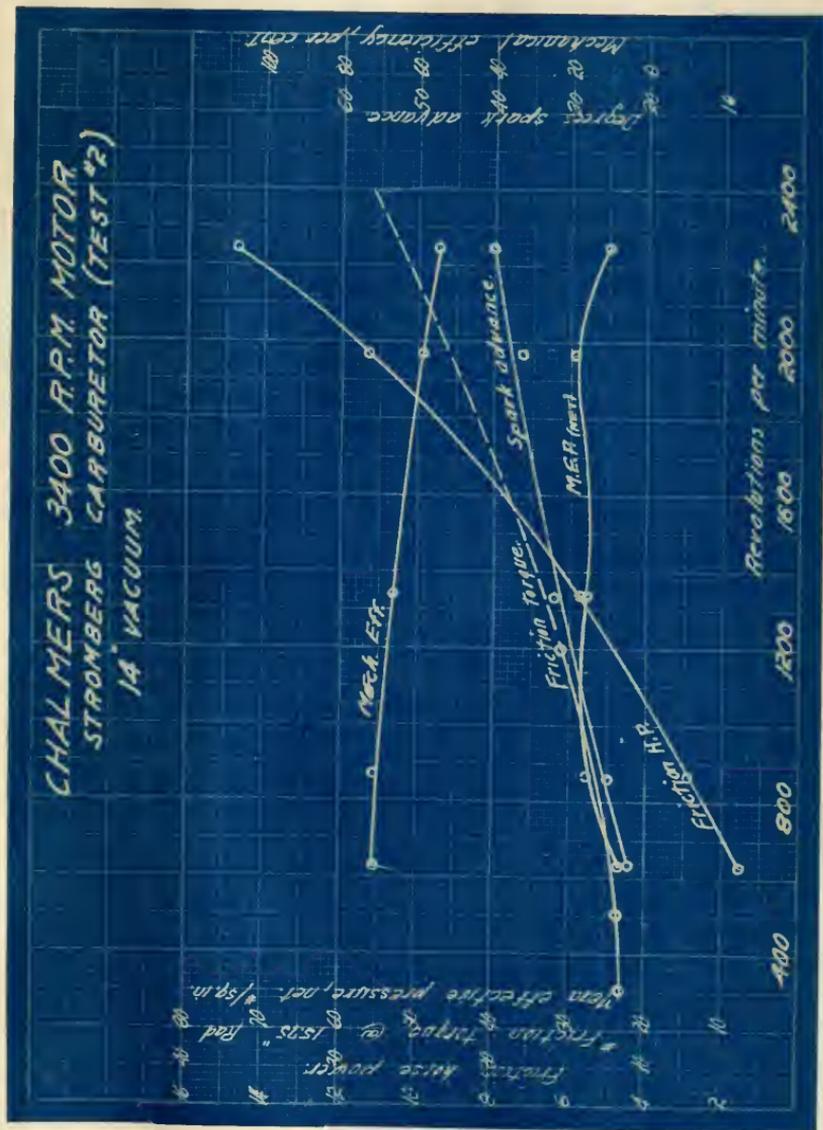


Fig. 51

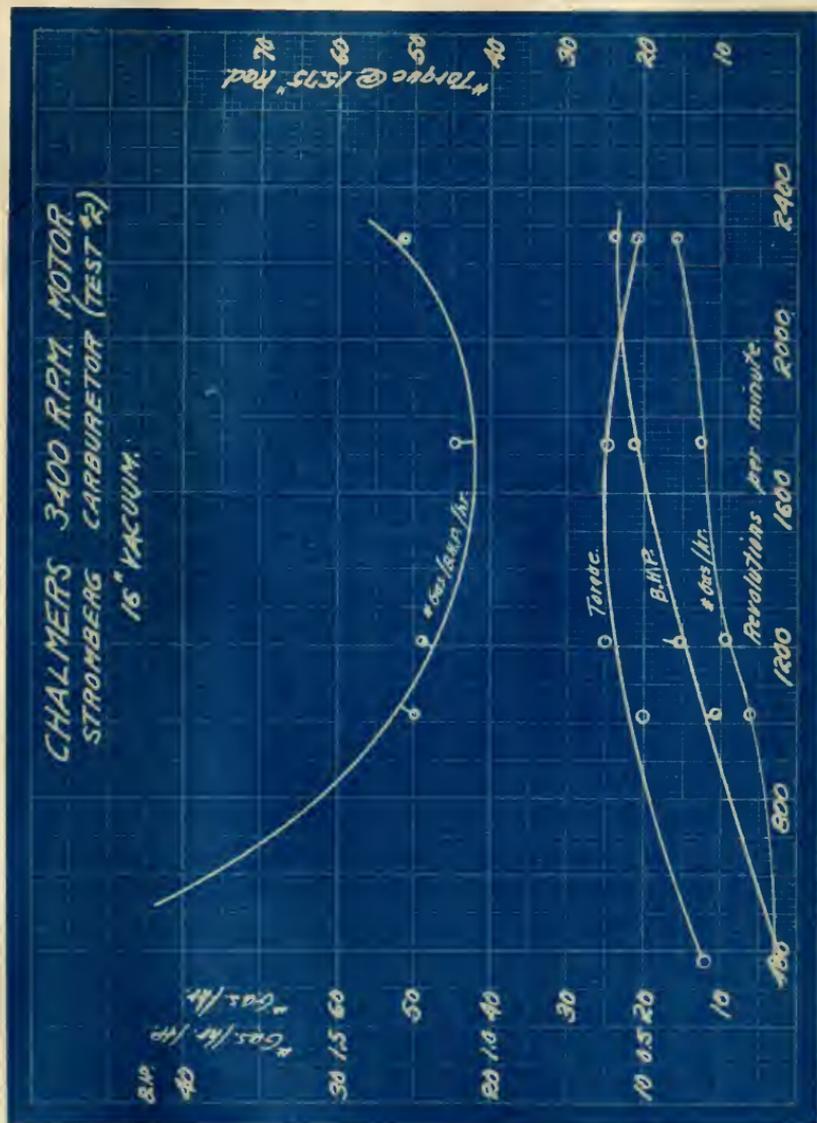


Fig. 52

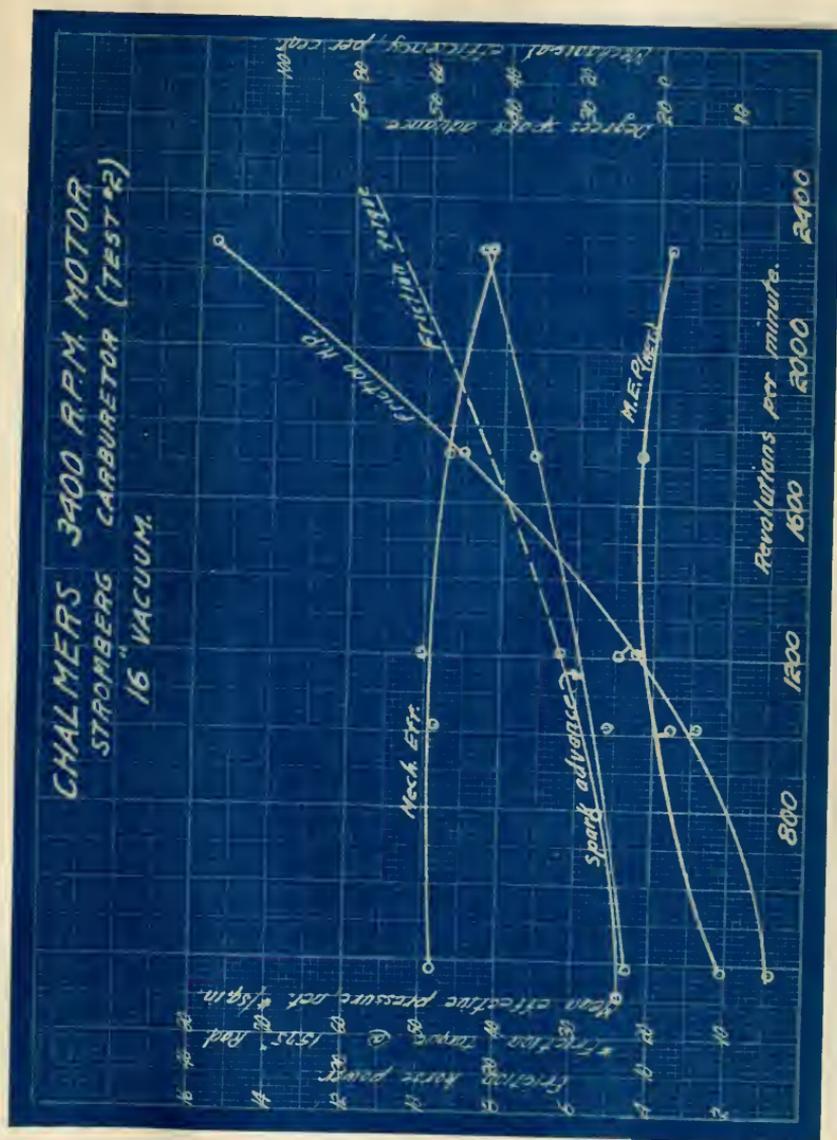


Fig. 53

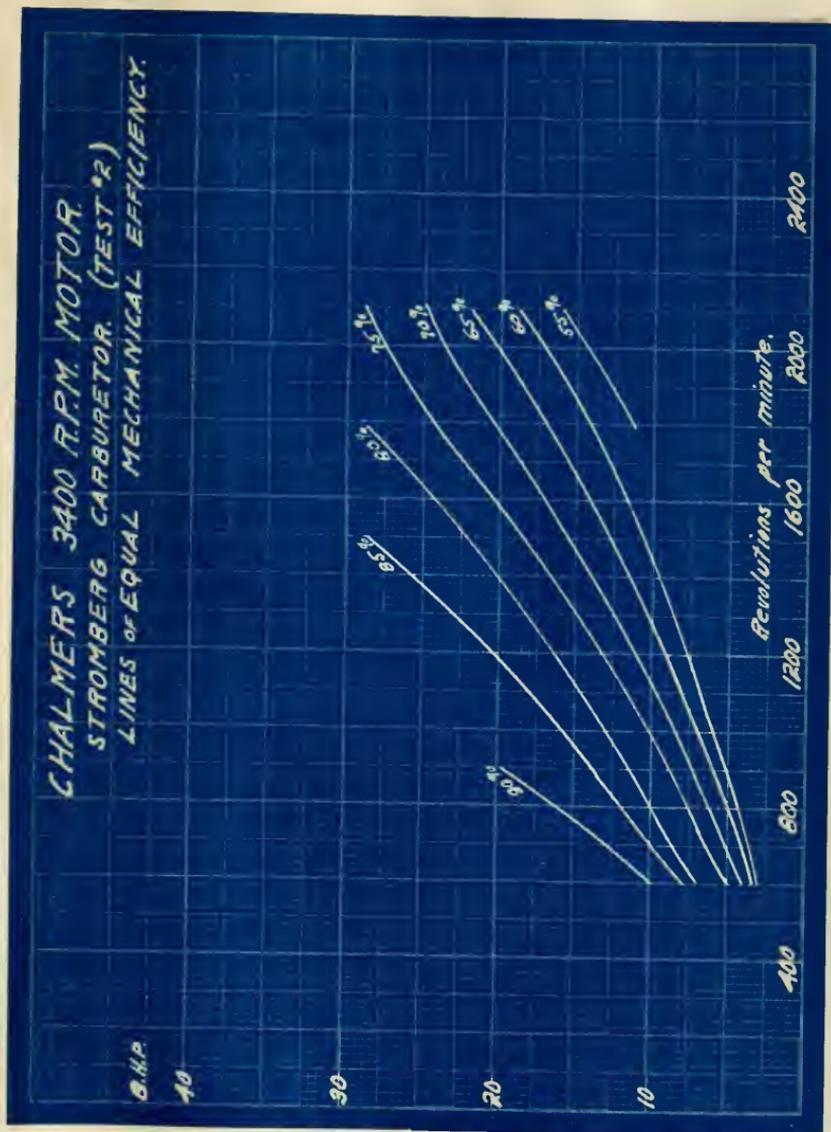


Fig. 55

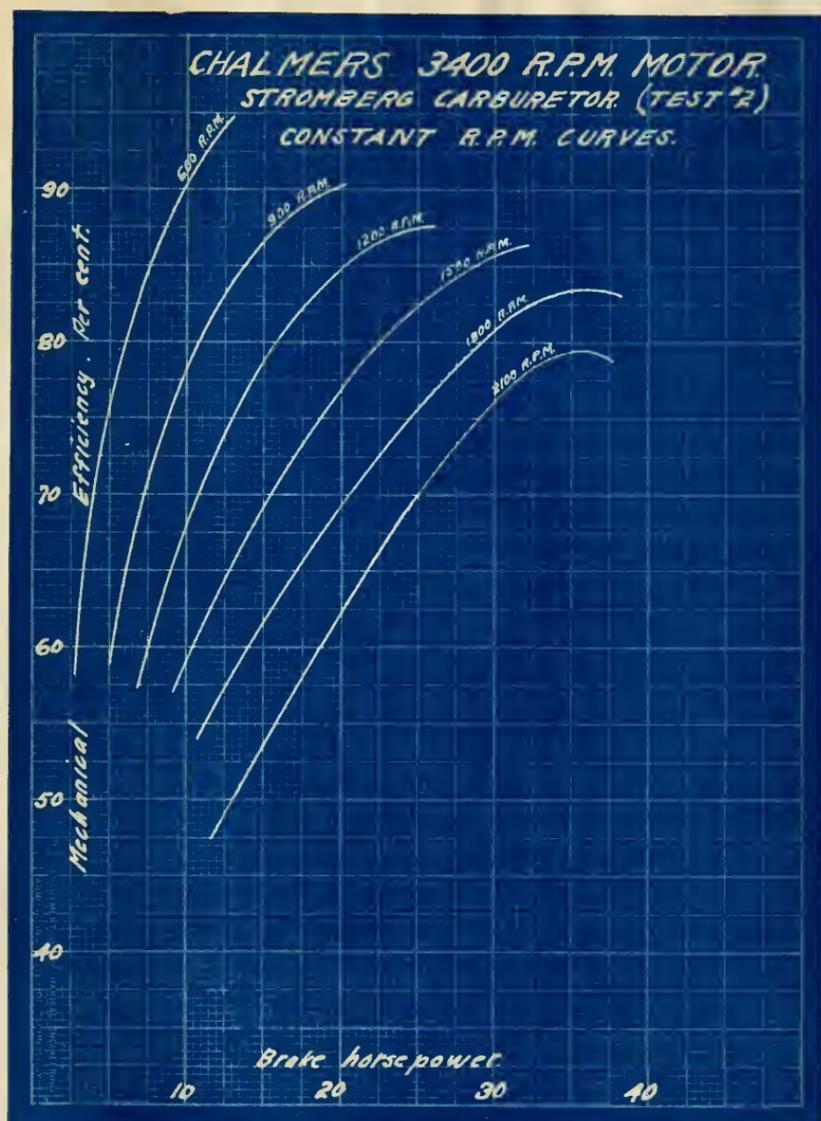


Fig. 56

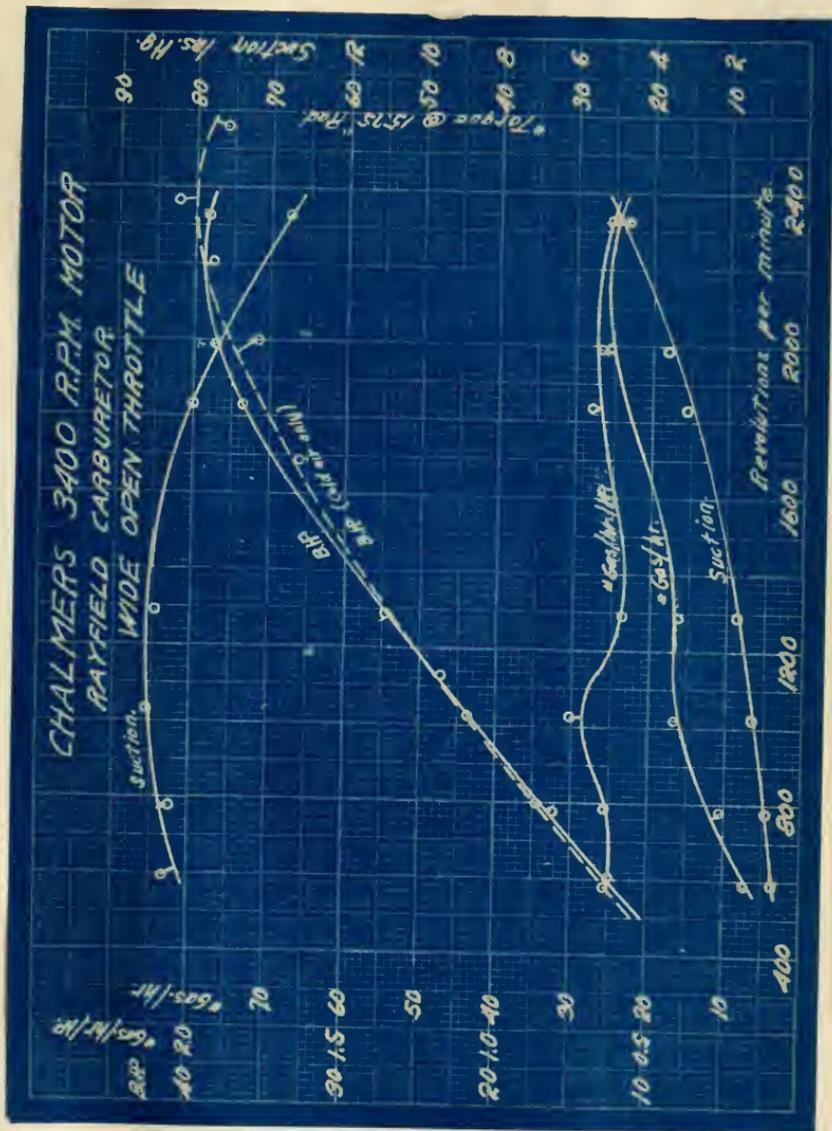


Fig. 57

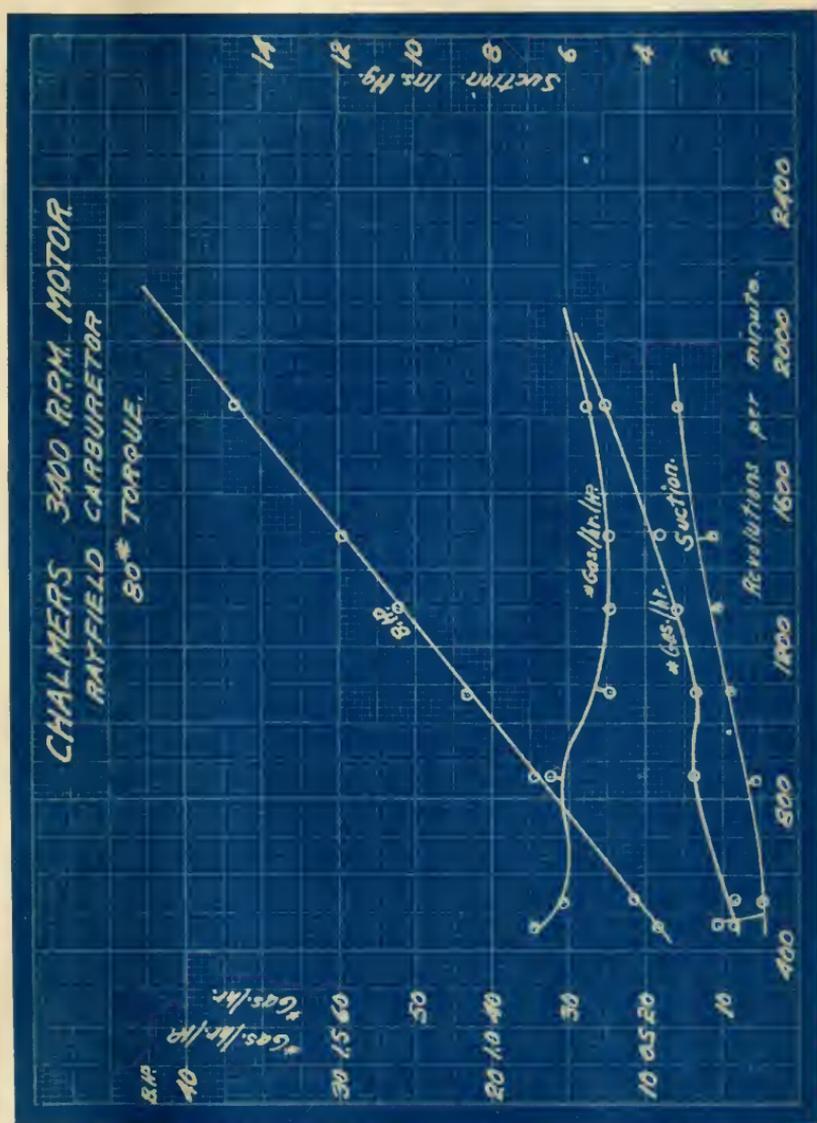


Fig. 58

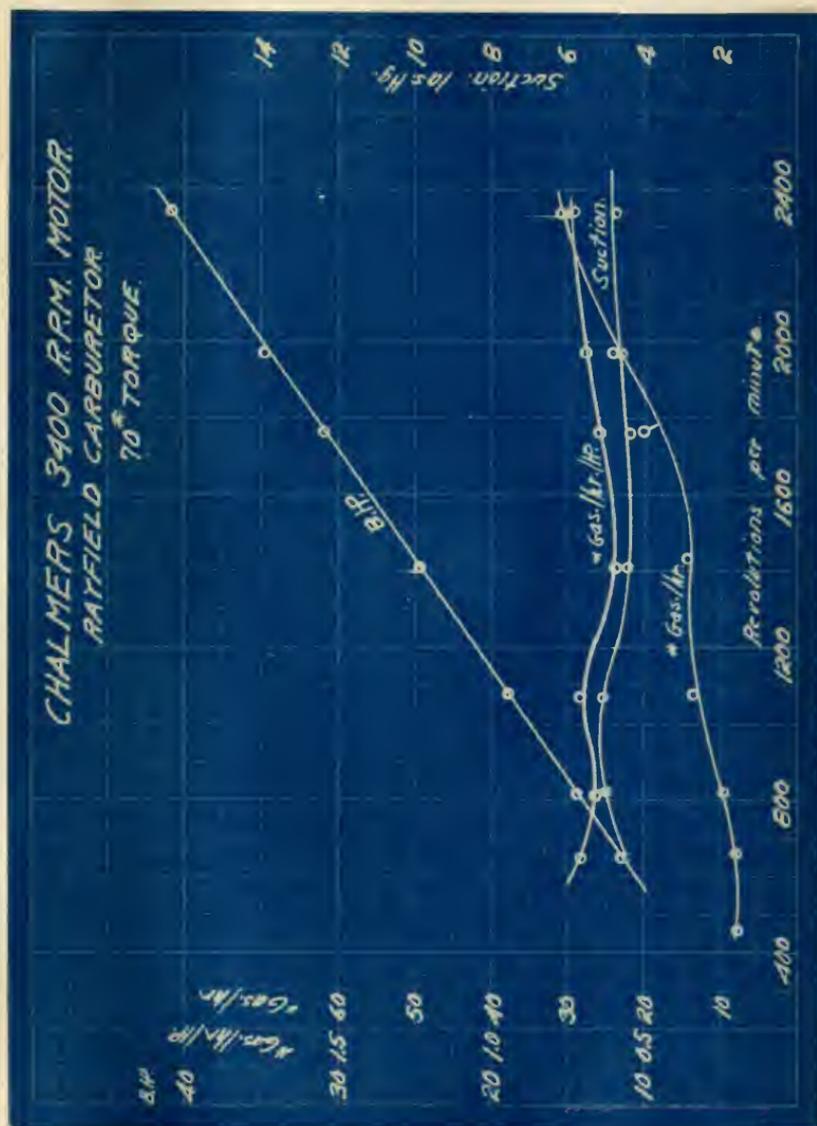


Fig. 59

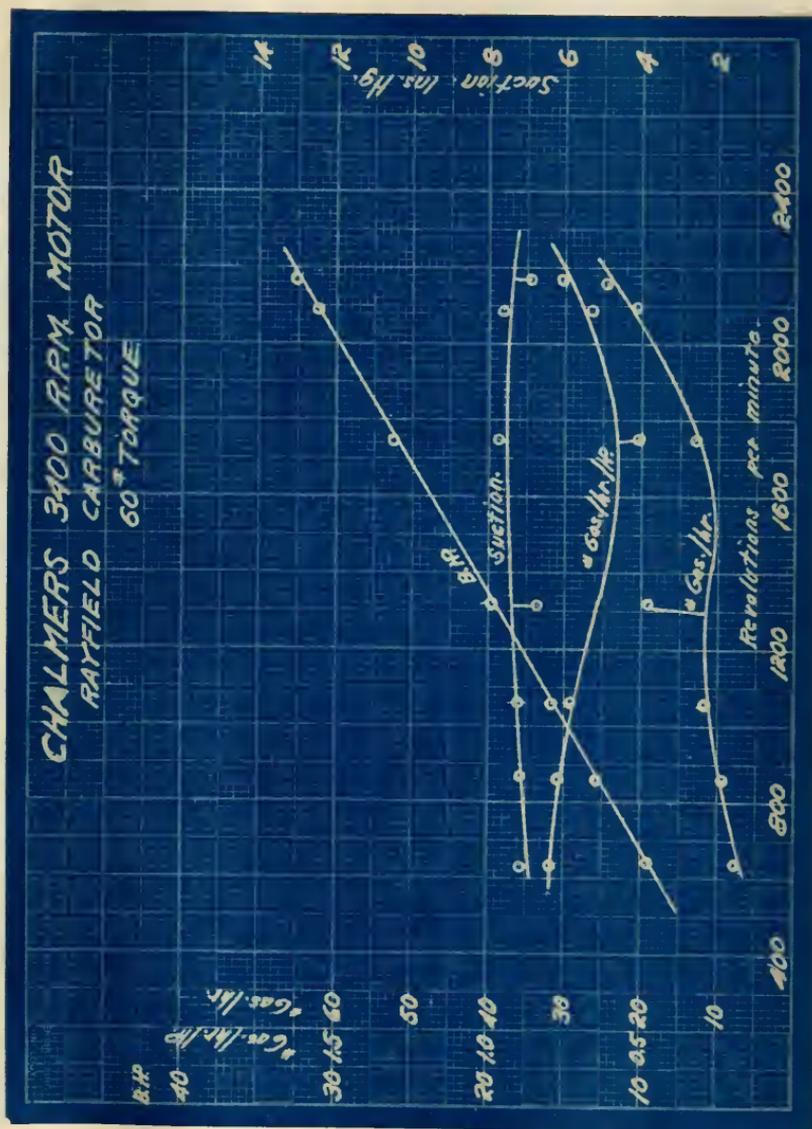


Fig. 60

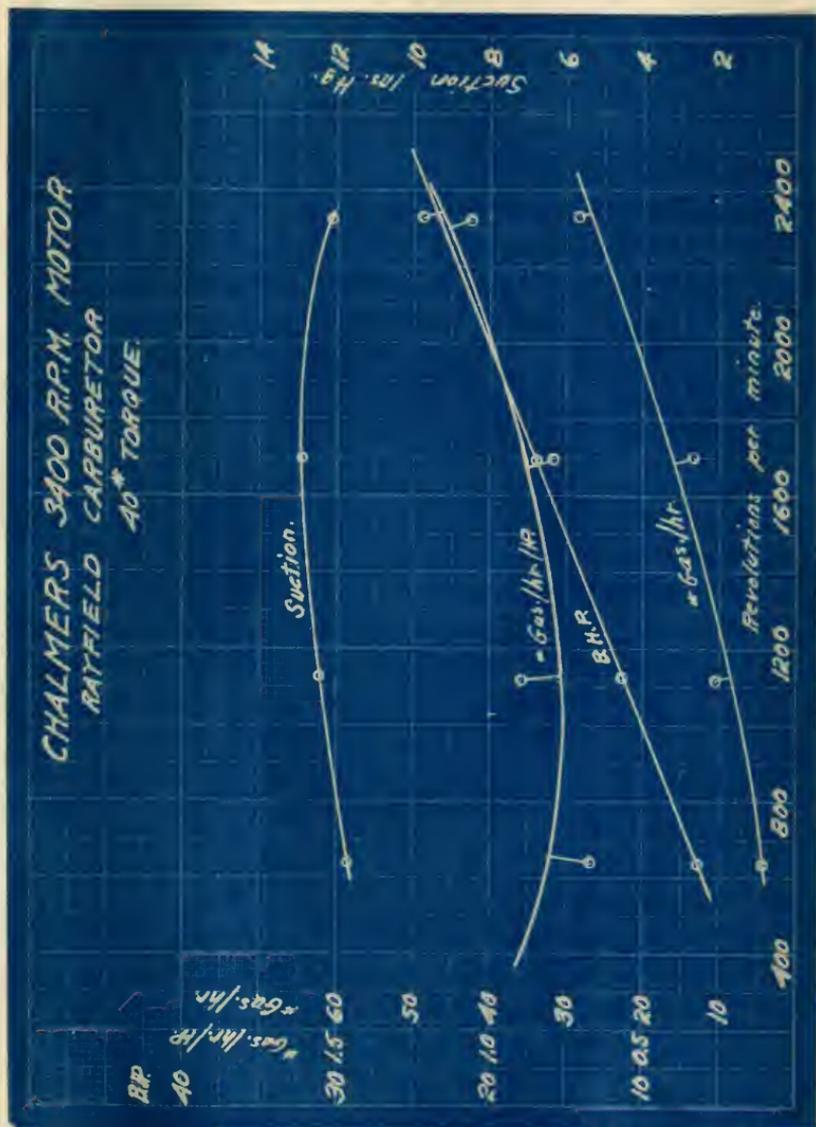


Fig. 62

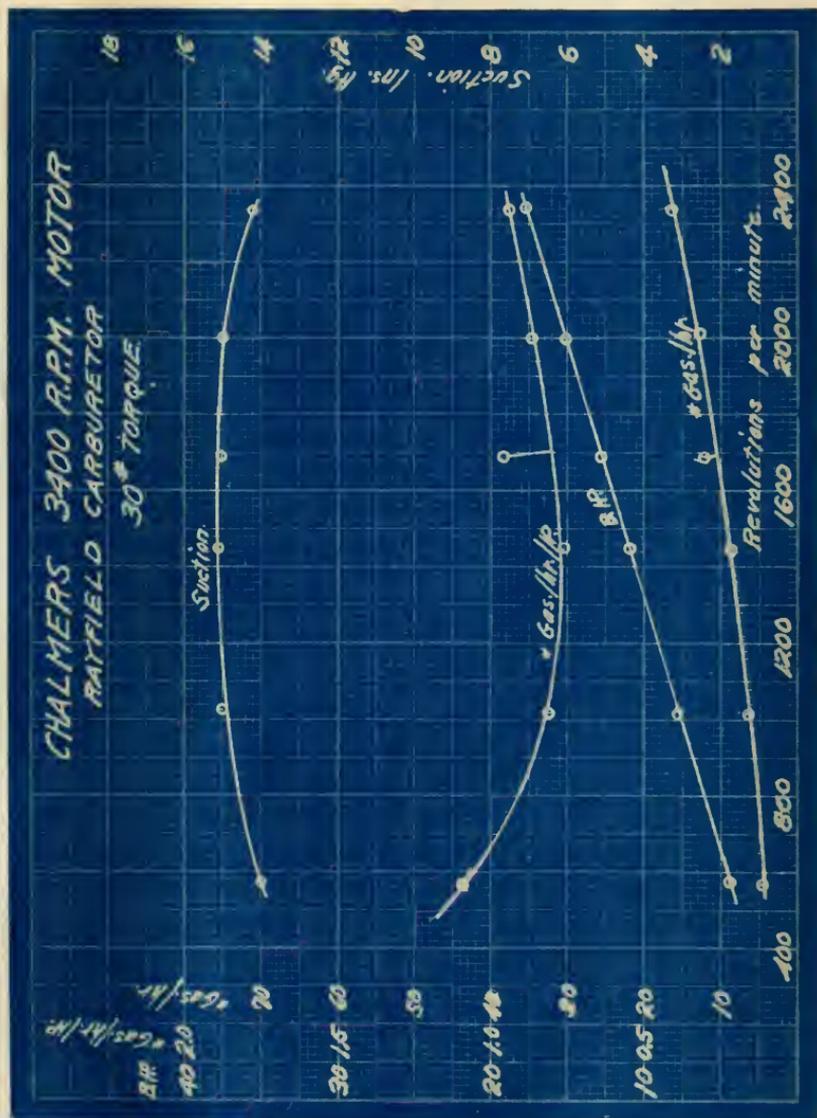


Fig. 63

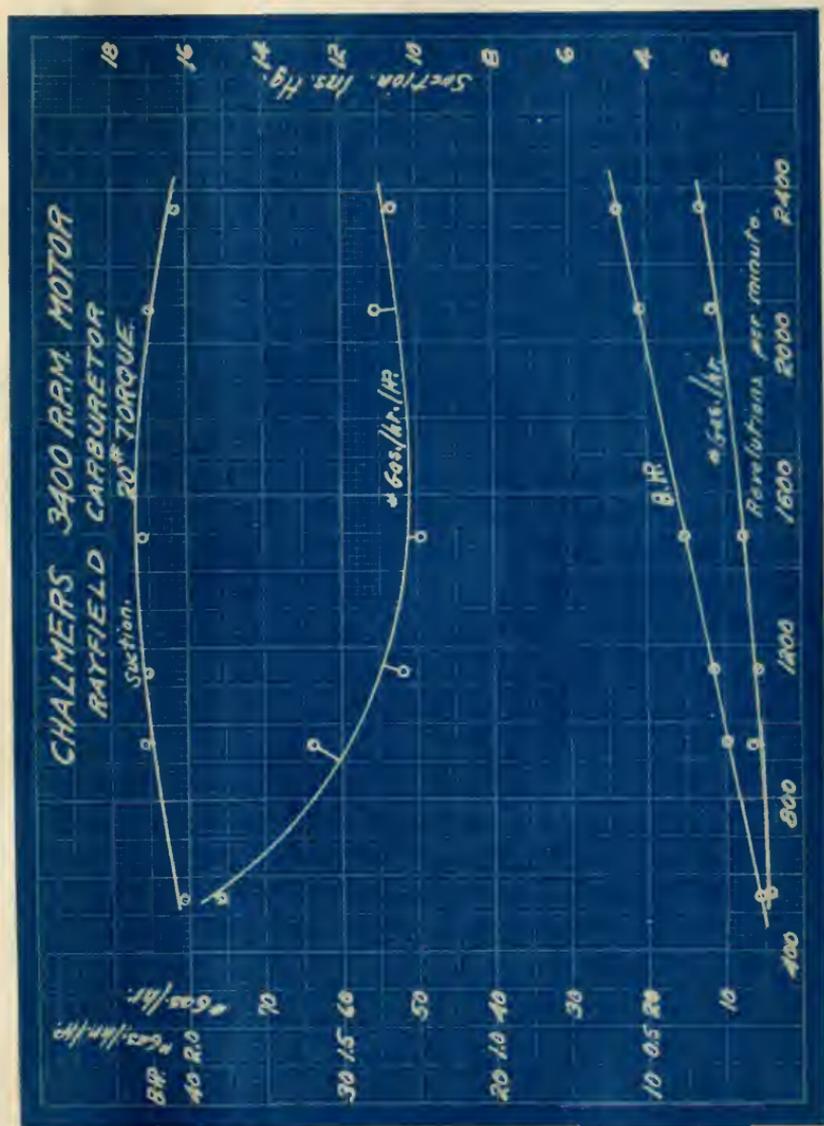


Fig. 64

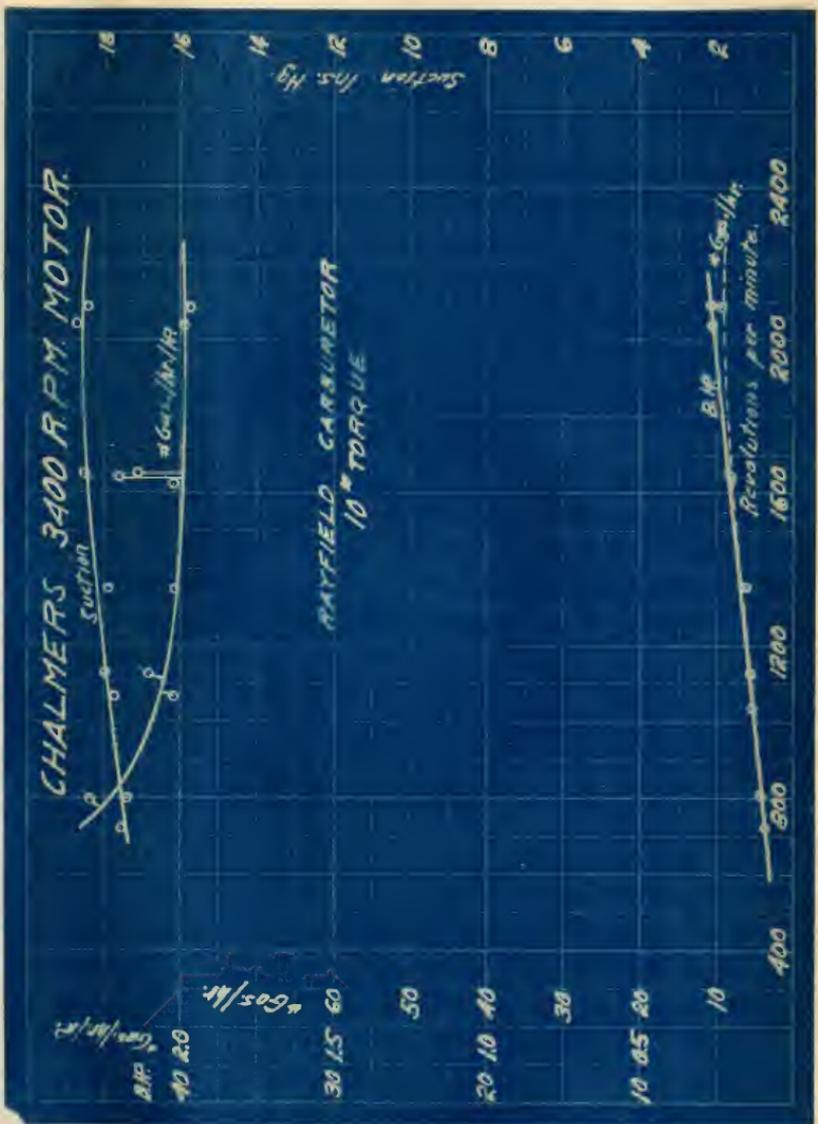


Fig. 65

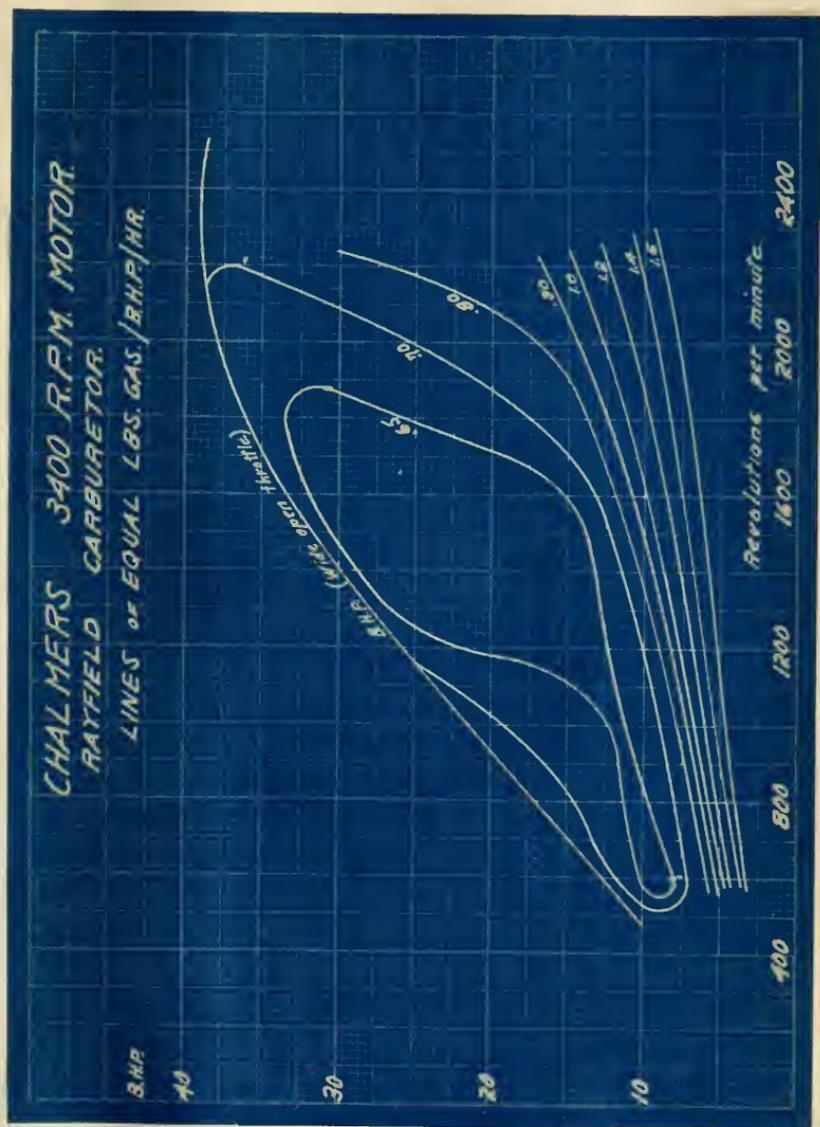


Fig. 66





CHALMERS 3400 RPM MOTOR
STROMBERG CARBURETOR (FEST "I")

Pounds gasoline per hour
(Calculated from original data)

Pounds gasoline per hour

Wagon
18" vacuum
16" vacuum
14" vacuum
12" vacuum
10" vacuum
8" vacuum
6" vacuum
4" vacuum
2" vacuum
Made seen in practice

CHALMERS 3400 RPM MOTOR
STROMBERG CARBUJETOR (TEST #2)
Pounds gasoline per hour
(Further than original data)

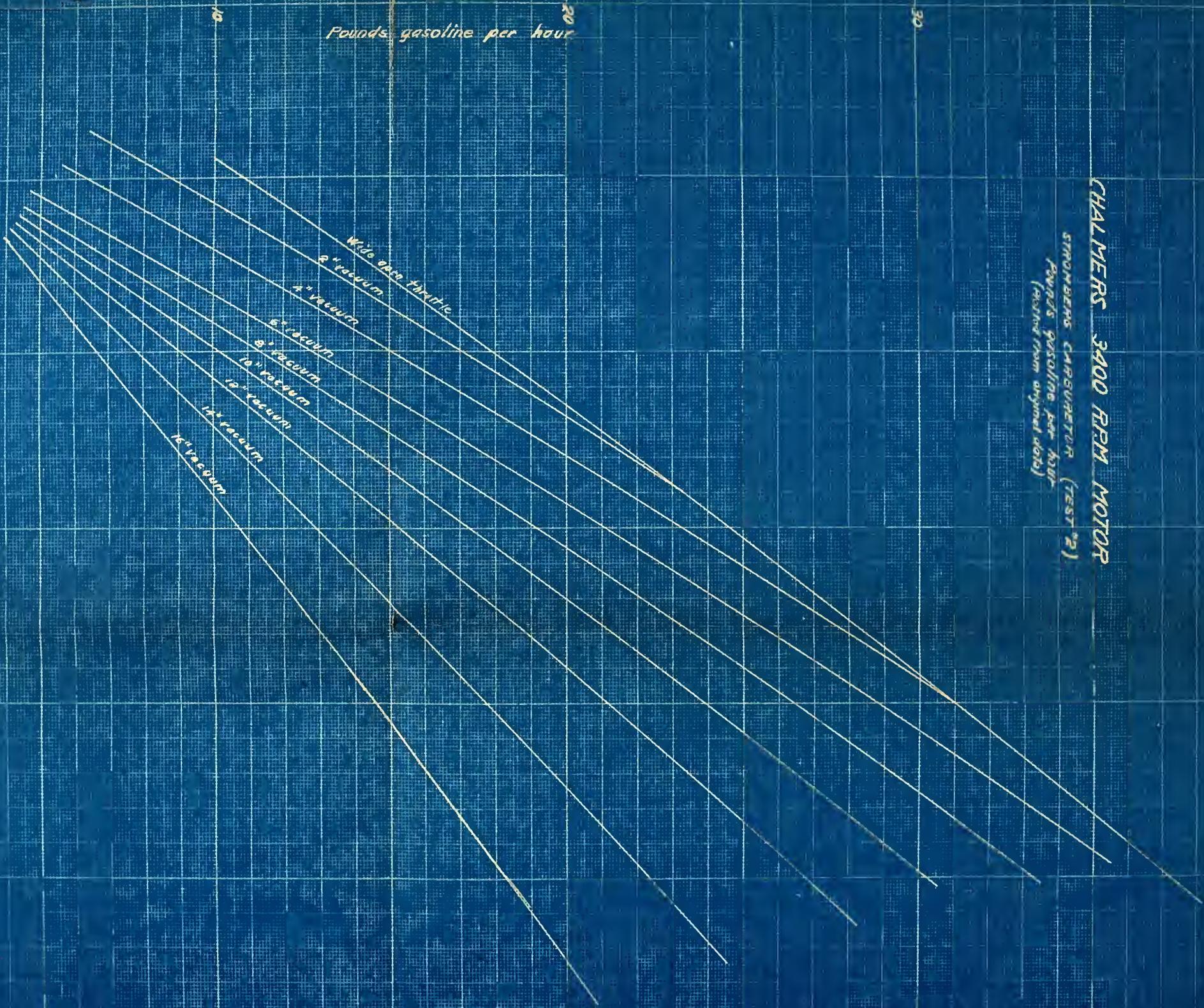


Fig. 69
Revolutions per minute

0

1000

2000

3000

Pounds gasoline per hour

0

20

30

40

CHALMERS 3400 R.P.M. MOTOR
STROMBERG CARBURETOR (TEST "2")
CONSTANT R.P.M. CURVES.

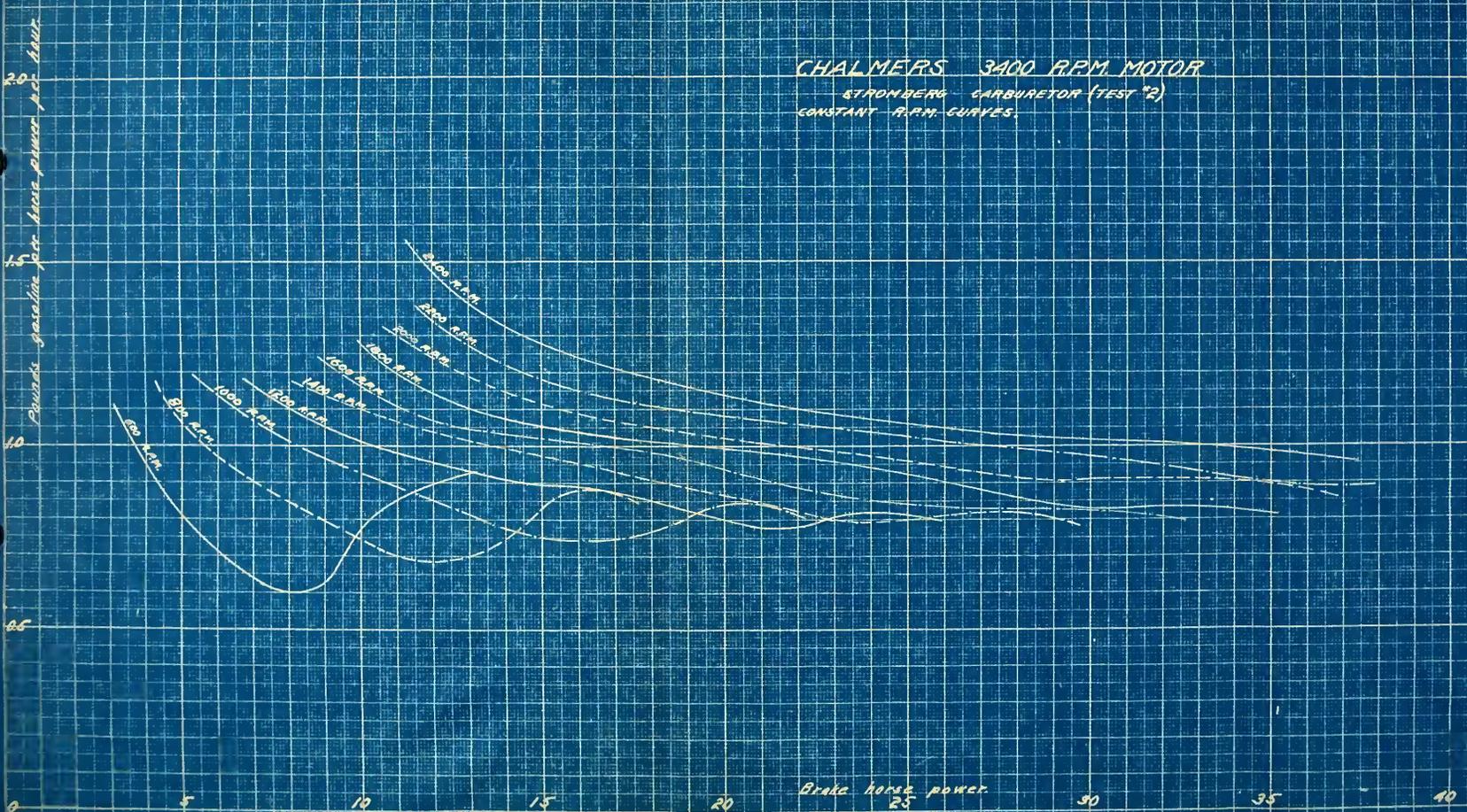


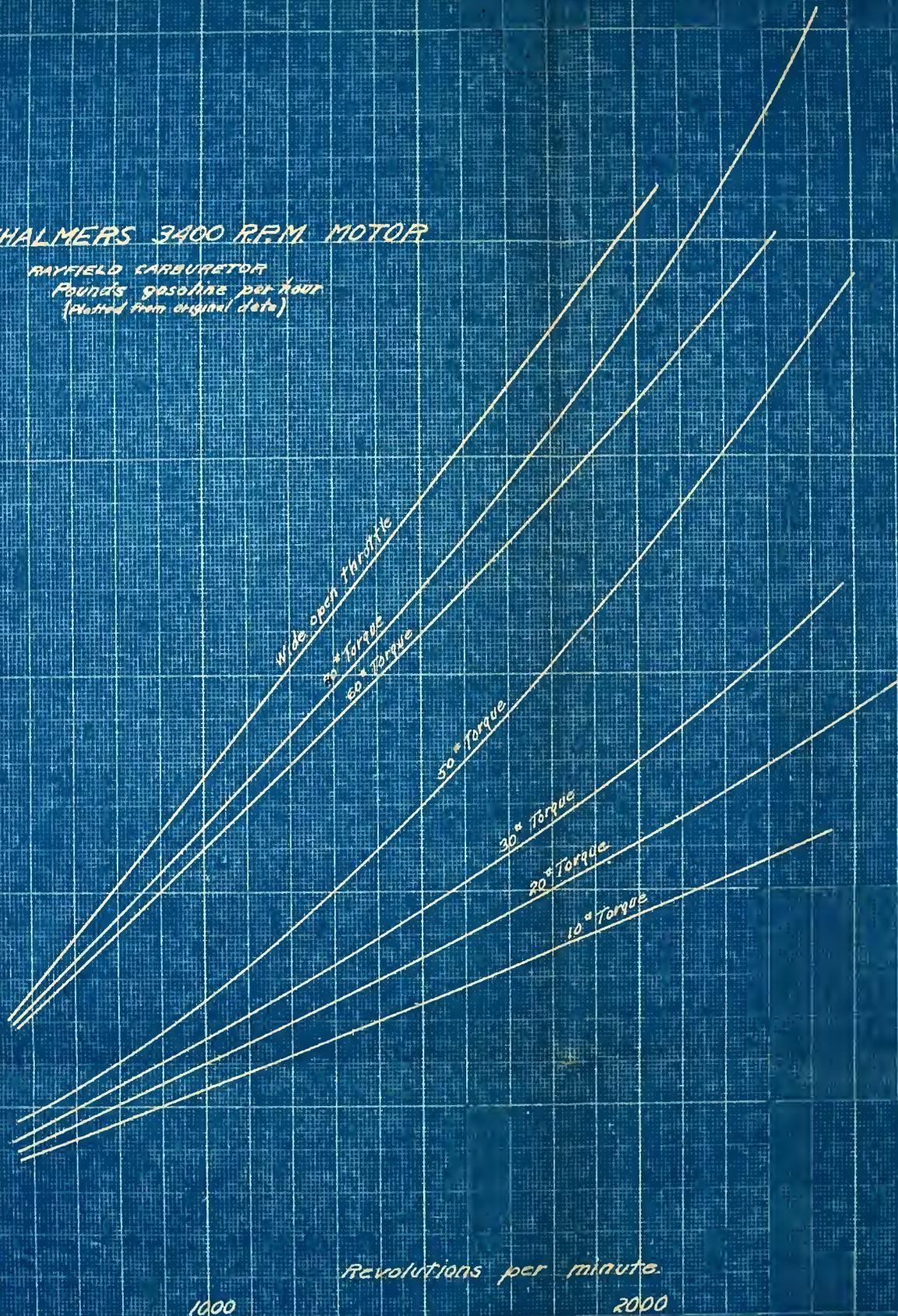
Fig. 70.

CHALMERS 3400 R.P.M. MOTOR

RAYFIELD CARBURETOR

Pounds gasoline per hour
(Plotted from original data)

Pounds gasoline per hour.



Revolutions per minute.

1000

2000

3000

Fig 71

CHALMERS 3400 R.P.M. MOTOR
 RAYFIELD CARBURETOR
 CONSTANT R.P.M. CURVES

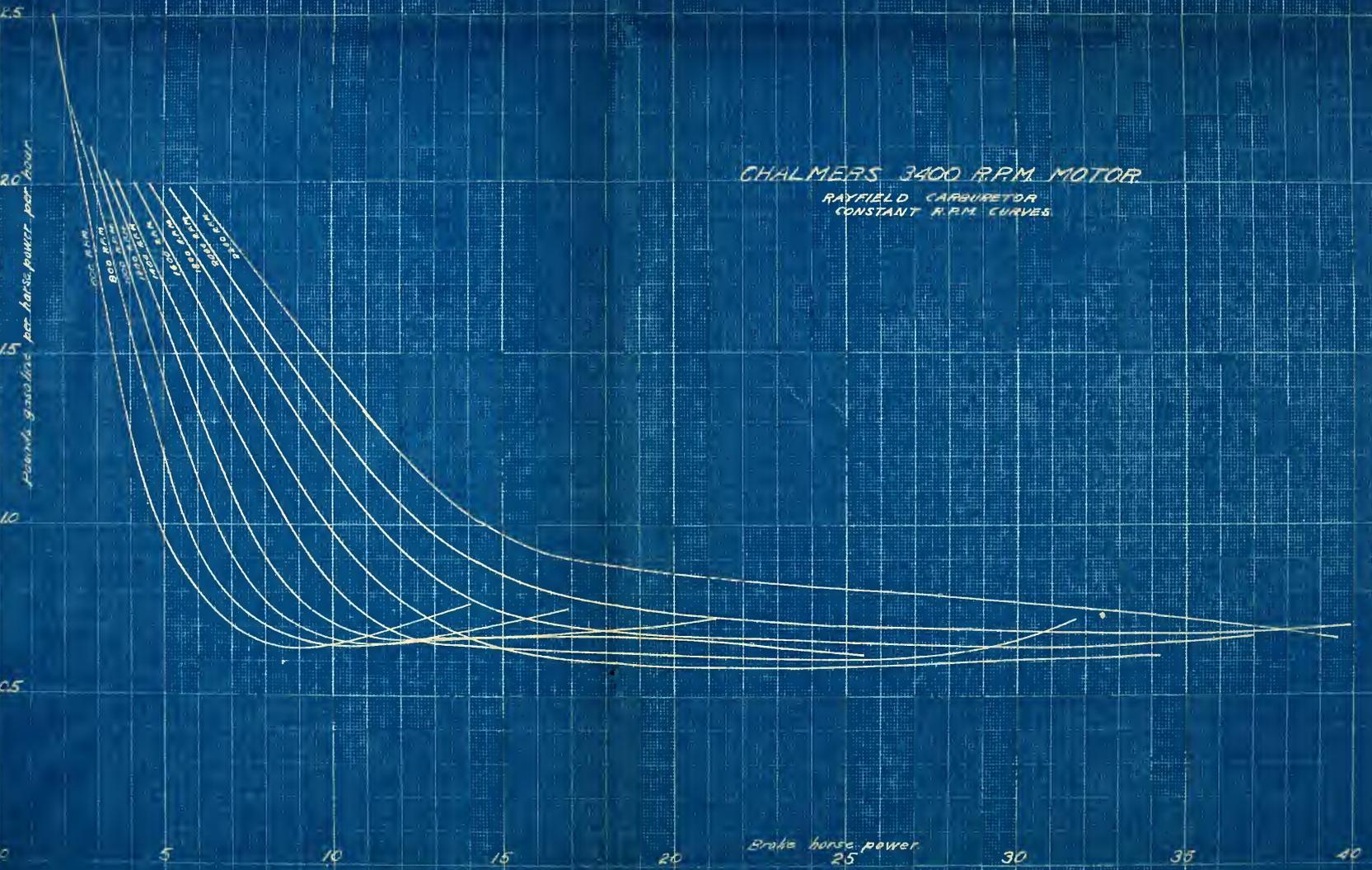


Fig. 72

